



Suspensions and Cryogenics for the HF and LF Interferometers

Fulvio Ricci

presentation dedicated to the memory of our friend *Stefano Braccini*

and

based on the contributions of several physicists of the ET team

F Acernese, F Barone, S Braccini, A Chincarini, R De Rosa, P Falferi, J Franc, F Frasconi, G Gemme, A Gennai, M Lorenzini, E Majorana, R Nawrodt, G M Perciballi, R Poggiani, P Puppo, D Rabeling, S Reid, S Rowan, M Tonelli, J F J van den Brand.....and many others.....

Talk outline

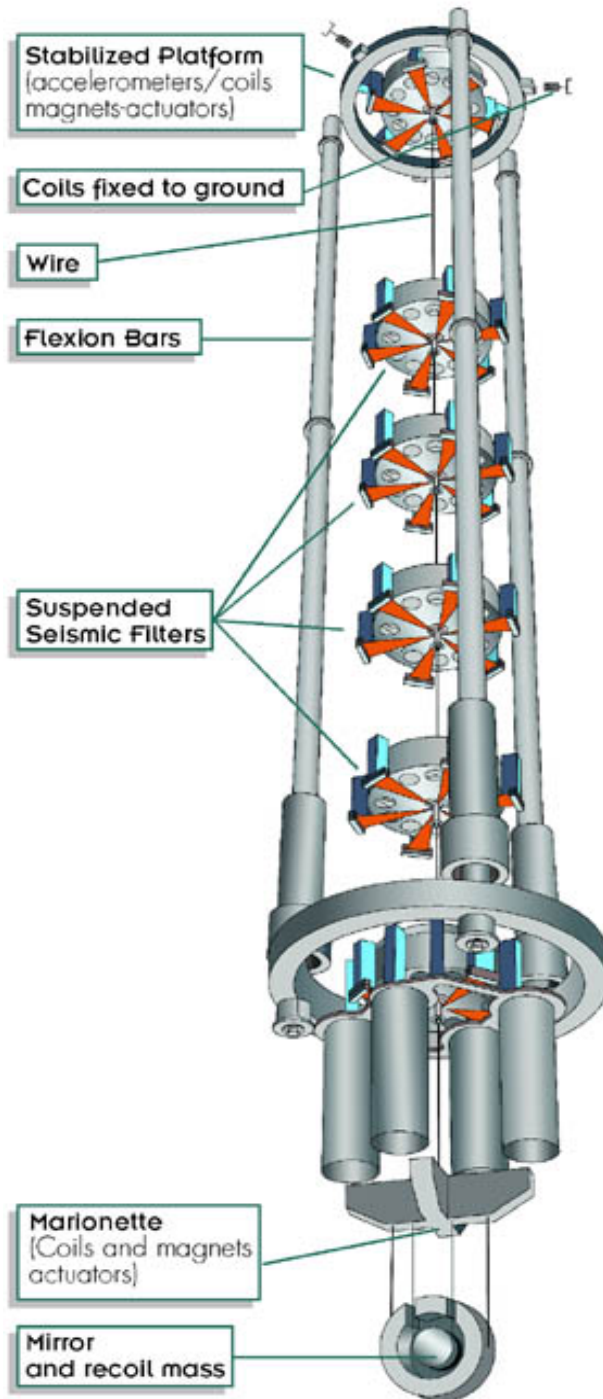
- Upper part suspensions for the seismic noise attenuation for both
 - ✓ ET-LF Interferometer and ET-HF Interferometer
- Last stage Suspension
 - ✓ for ET-HF Interferometer
 - ✓ for ET-LF Interferometer
- Local control issues
 - ✓ Local Sensors and Actuators
- Cooling constraints for the ET-LF Interferometer
 - ✓ cryo-fluid and cryo-generator
- Conclusion

Assumption

- Two independent interferometers
 - Main advantages:
 - Maximize the sensitivity in the entire frequency bandwidth
 - Commissioning and data taking activity in parallel

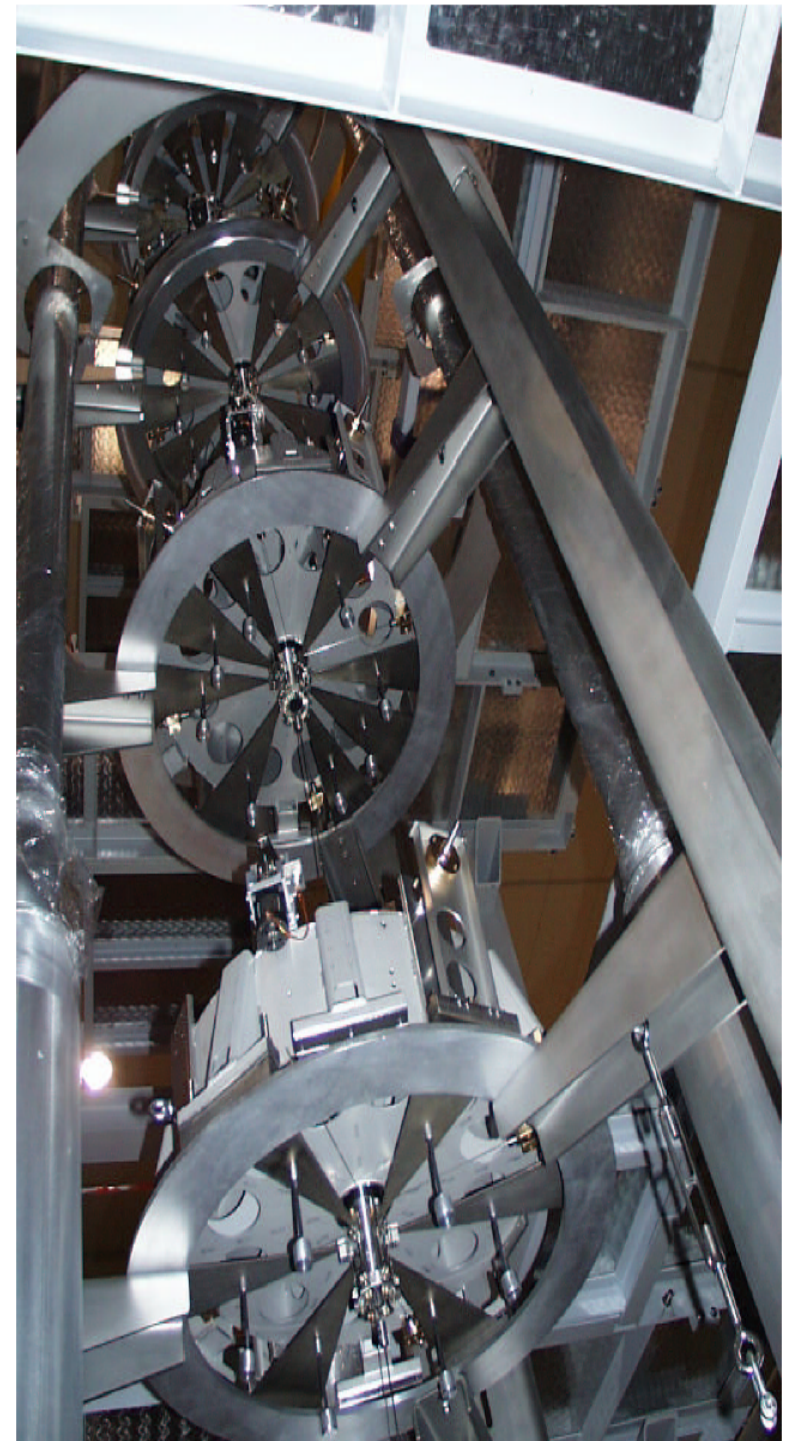
Consequence

- Double design
 - Two kinds of seismic attenuator chains
 - Two different payloads



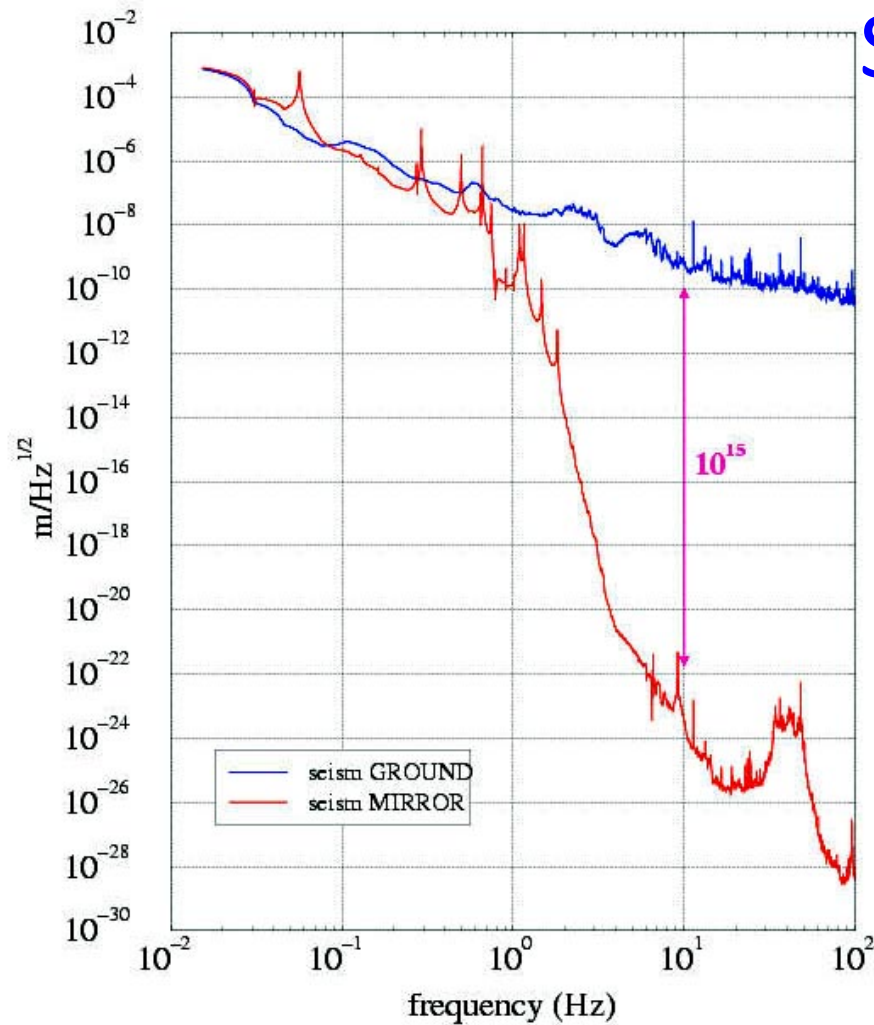
Upper part of the suspension for the seismic noise reduction

- First Stage: Inverted pendulum
- Five Pendulum Stages
- Suspension Last Stage: Marionette and Reaction mass for Control

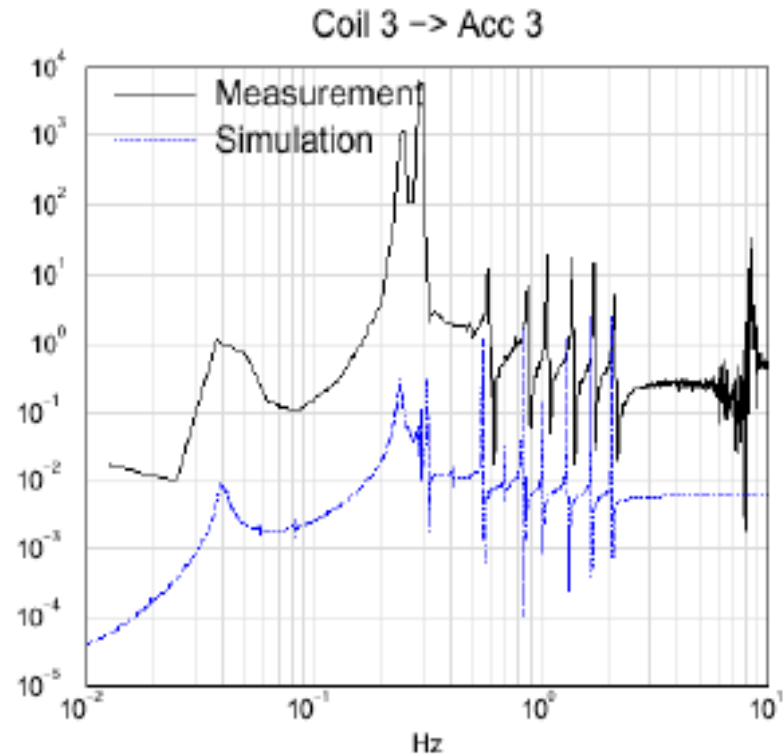


Starting point: VIRGO data

*Attenuation derived
by combining
the contribution of each filter*

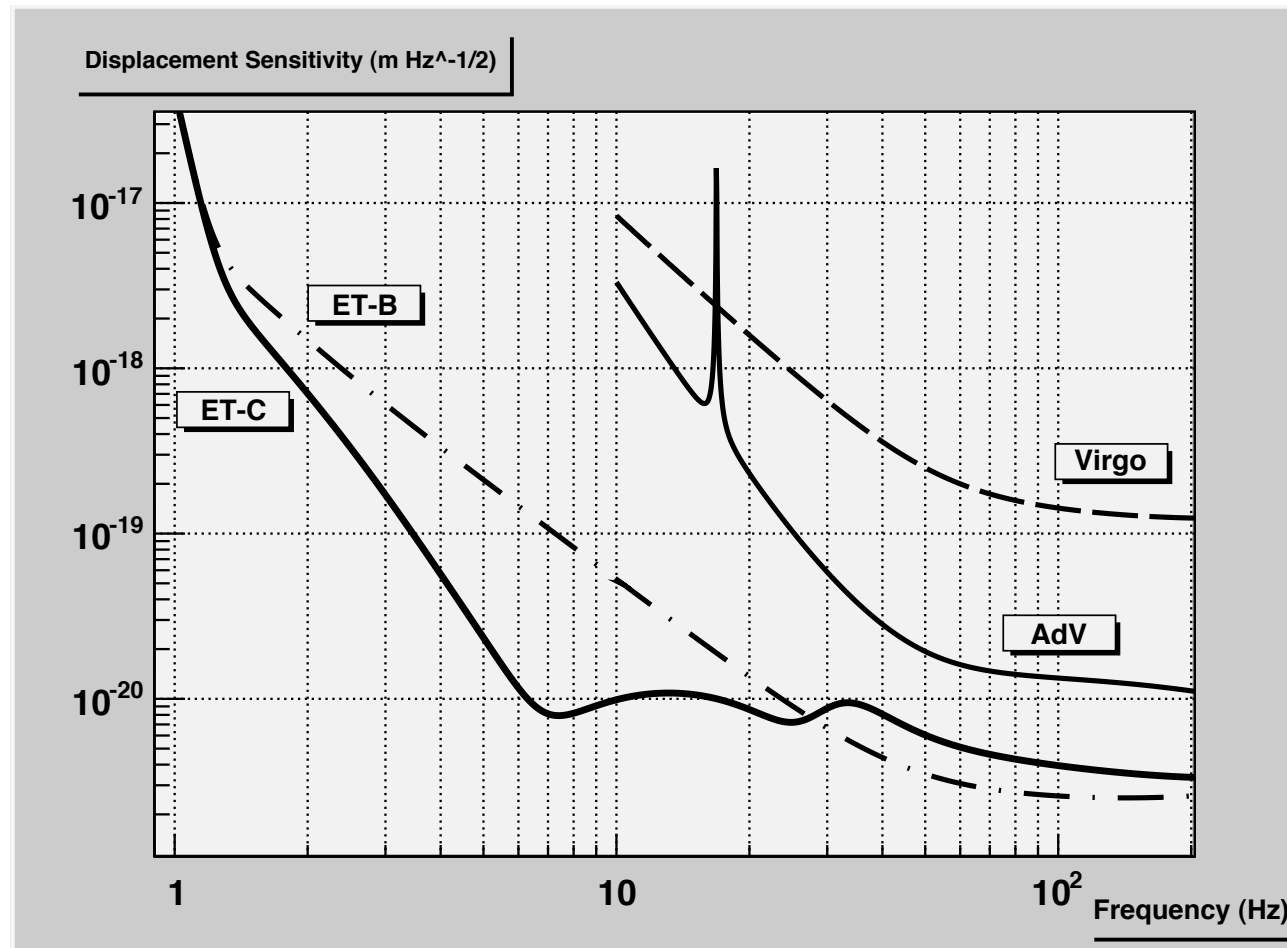


Suspended Mirror motion
compared to the
seismic motion



Single stage attenuation

Seismic vibration transfer function requirements



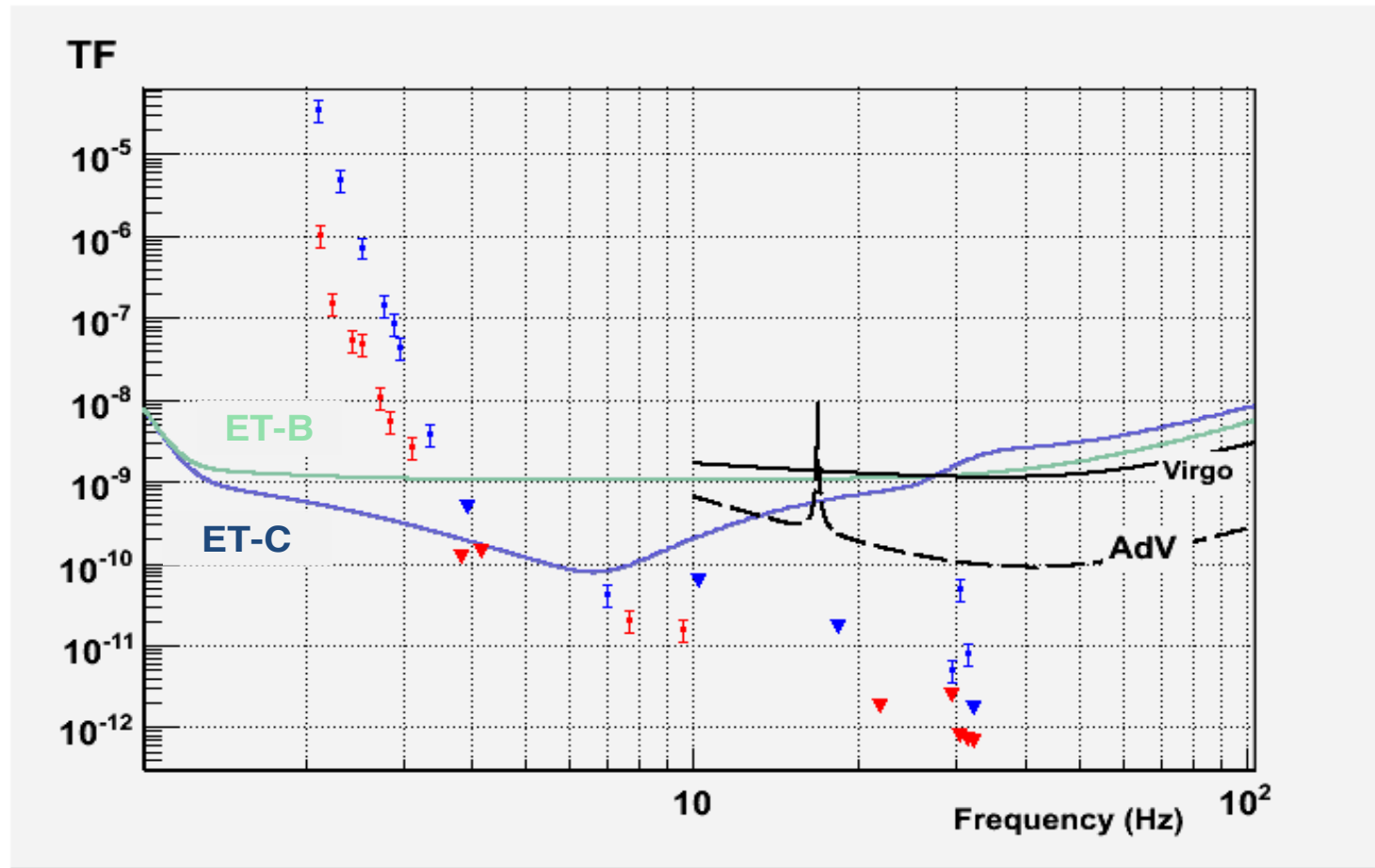
The curves represent the displacement sensitivities of the different instruments referred to different seismic noise level (VIRGO site seismic noise and Kamioka mine seismic noise.

The seismic noise will be at least a couple of orders of magnitude smaller in underground environment.

ET is less demanding in terms of seismic attenuation at high frequency.

Super-Attenuation Measurements on VIRGO

Acernese F. et al. , Astropart. Phys. 33, 182 (2010)

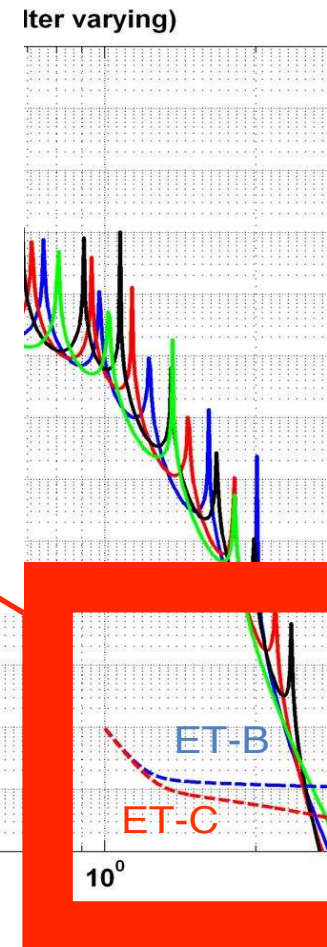
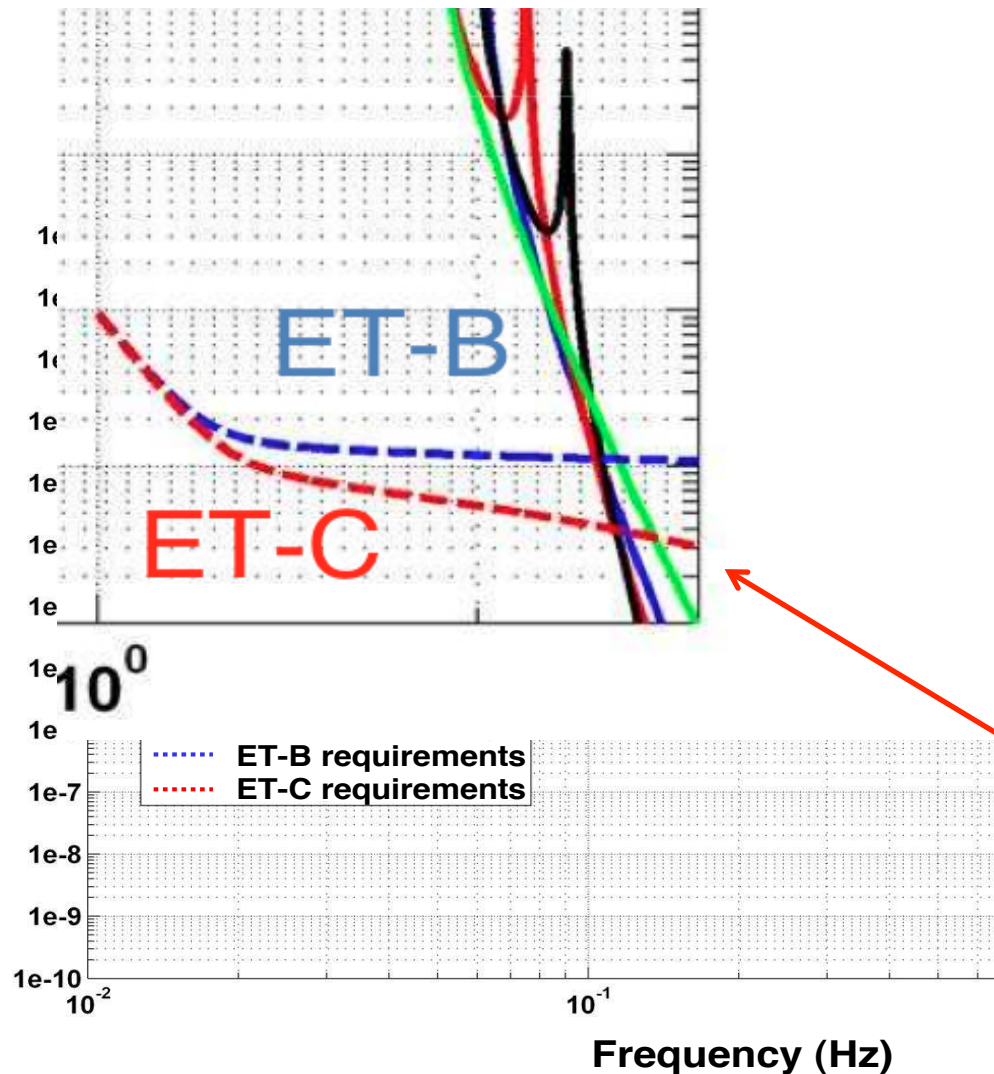


Red points: *measurements where a vertical excitation of the top stage is applied.*

Blue points: *measurements with the excitation in horizontal direction.*

Triangles → upper limits

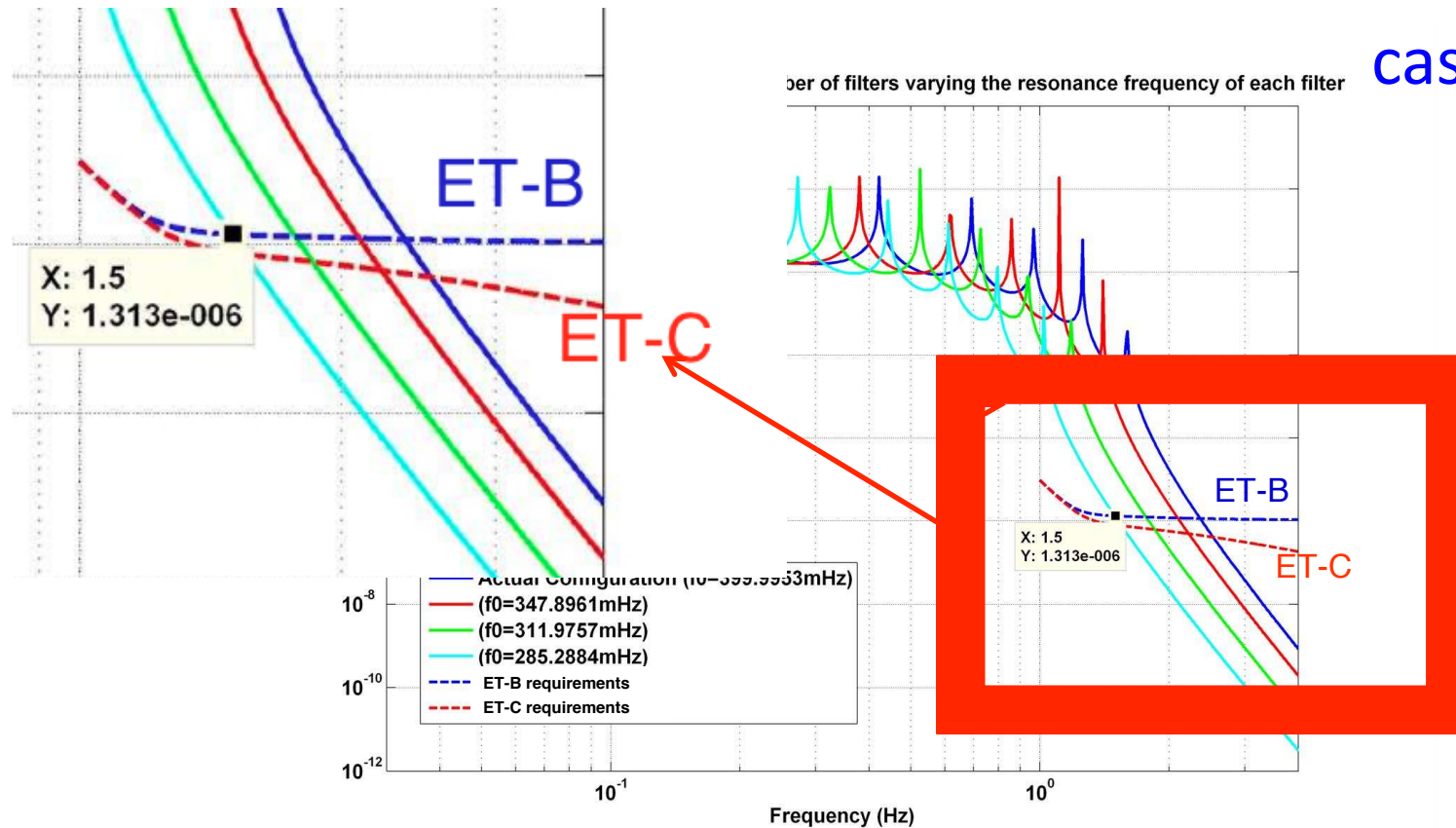
Bars → direct measurements of the signal at the level of the mirror



Simulation
results for
the two ET
cases I

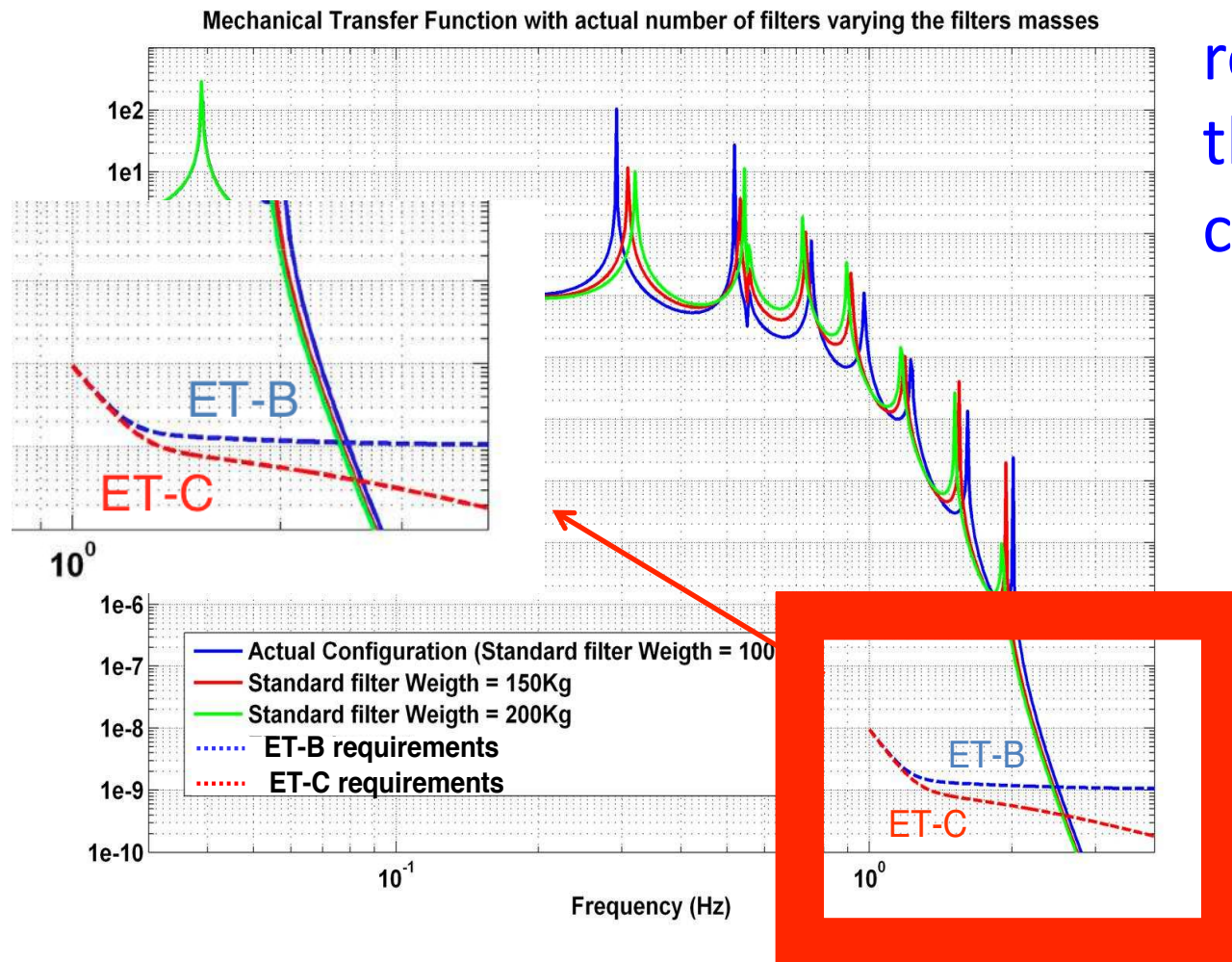
Simulation results of the SA horizontal transfer function with the present chain length (9 m). Changing the number of (“equal-spaced”) filters the resulting horizontal transfer function is compared with the ET requirements (for High Frequency—ET-B and ET-C). Adding or removing filters along the chain length do not have remarkable role in the positioning **of the cross-over frequency with the requirements**

Simulation results for the two ET cases II

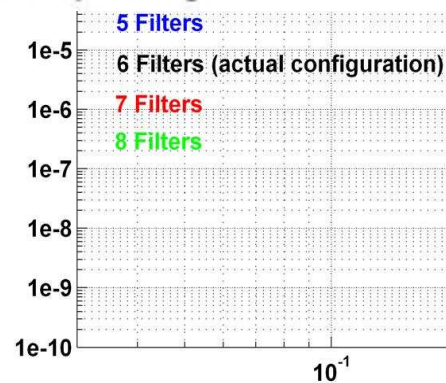
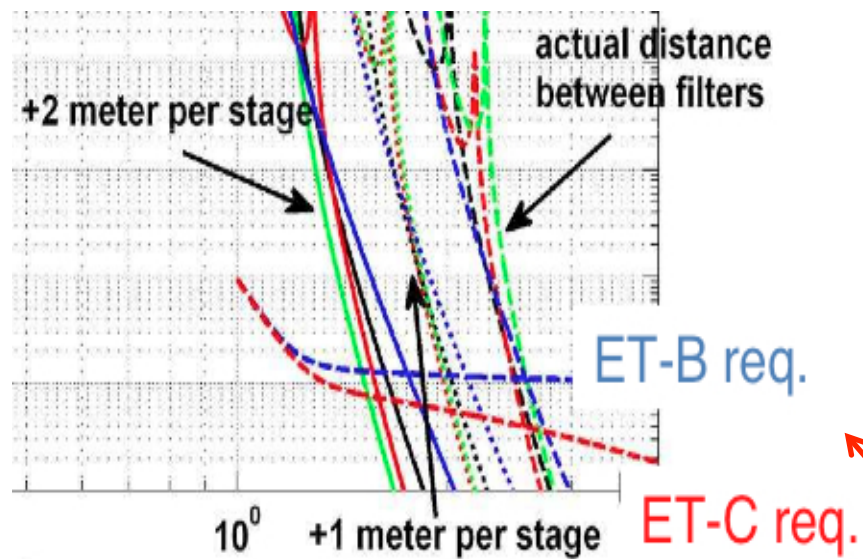


Vertical Transfer Function of the SA considering the six stages (as it is now, i.e. with the pre-isolator or "Filter Zero" plus other five mechanical filters). The different curves have been obtained changing the filter vertical resonant frequency. With filters working around 300 mHz it is possible to move the cross-over below 2 Hz.

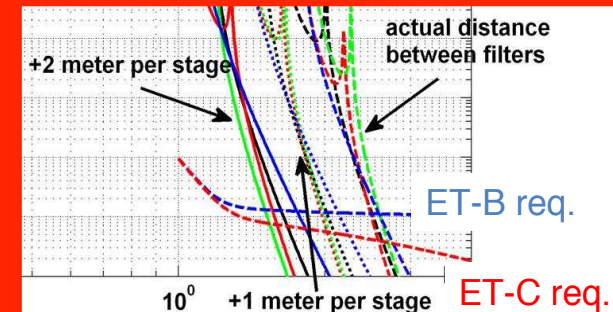
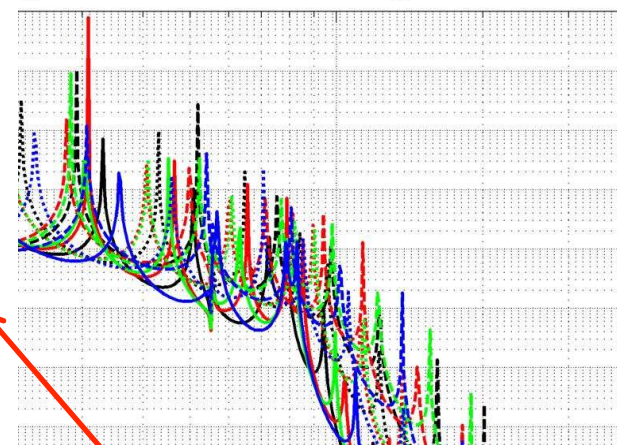
Simulation results for the two ET cases III



The horizontal transfer function of the present SA (6 filters weighting 100 kg each one for a total length of about 9m) is compared with the same transfer function changing the mass of each filter (150kg and 200 kg). Also in this case the cross-over frequency with the ET requirements is not remarkably affected by the change of the filter mass.

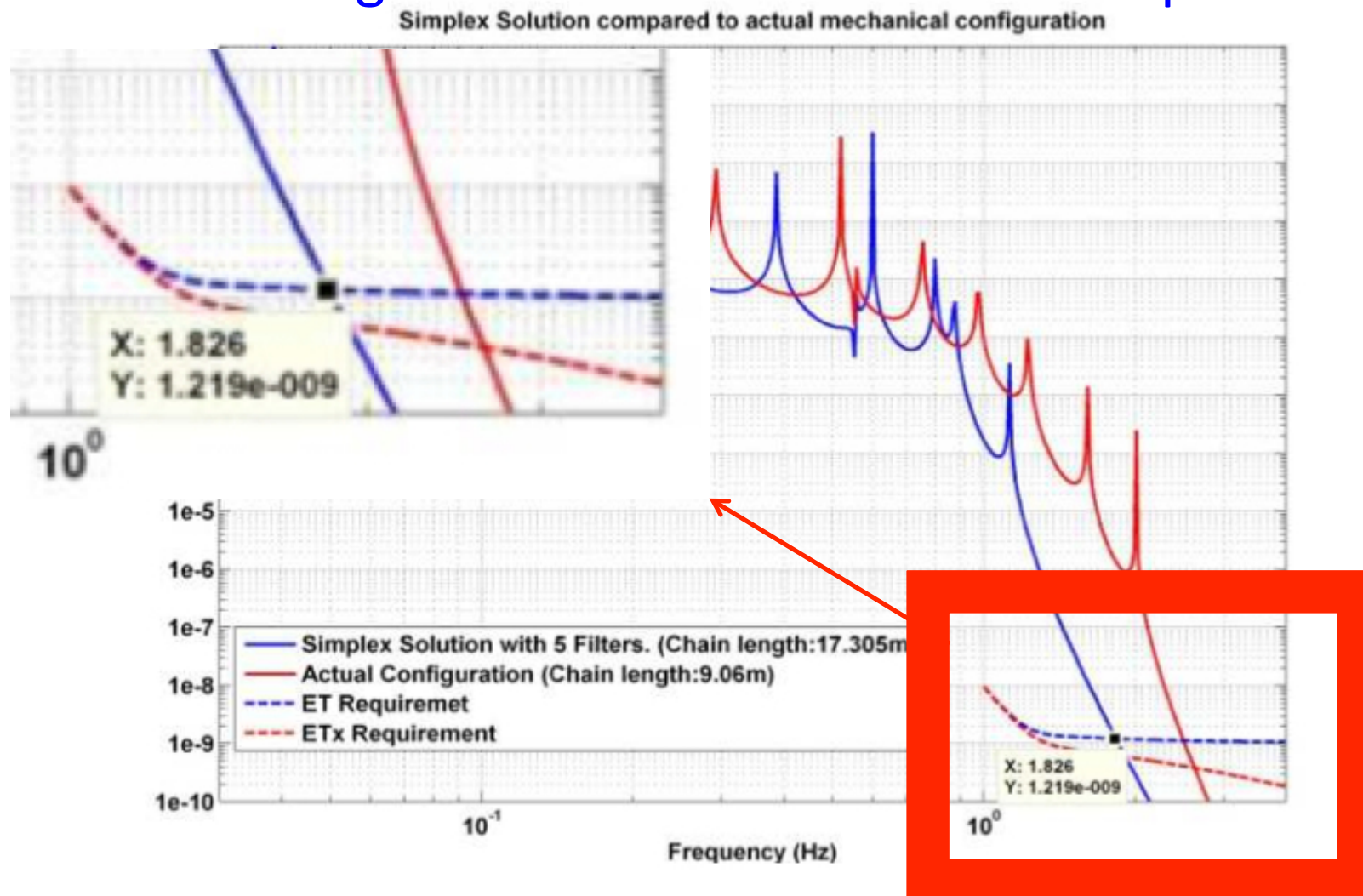


1g the number of filters and the length between filters

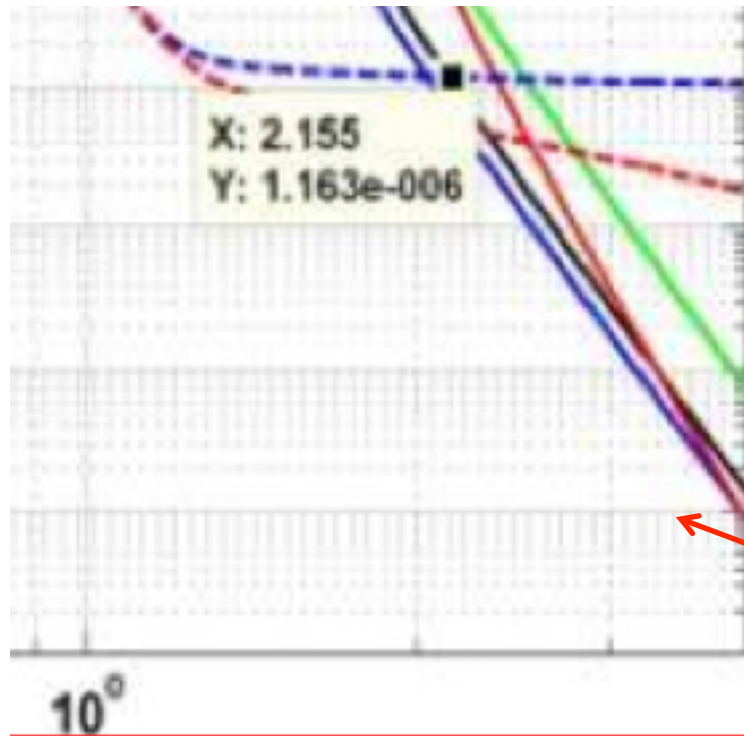


Simulation results for different configurations. The horizontal transfer function of the SA is plotted changing the number of filters and keeping fixed their relative distances ("equal-spaced" geometry) along the chain (changing, as a consequence, the full length of the SA).

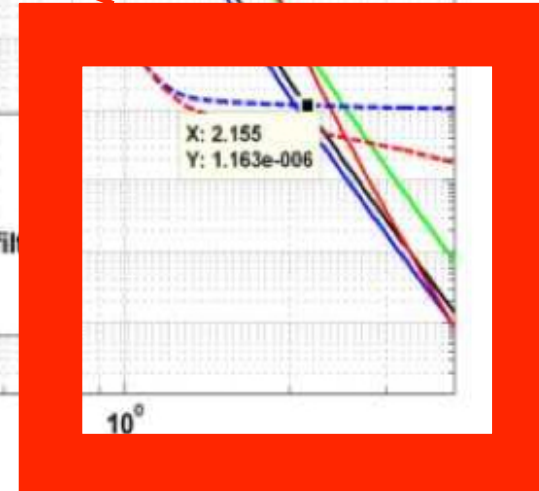
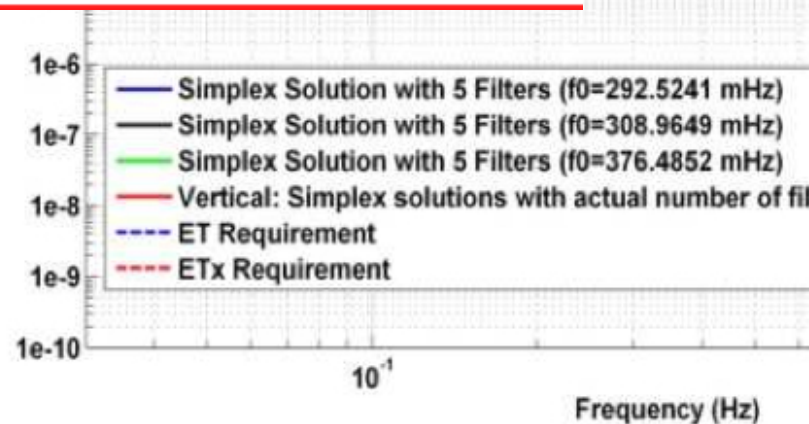
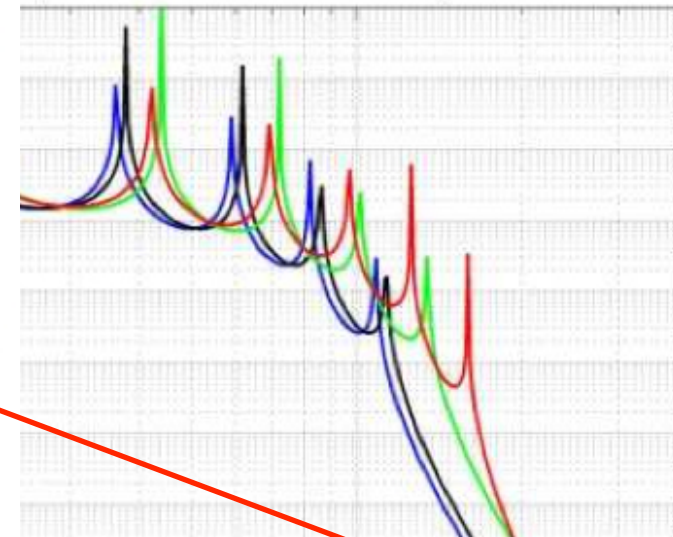
The proposed reference solution for the SA configuration of the Einstein Telescope.



The proposed reference solution for the SA configuration of the Einstein Telescope.



mpared to actual mechanical configuration



Conclusions on the Upper suspension I

- *ET-HF interferometer*

a Virgo-like Superattenuator can be used as seismic isolation system:

- ✓ the attenuation performance is compliant with the ET requirements;
- ✓ the construction technology is well tested ;
- ✓ the control strategy has been refined during over 10 years activity

Conclusions on the Upper suspension II

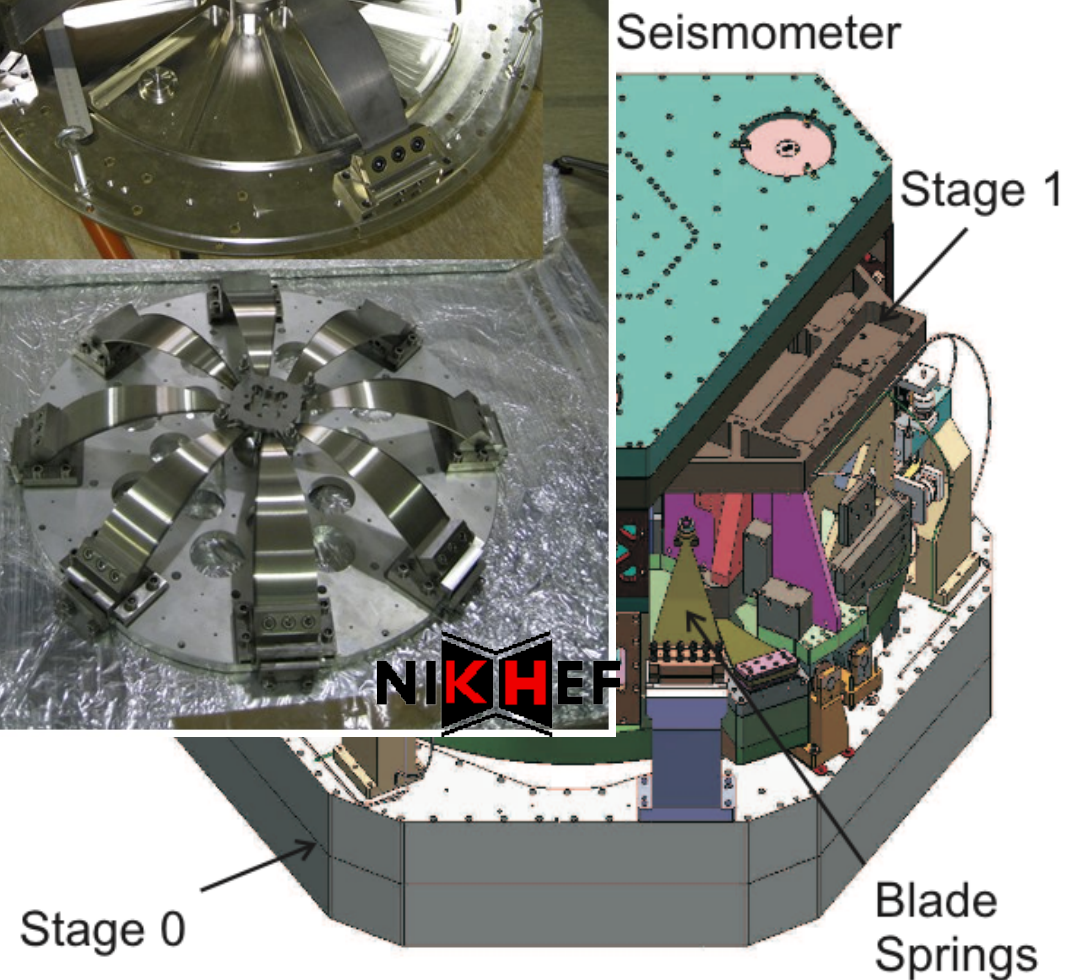
- *ET-LF Interferometer*

Super-attenuator **17 m tall** with 6 stages (filters) in "equal-spaced" configuration

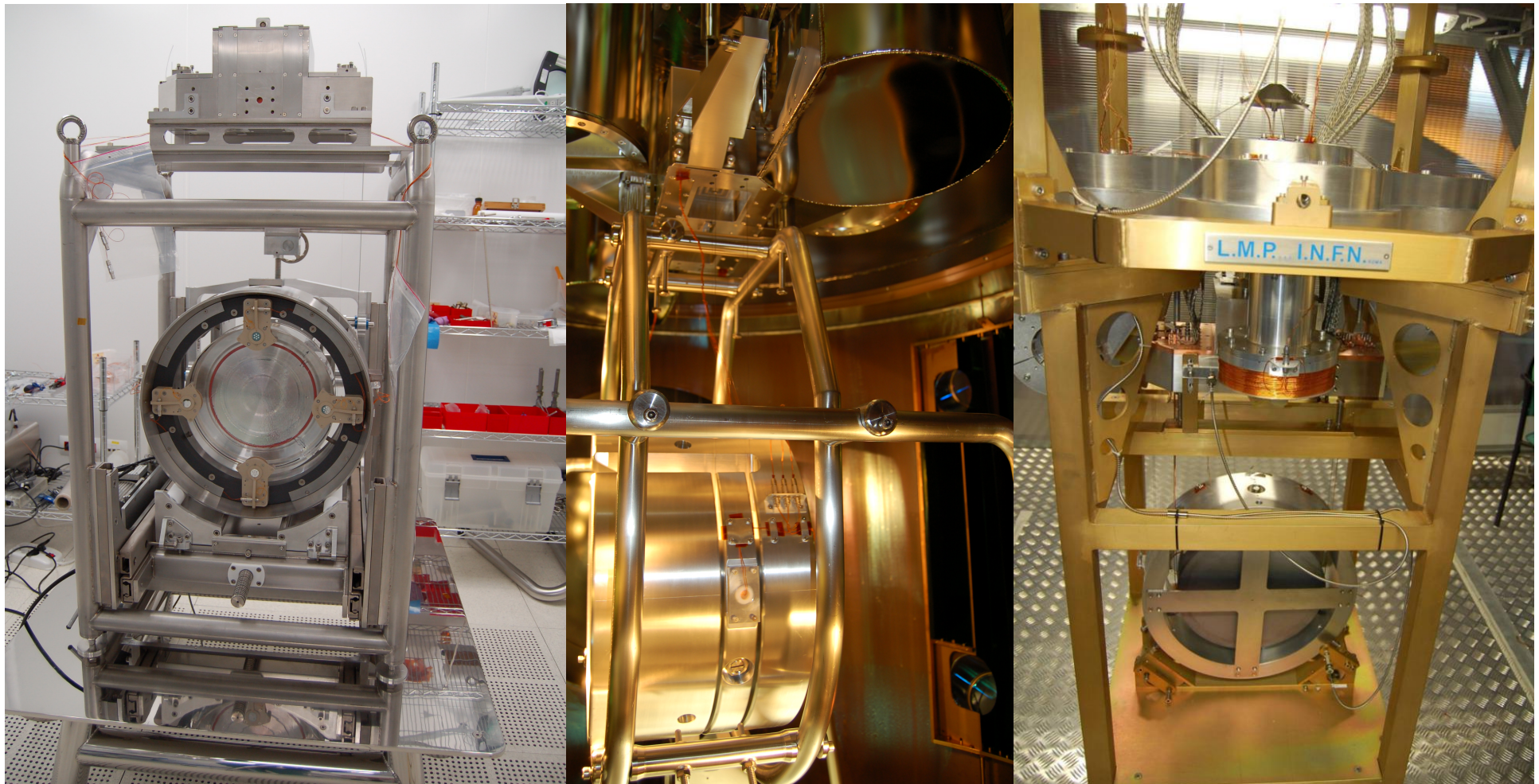
- ✓ filter tuning with a vertical cut-off around 300 mHz
- ✓ with this set-up a suspension system with a conservative cross-over frequency around 1.8 Hz (compliant with ET requirements) can be built.

Developments and alternative approaches

NIKHEF at work on GAS springs: credit J. van den Brand



The last stage of the suspension



Payload mechanical Issues

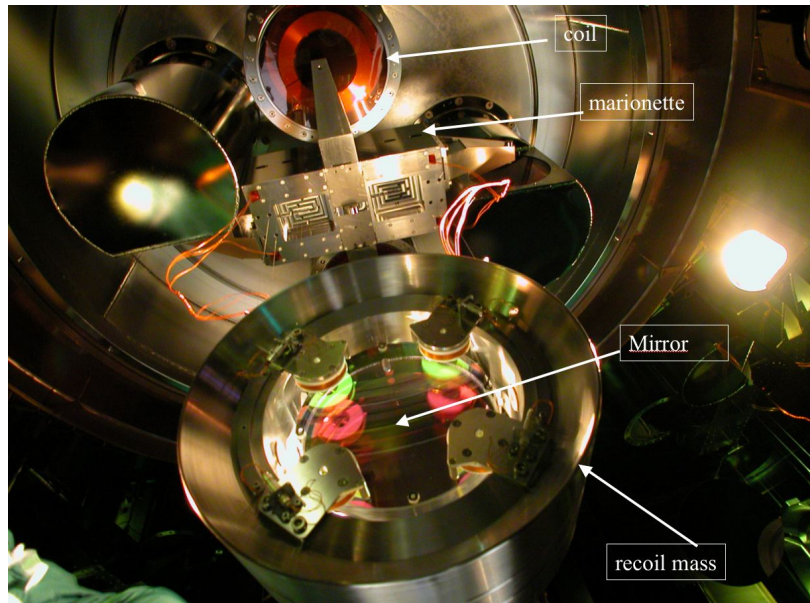
- Large Masses:
 - ✓ reduce the recoils (good for suspension thermal noise)
 - ✓ increase the violin mode frequencies (advantage for the pay control)
 - ✓ reduce the vertical modes ((trouble for the pay control)
 - ✓ excess thermal load for ET-LF
- Wire Length Increment
 - ✓ reduce the pendulum frequencies (advantage for susp. thermal noise)
 - ✓ reduce the violin modes (trouble for the pay control)
 - ✓ reduce the vertical modes trouble for the pay control)
- Wires Diameter Increment:
 - ✓ increment of the wire sections (more efficient cooling for ET-LF)
 - ✓ reduce the violin mode frequencies (trouble for the pay control)
 - ✓ reduce the dilution factor (trouble for suspension thermal noise)

Two different payloads

ET-HF

- Marionette : 400 kg
- Reaction Mass: 200 kg
- Mirror (Suprasil): 200 kg
- Mirror Diameter 62 cm
- Mirror thickness 30 cm

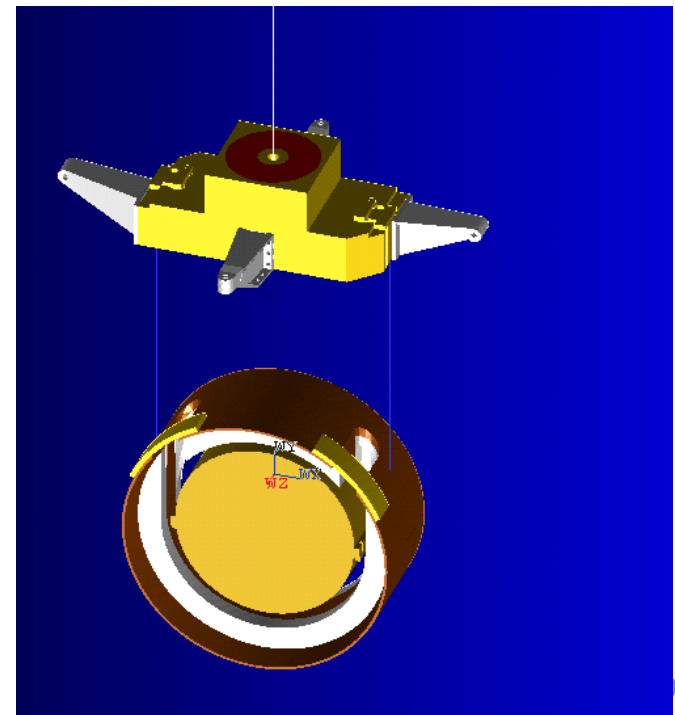
Overall payload weight: 800 kg



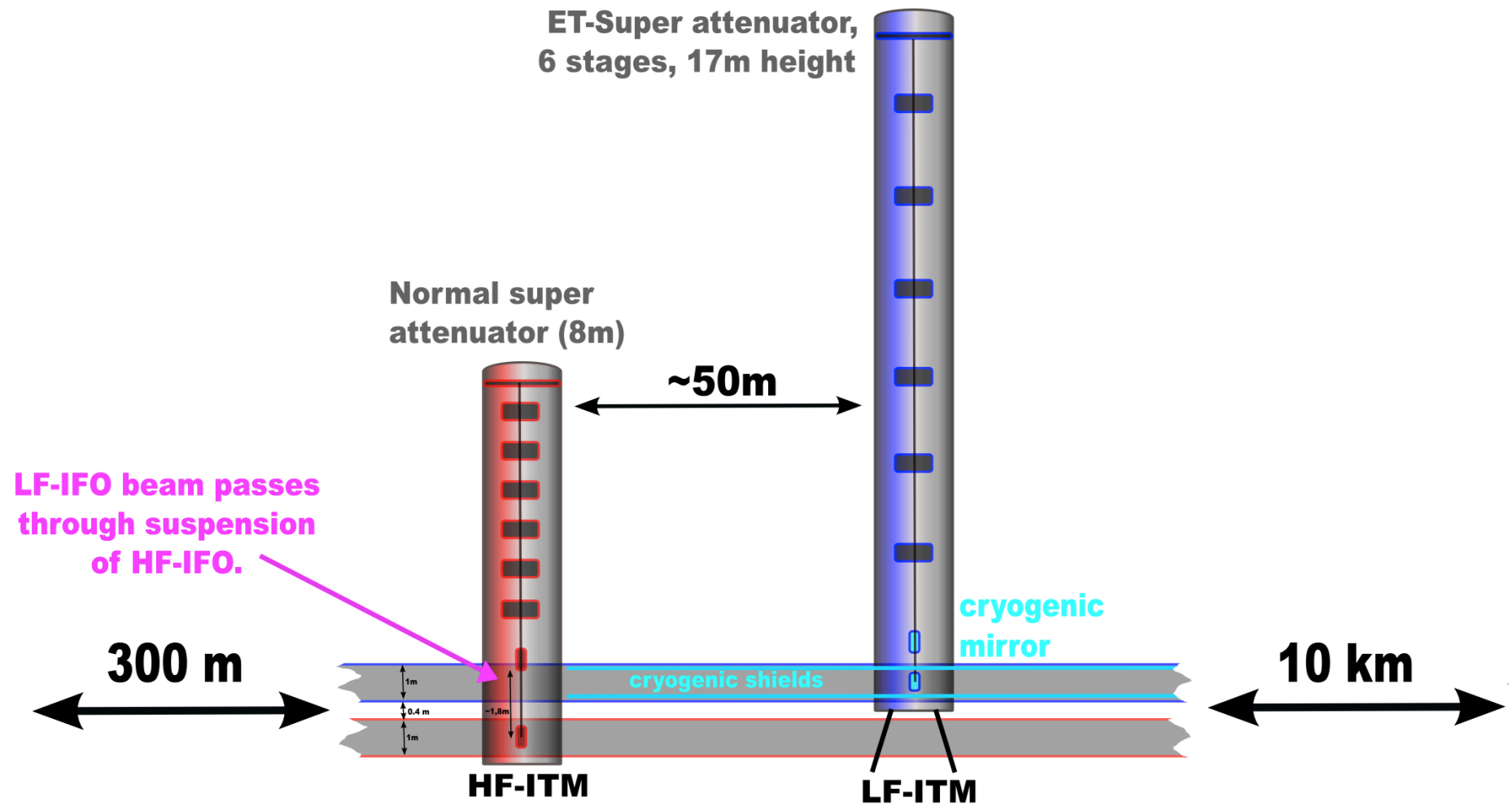
ET-LF

- Marionette : 422 kg
- Reaction Mass : 211 kg
- Mirror (Silicon): 211 kg
- Mirror Diameter 45 cm
- Mirror thickness 57 cm

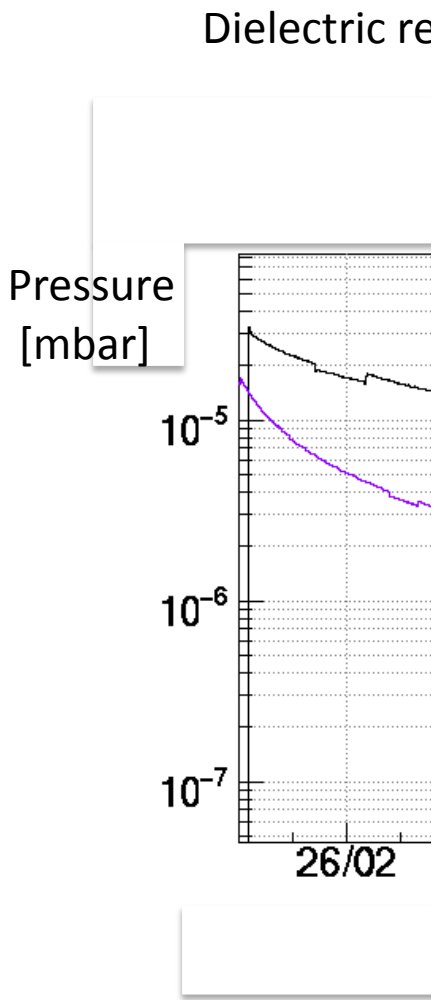
Overall payload weight: 844 kg



ET-HF payload



Material

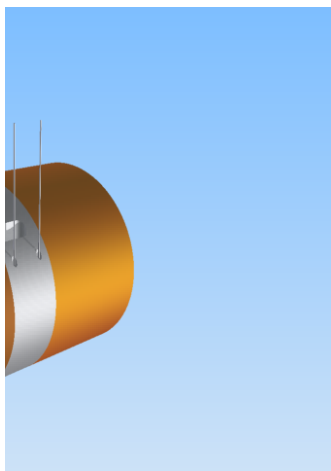


Approximate outgassing rates to use for choosing vacuum materials or calculating gas loads

(All rates are for 1 hour of pumping)

Vacuum Material	Outgassing Rate (torr liter/sec/cm ²)
Stainless Steel	6×10^{-9}
Aluminum	7×10^{-9}
Mild Steel	5×10^{-6}
Brass	4×10^{-6}
High Density Ceramic	3×10^{-9}
Pyrex	8×10^{-9}
Viton (Unbaked)	8×10^{-7}
Viton (Baked)	4×10^{-8}

nt.



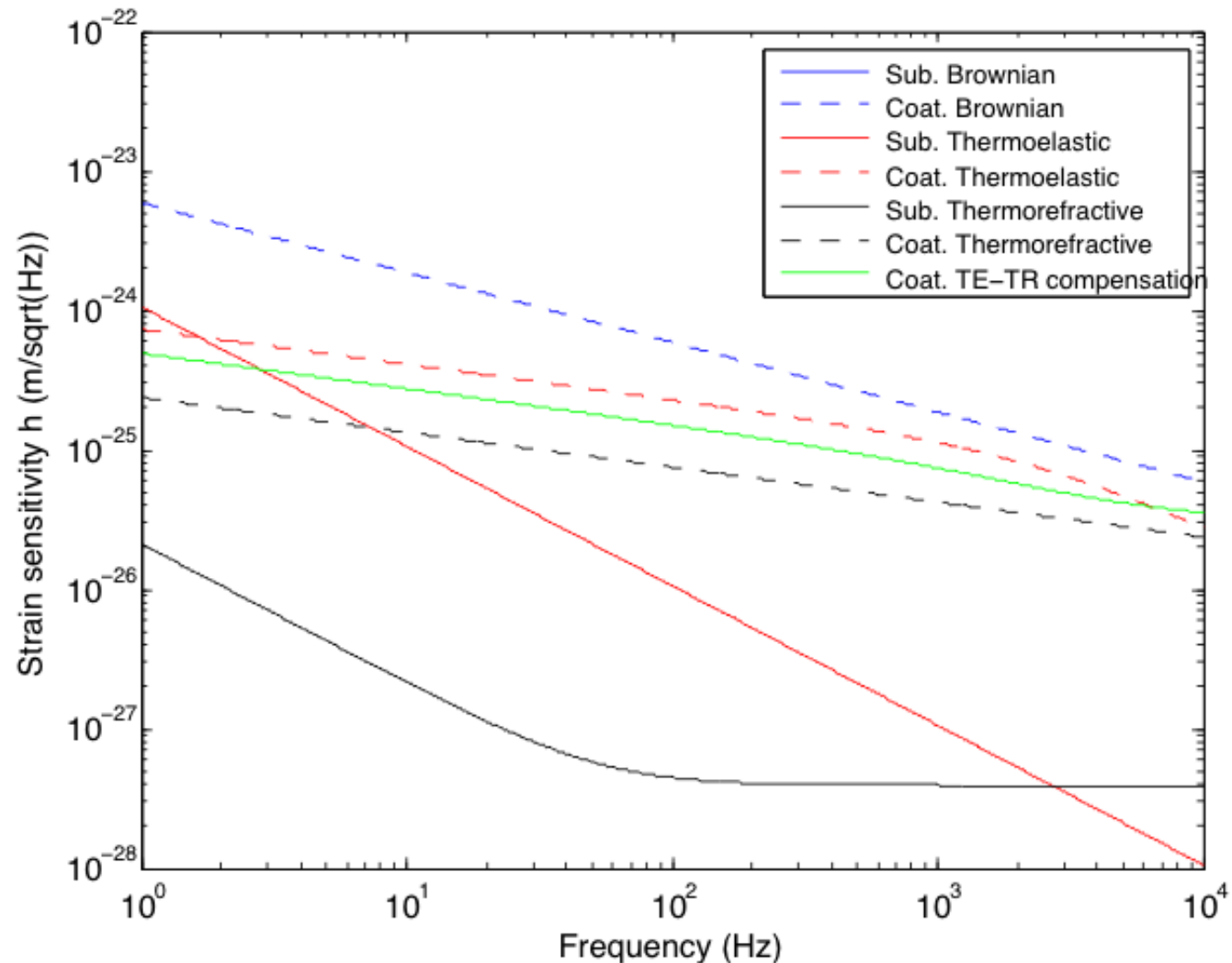
mass

ium
plastic

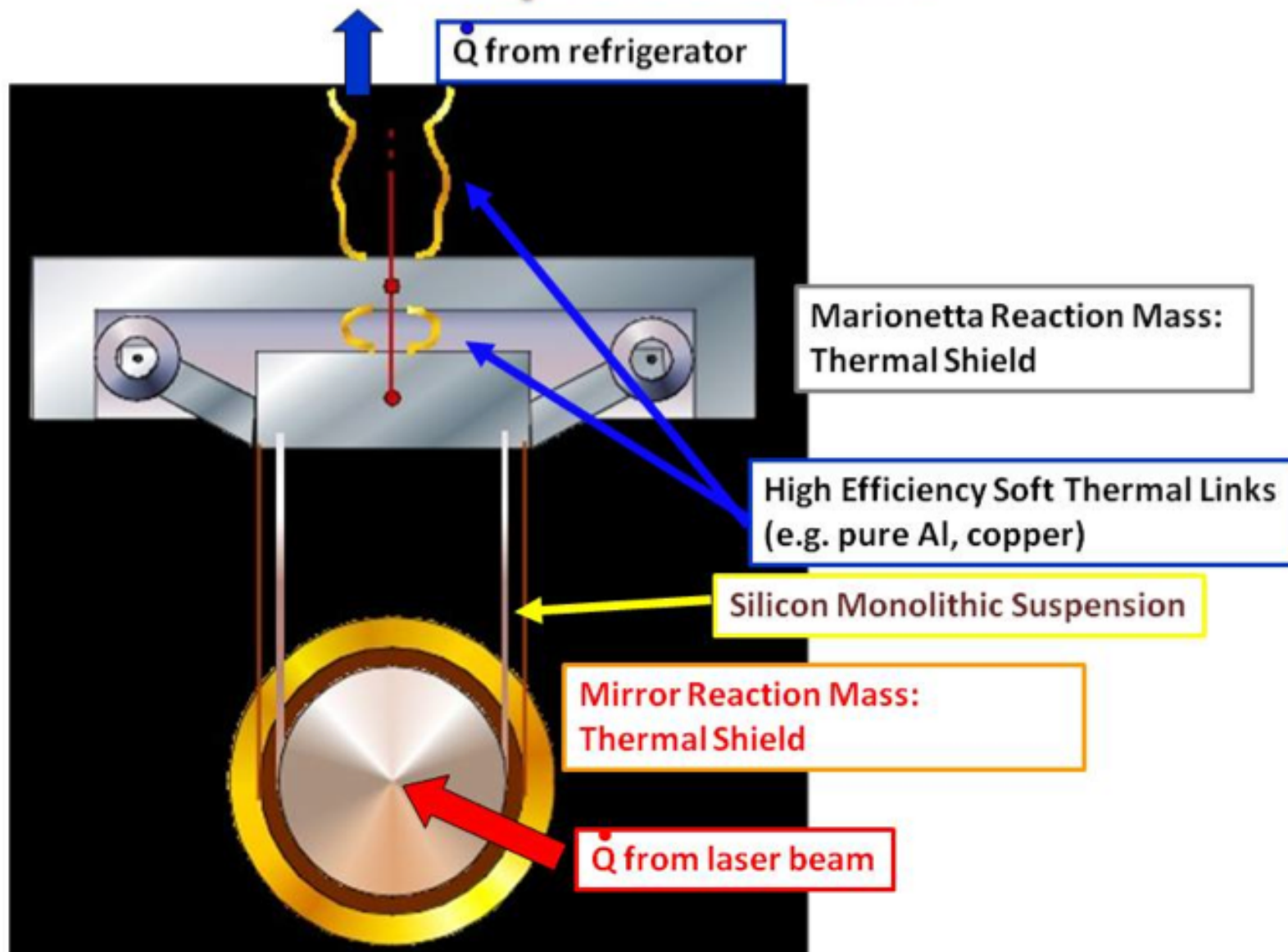
The time evolution of the vacuum pressure
for metal (purple) and dielectric RM of the payload (text carried on a VIRGO tank)

Mirror thermal noise contributions in the ET-HF case

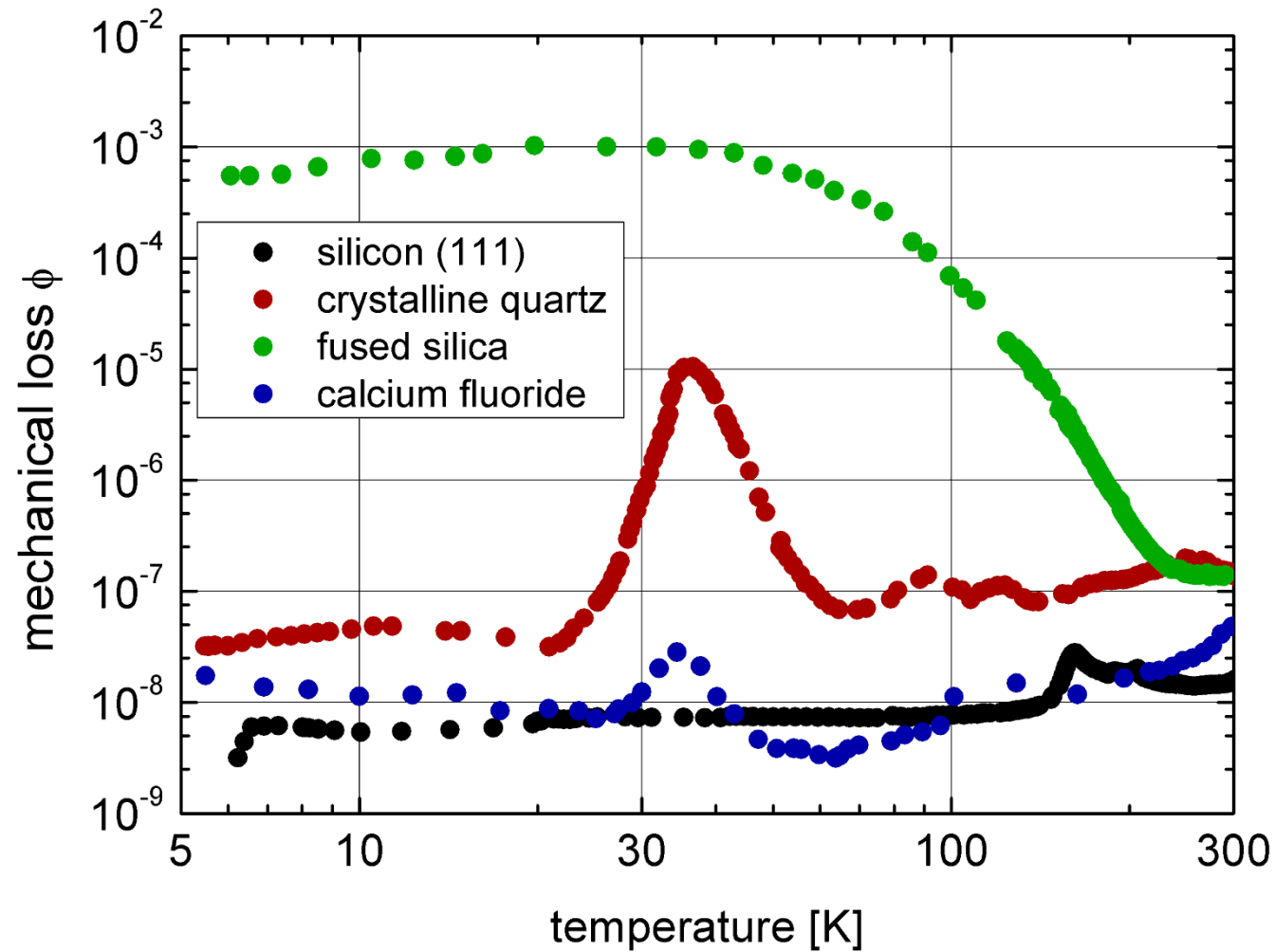
Para
temp
arm
mirror
mirror
mirror
mirror
laser wa
beam
beam
coating l
coating



ET-LF payload



Mirror bulk loss



Thermal noise – Material Choice

- Two main candidates

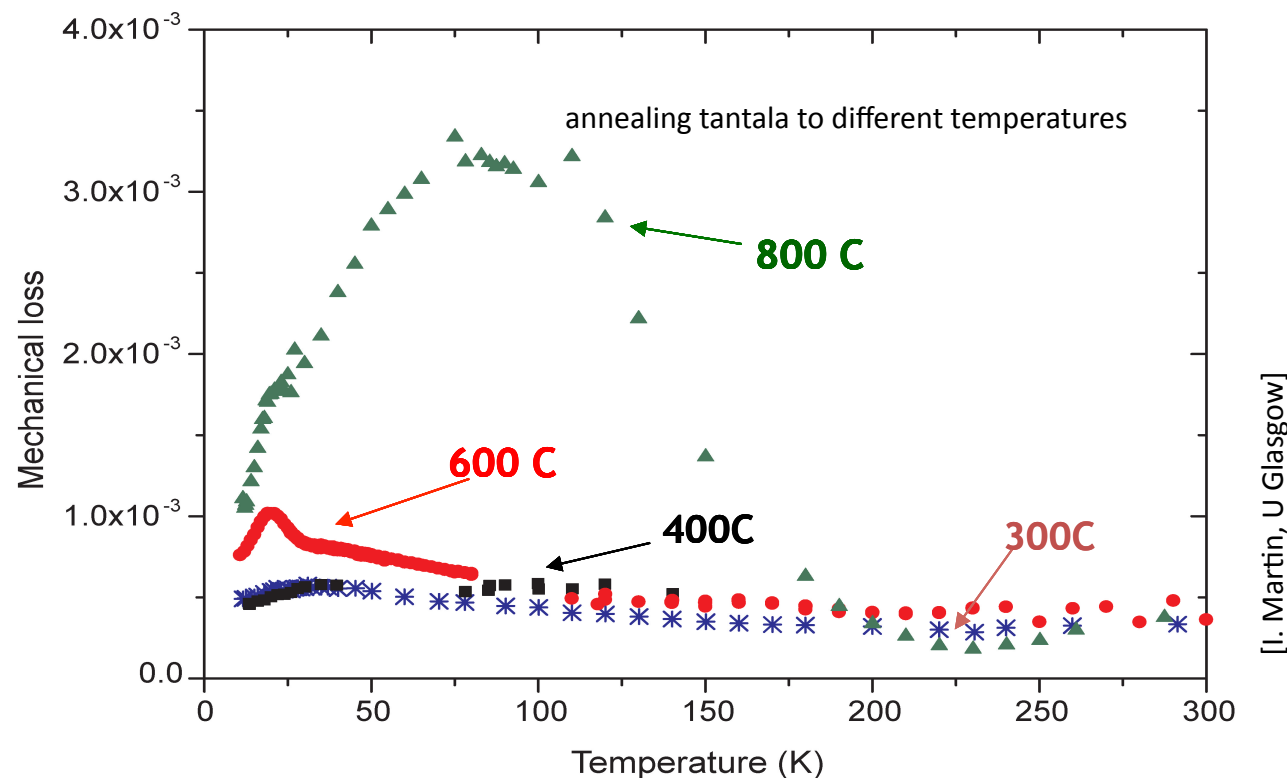
Credits to R. Nawrodt		Sapphire	Silicon
	mechanical loss	++	++
	mechanical strength	+(+)*	++
	optical material	+	0
	thermal conductivity	++	++
	polishing	-	+
	size availability	-...+	+...++ (semicond. industry)



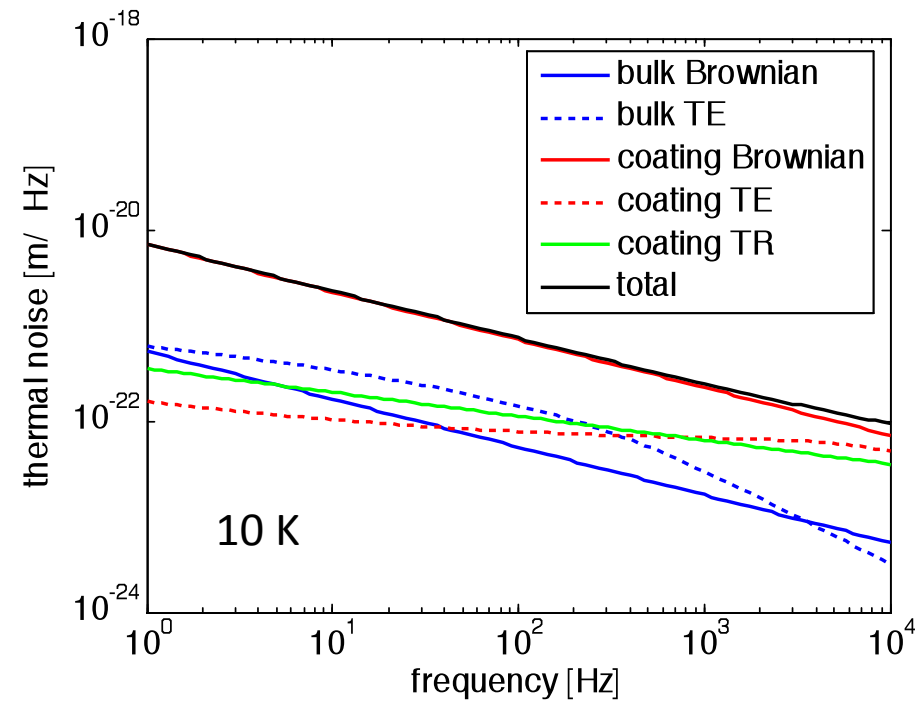
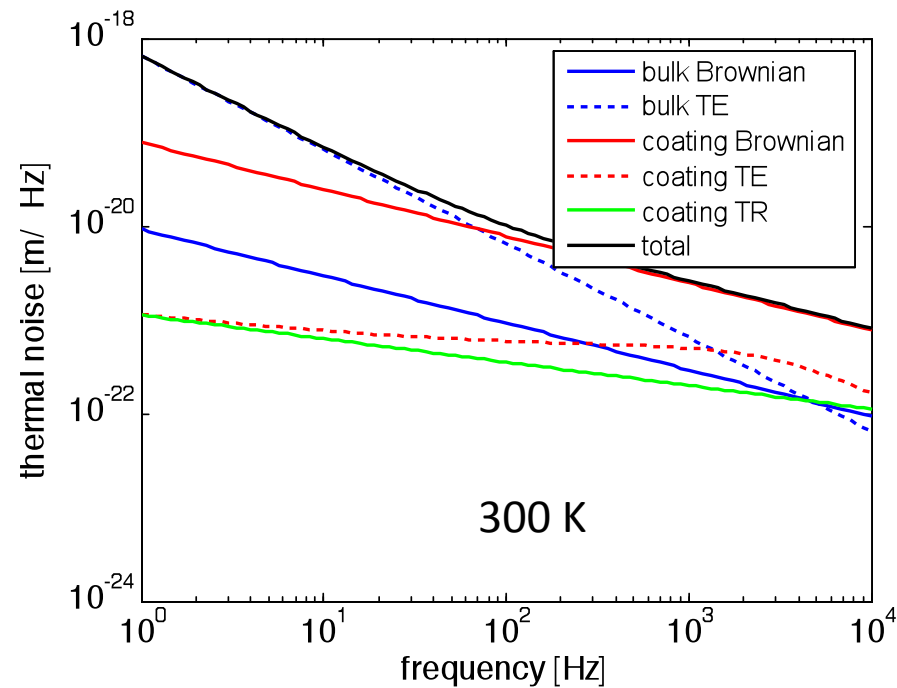
* Bonding method should require further R&D: strength might not be higher enough

Mirror thermal noise – Coatings

- Coating thermal noise dominates all other thermal noise sources of the mirrors in current detectors (coating = amorphous)
- R&D ongoing effort to understand loss mechanisms

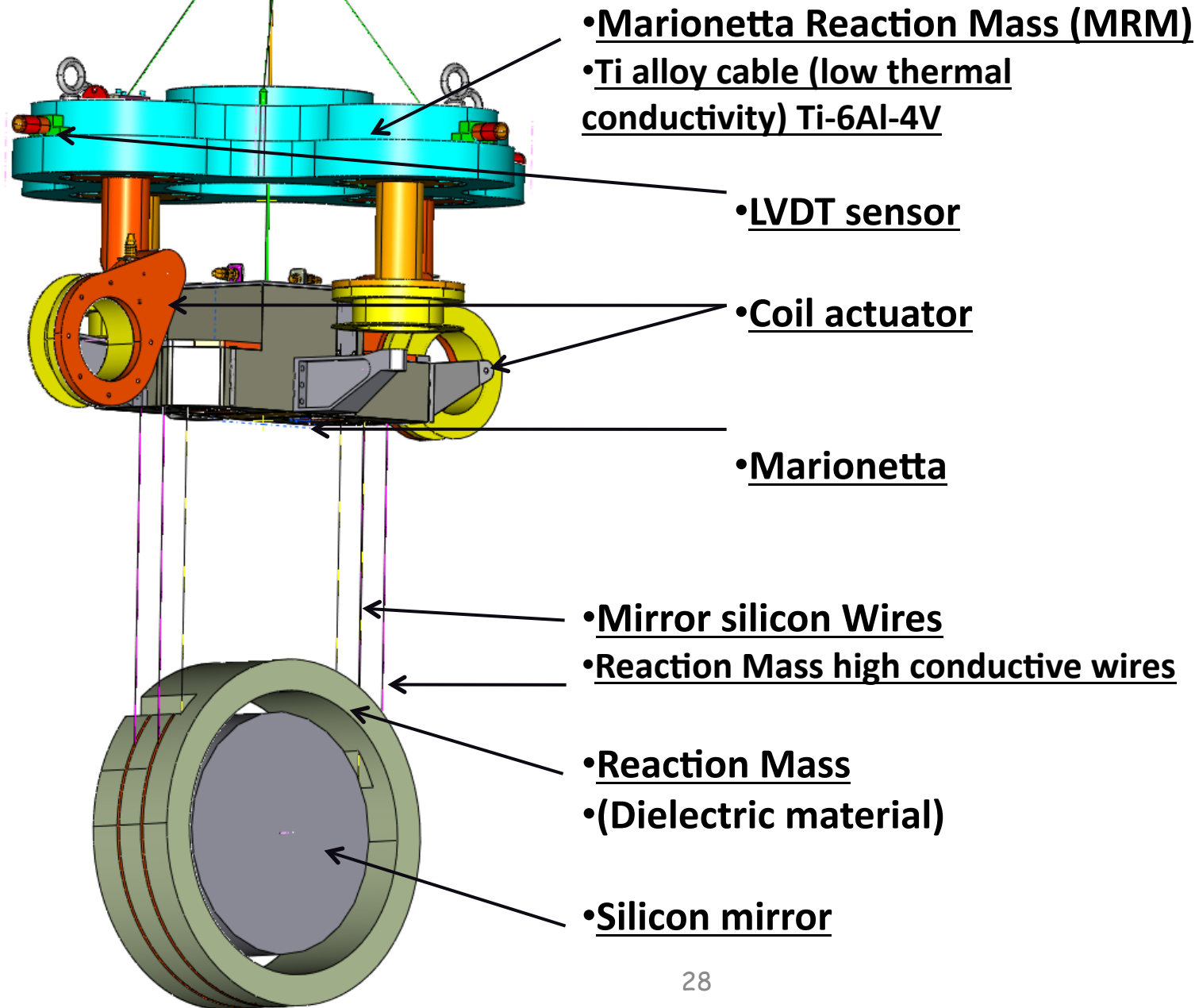


Mirror thermal Noise – silicon case

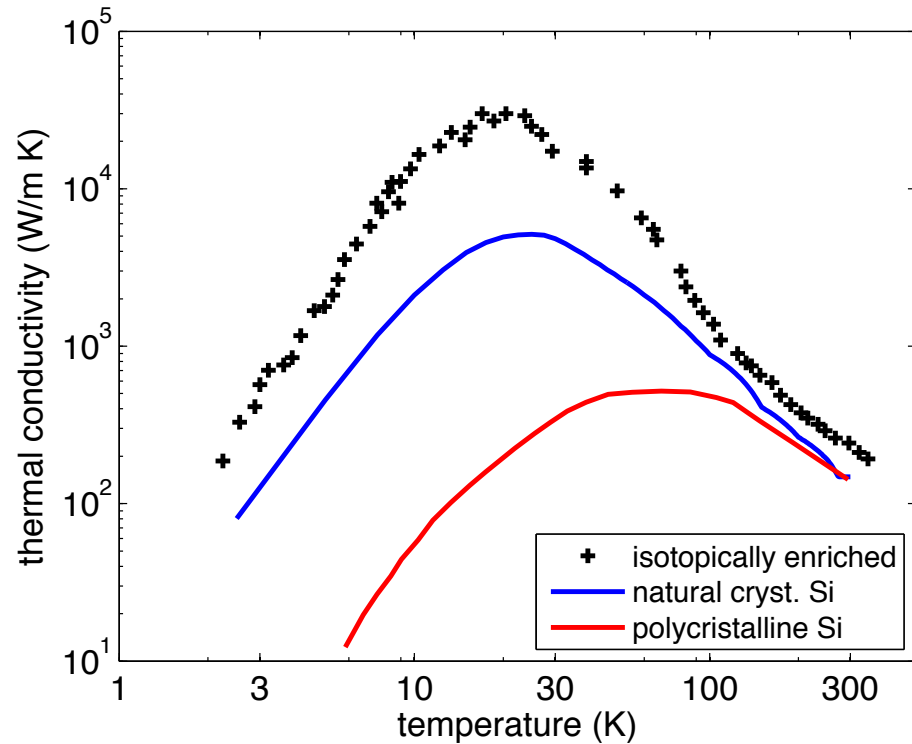


Coating 18 doublets, $\text{Ta}_2\text{O}_5:\text{TiO}_2$, SiO_2
Beam diameter $w = 90$ mm

The Cryogenic Payload



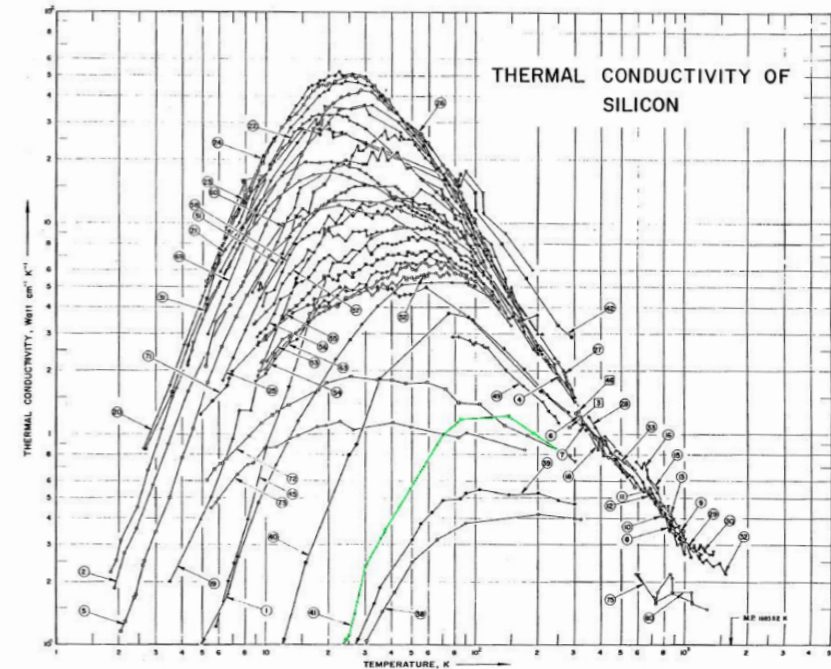
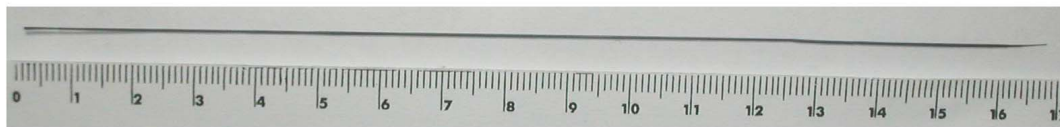
Silicon fiber : the thermal conductivity



Pure Si $L \approx 21$ cm



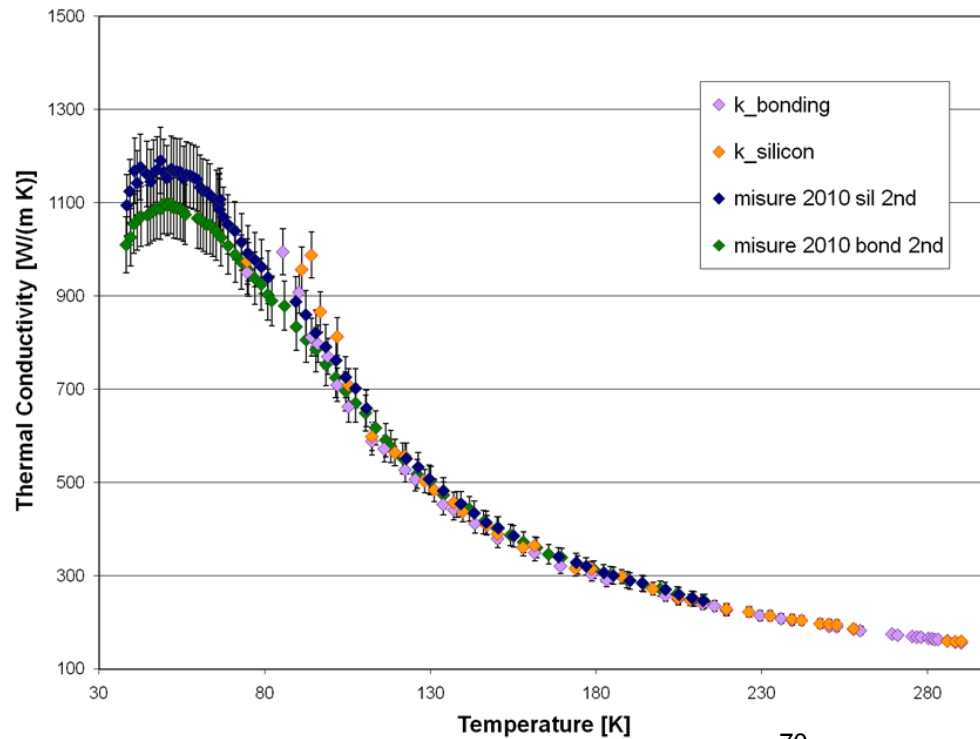
Pure Si $L \approx 17$ cm



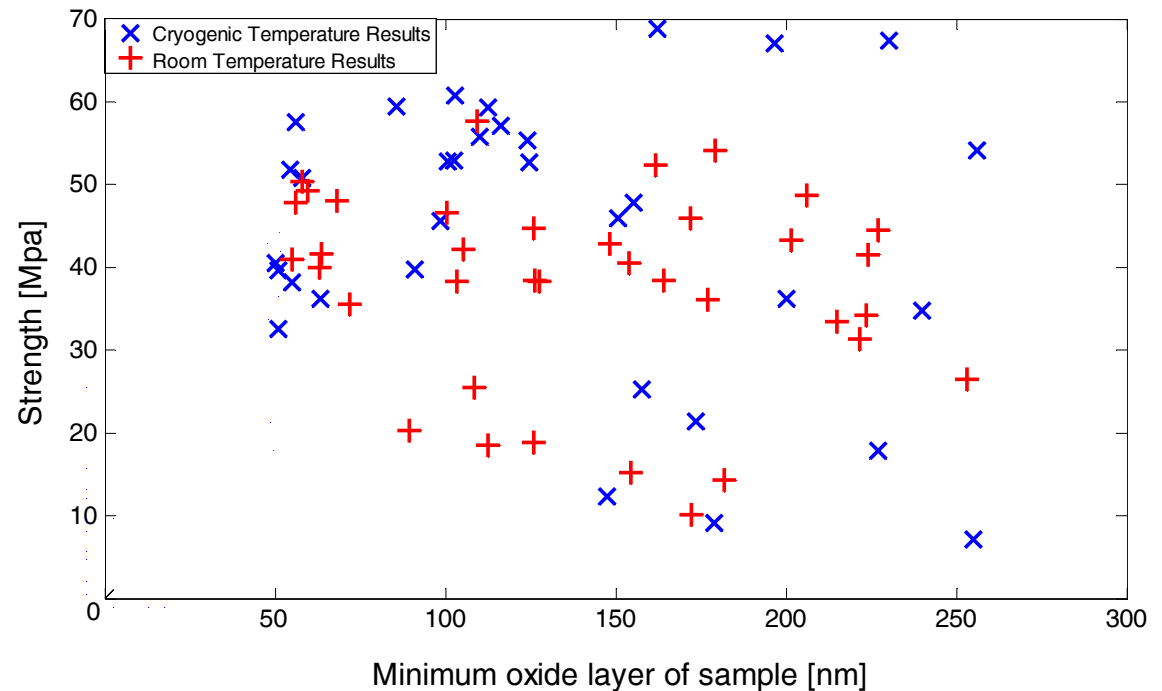
Silicon sample prepared by micropulling

R&D is required:
Fiber payload
 $\sim 60/70$ cm long

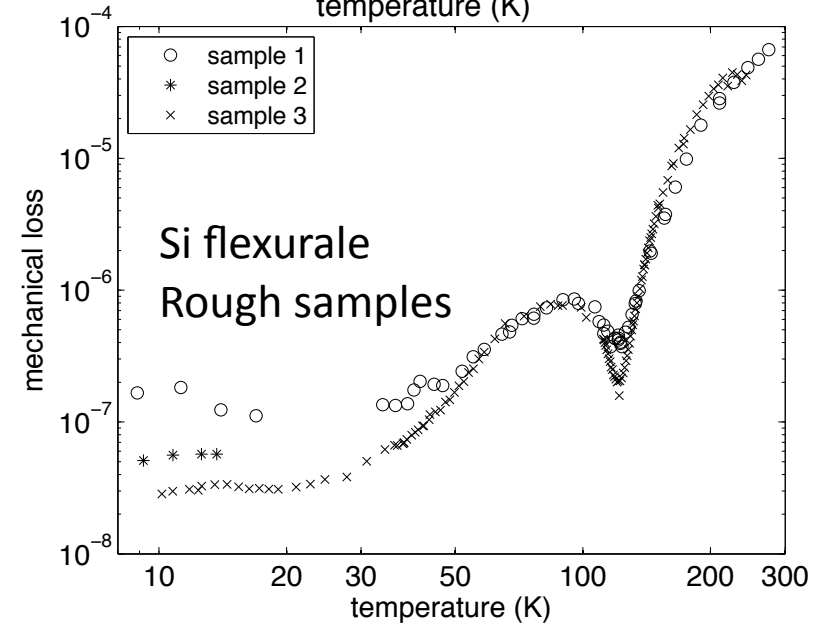
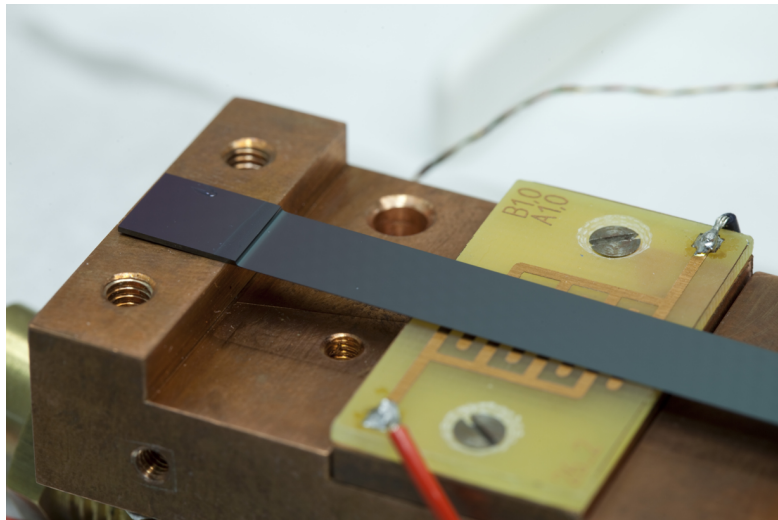
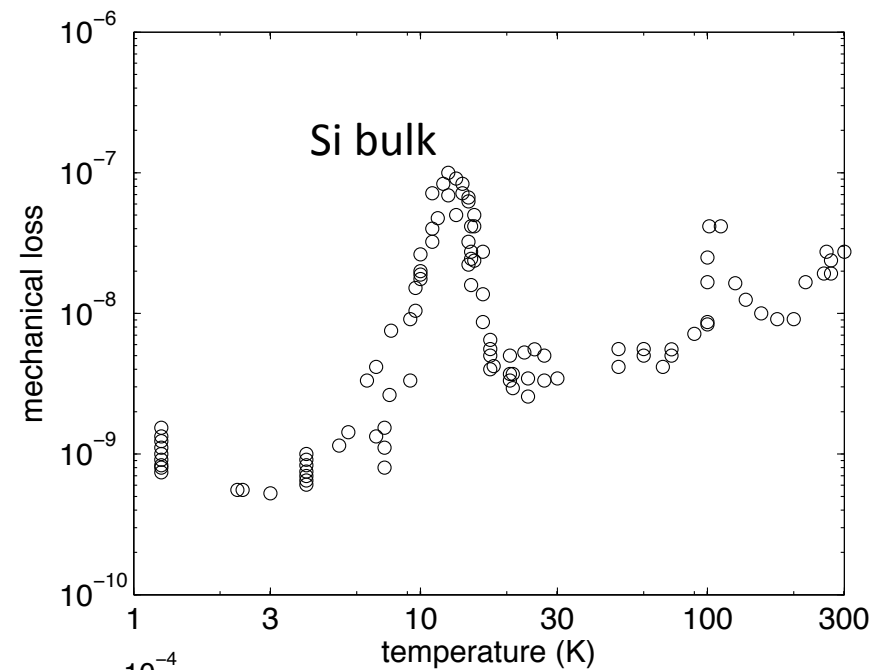
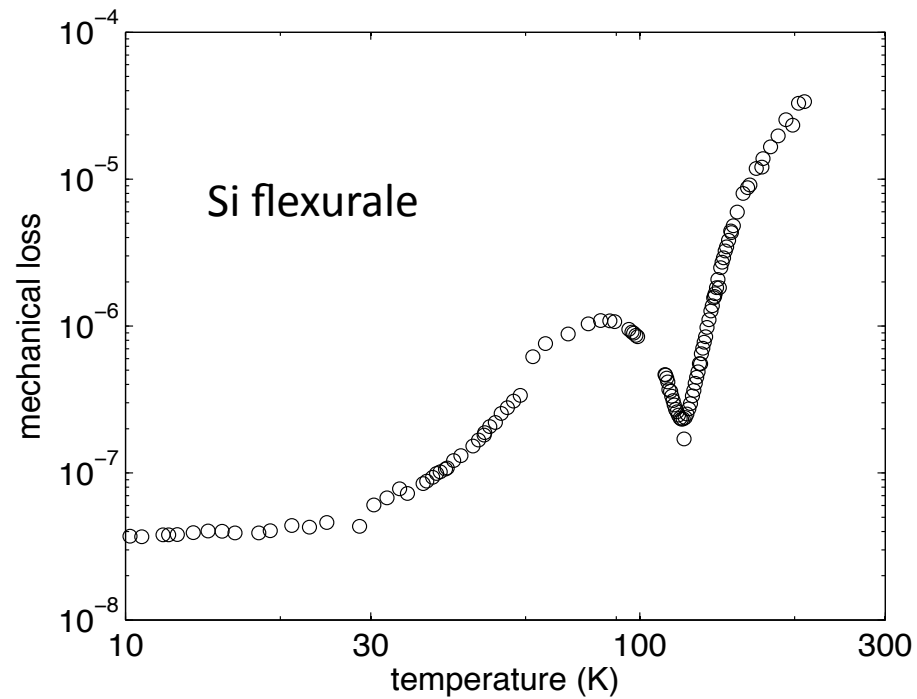
Silicon Bonding: Thermal conductivity and strength



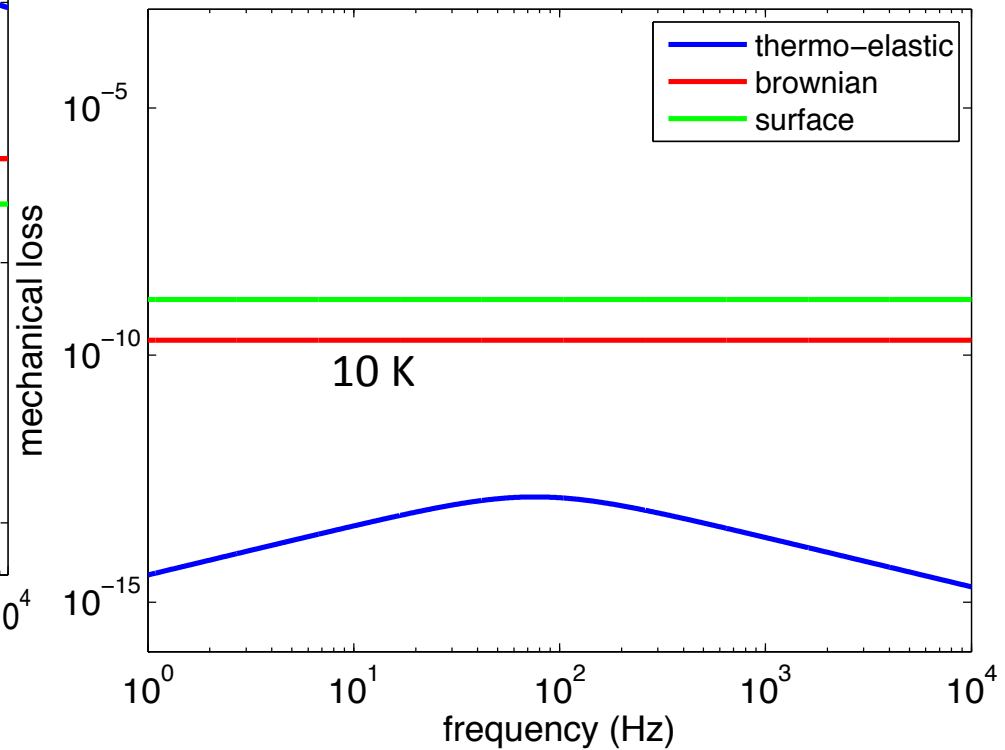
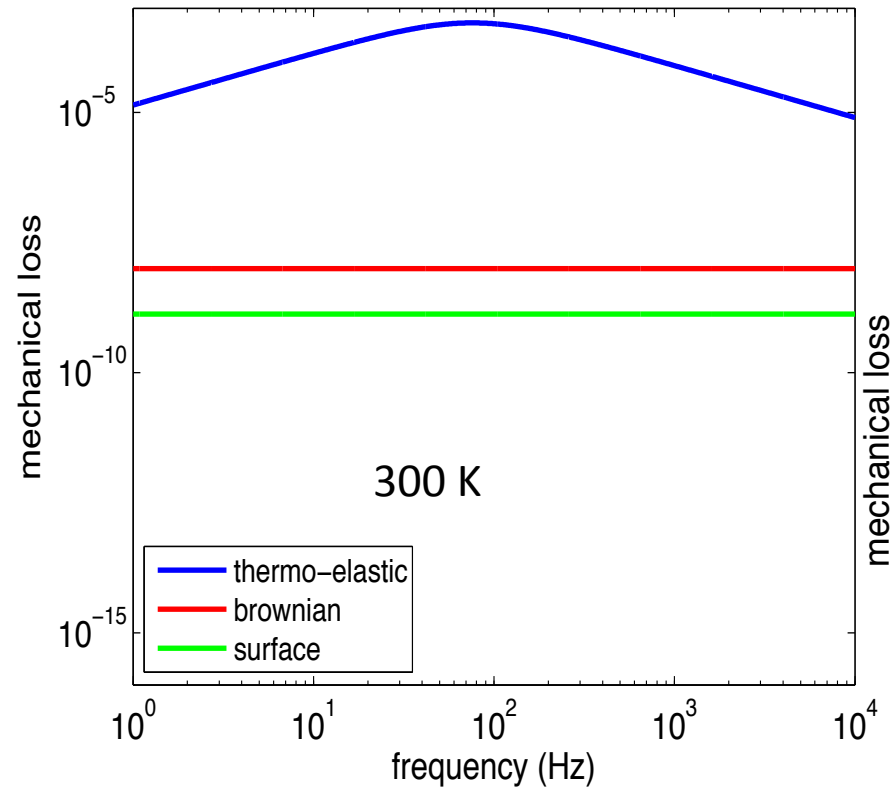
N. Beveridge talk on Thursday



Silicon acoustic losses



Acoustic loss contributions vs temperature



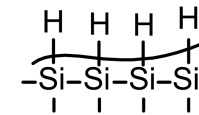
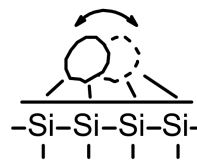
(a)

(b)

(c)

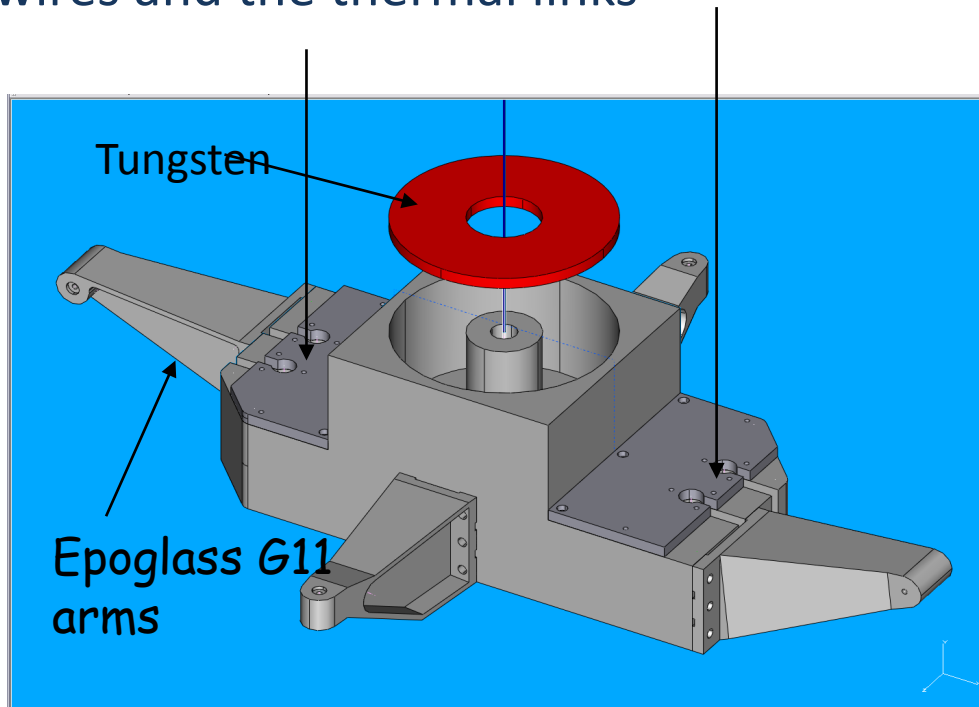
(d)

Surface loss mechanisms

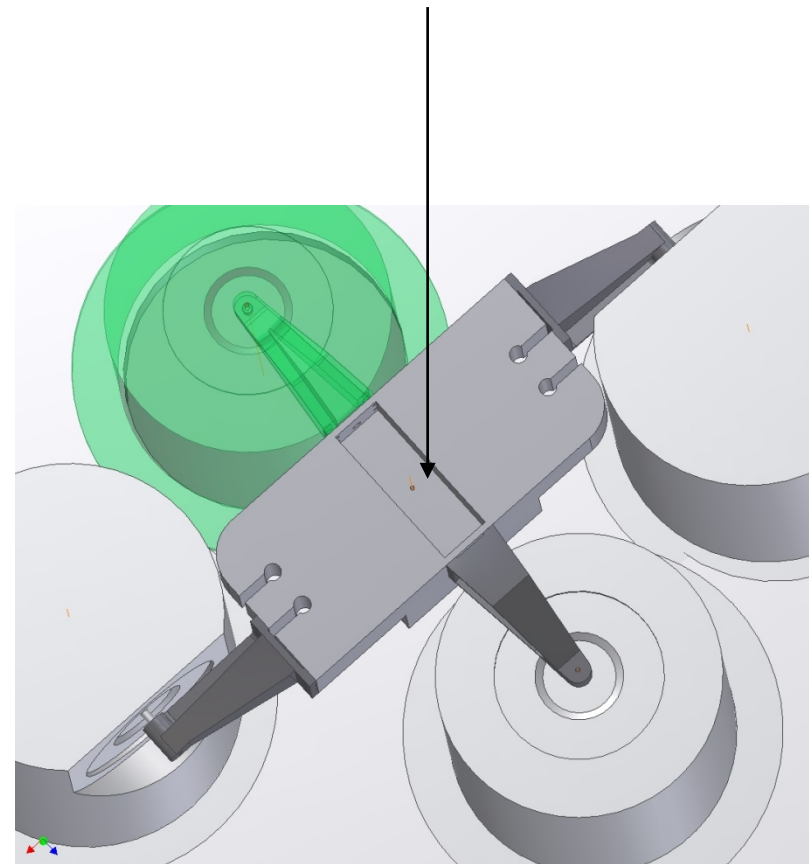


Marionette for the cryogenic environment

- Body in amagnetic Steel (AISI316L)
- Tungsten (or CuW) insert
- Epoglass arms G11 (suitable for cryo applications)
- Copper plate to clamp the suspension wires and the thermal links

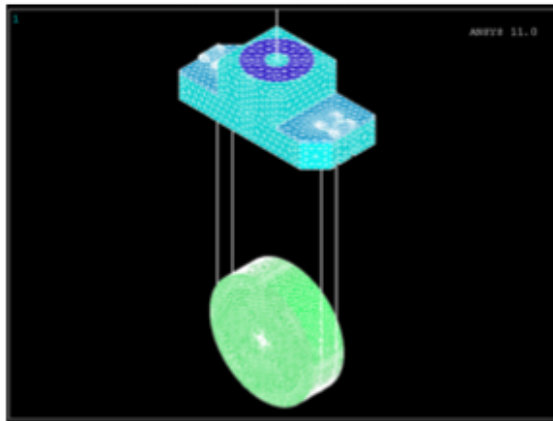


Mass for balancing the marionette by an electric motor



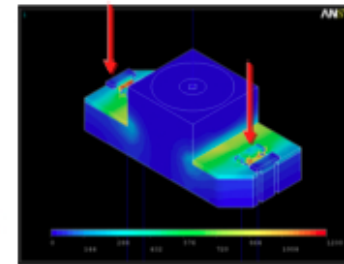
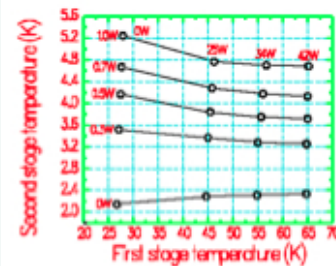
Thermal Simulation

The model

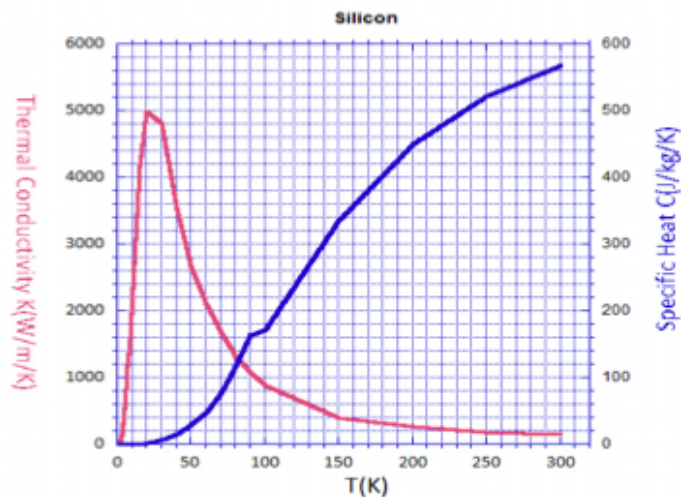


Boundary Conditions

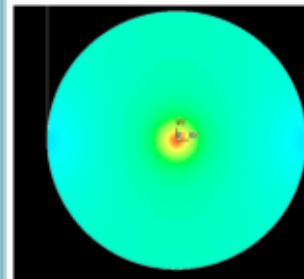
- Cryocooler Second stage link to the marionetta



P_{cooler} depends on the temperature (Cryomec PT curve used) with 2 coolers the extracted power at 5 K is 1 W



Laser power on the mirror



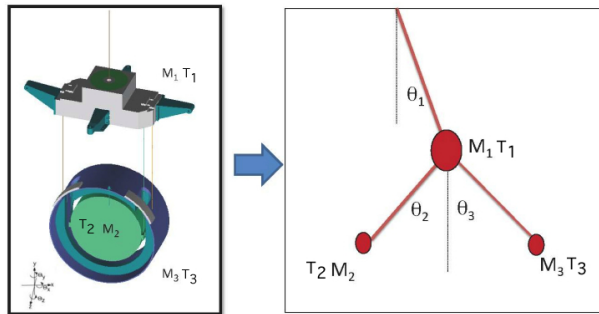
Gaussian beam

$$P_{\text{abs}} = 1\text{W}$$

The power absorbed by the coating is dominant

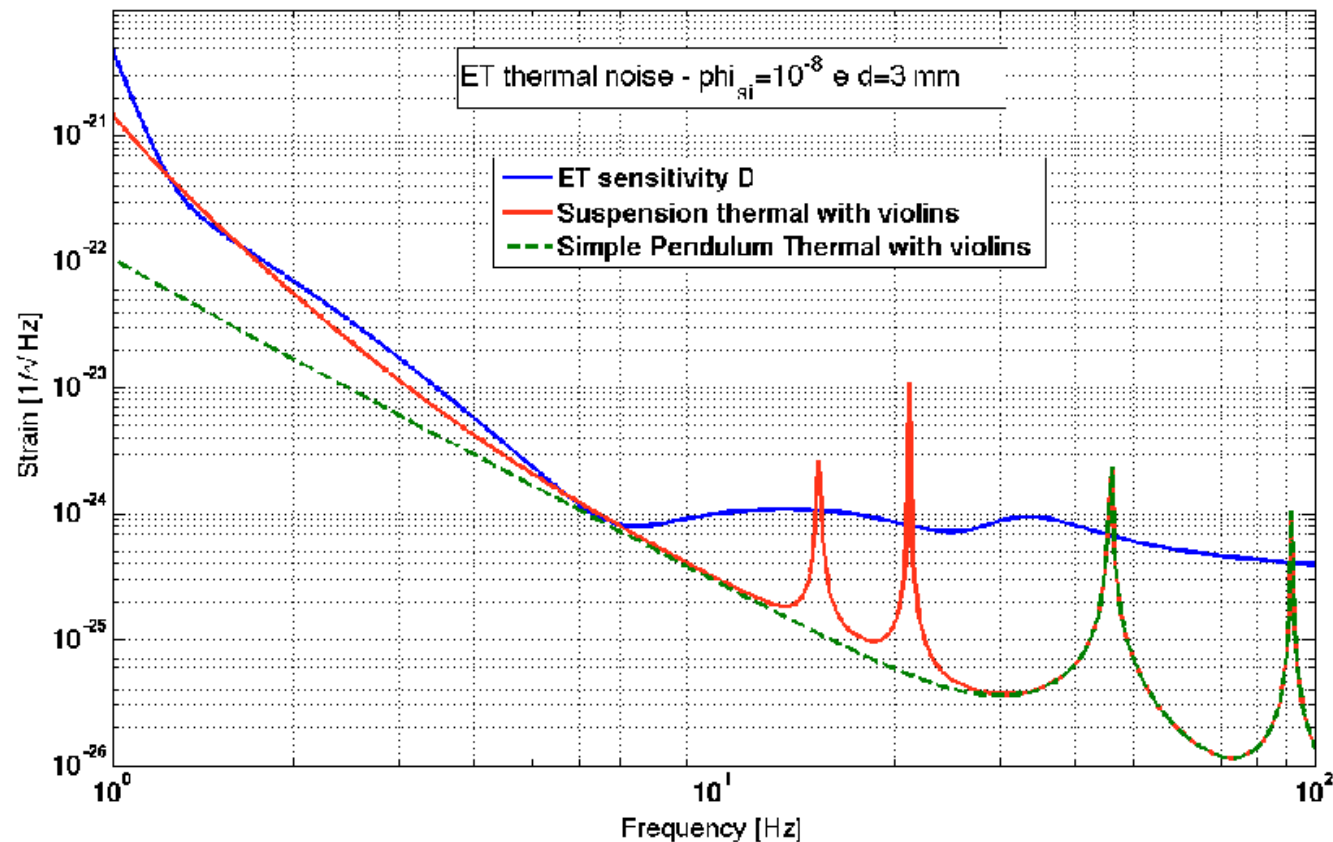
Starting point:

- thermalised system at 10 K
- laser power on
- surrounding system at 10K



ET-LF suspension thermal noise

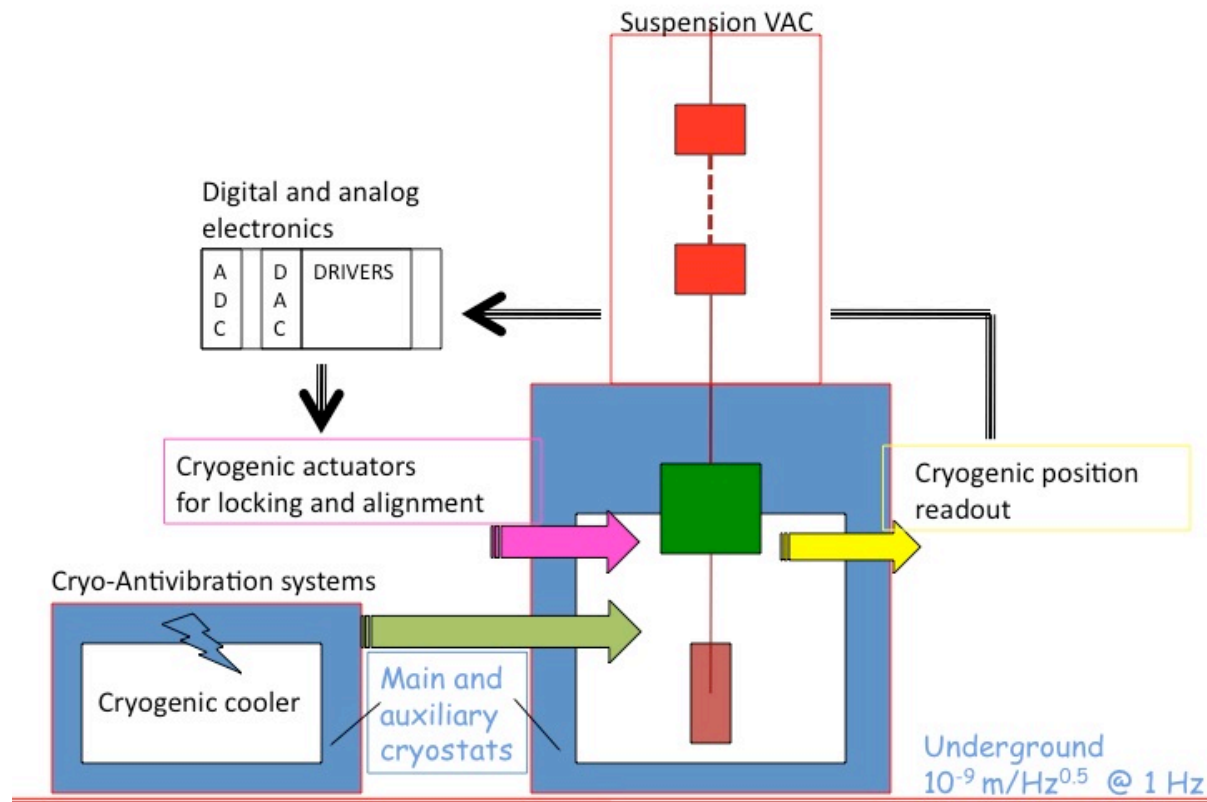
Based on the model,
which includes coupled oscillators at different temperatures



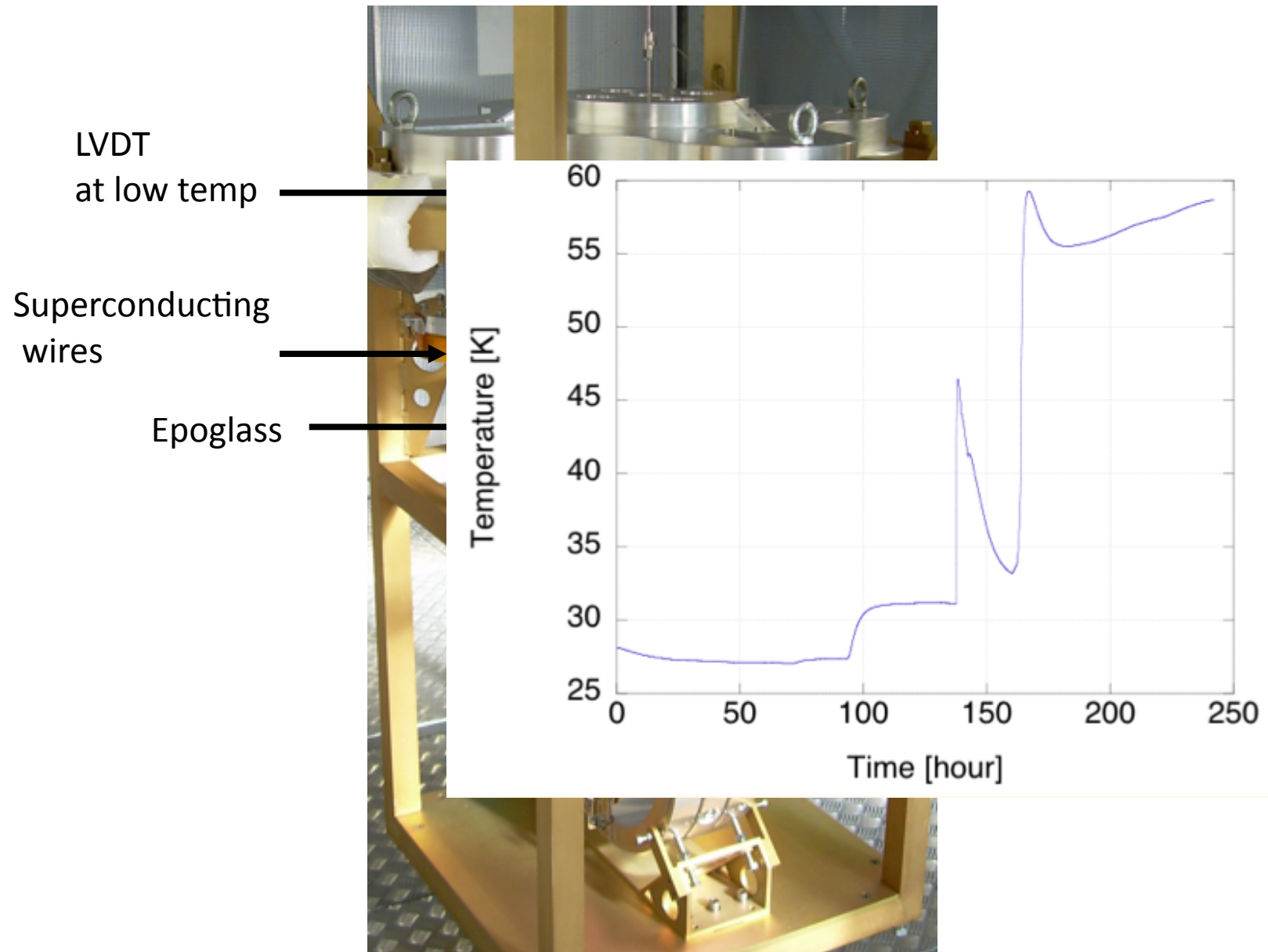
Suspension control: sensors and actuators

Coil - Magnet actuators: design according to constraints given by locking.

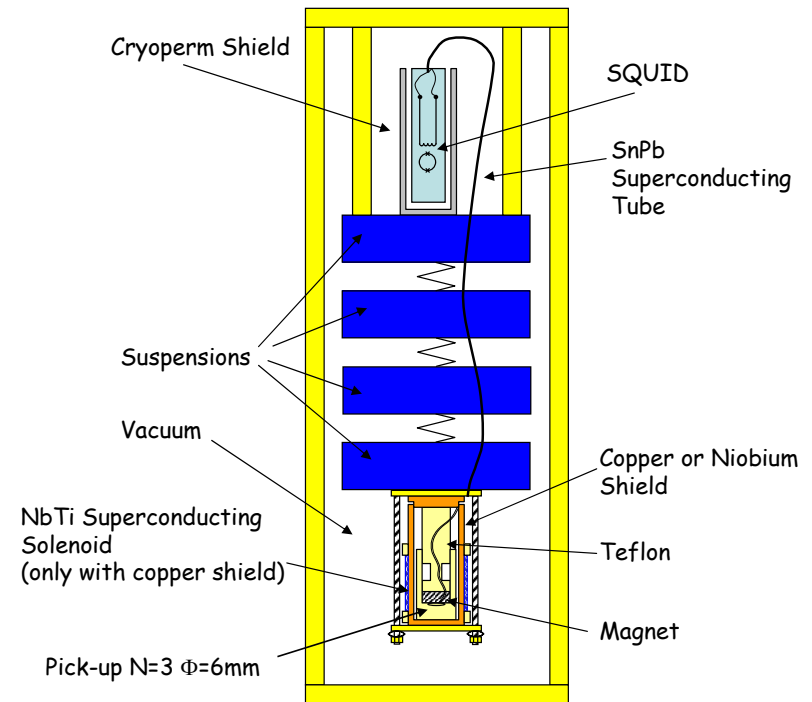
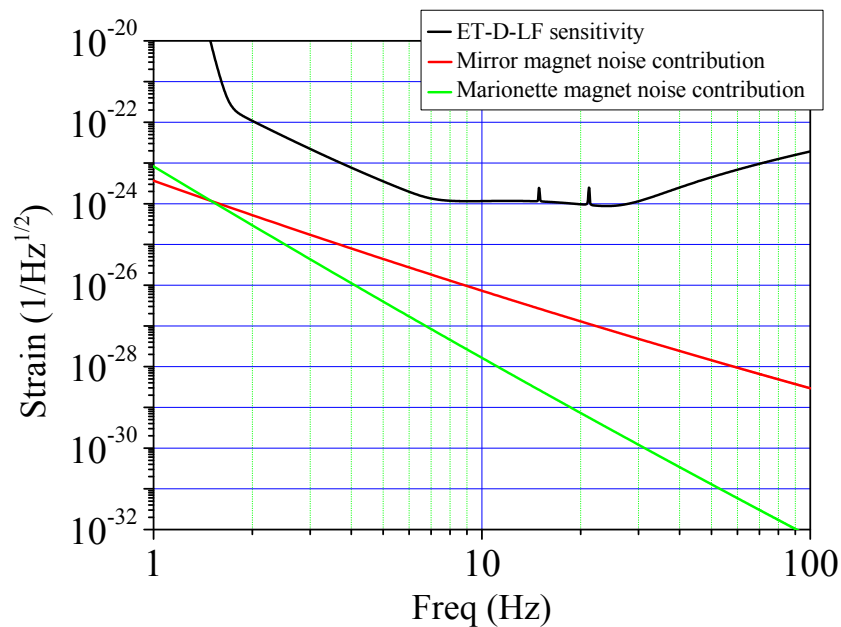
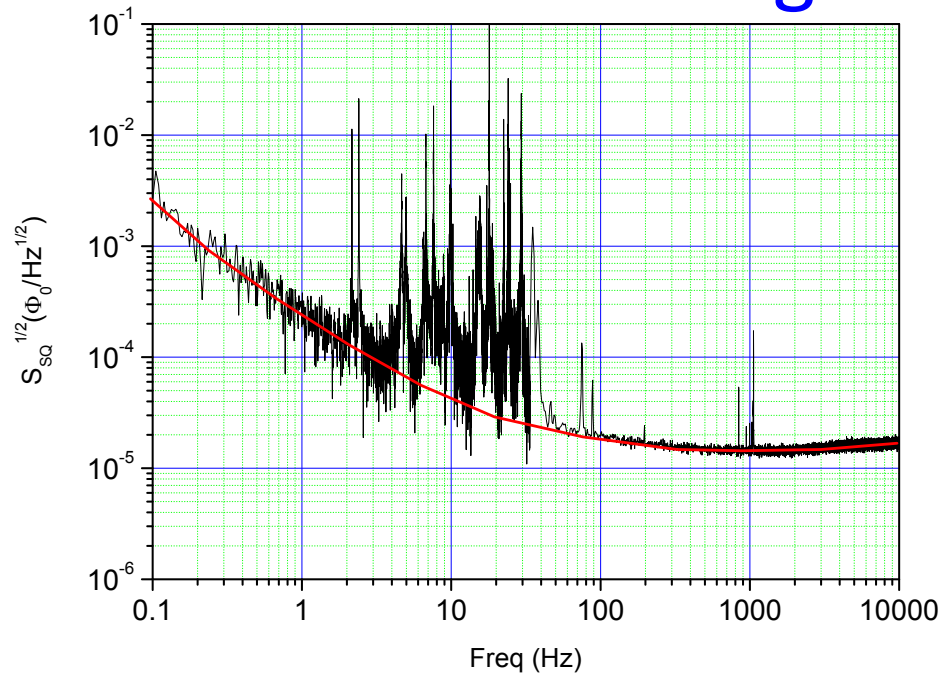
Electrostatic actuators easily adapted



Solutions from the previous R&D experience



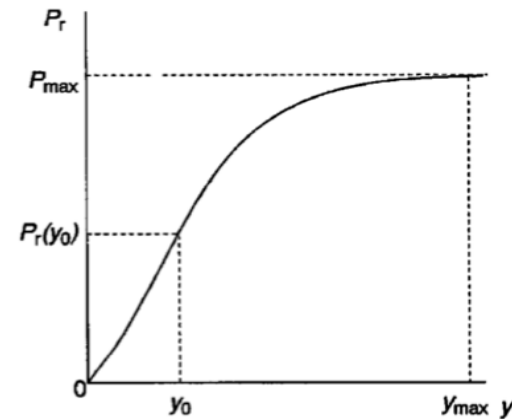
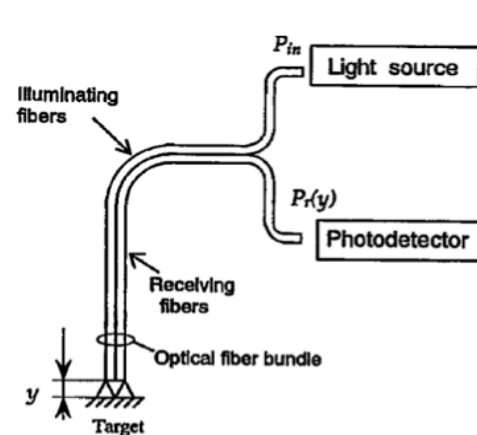
Coil-magnet actuation noise



Barkausen noise measurements
on a SmCo magnet at 4 K

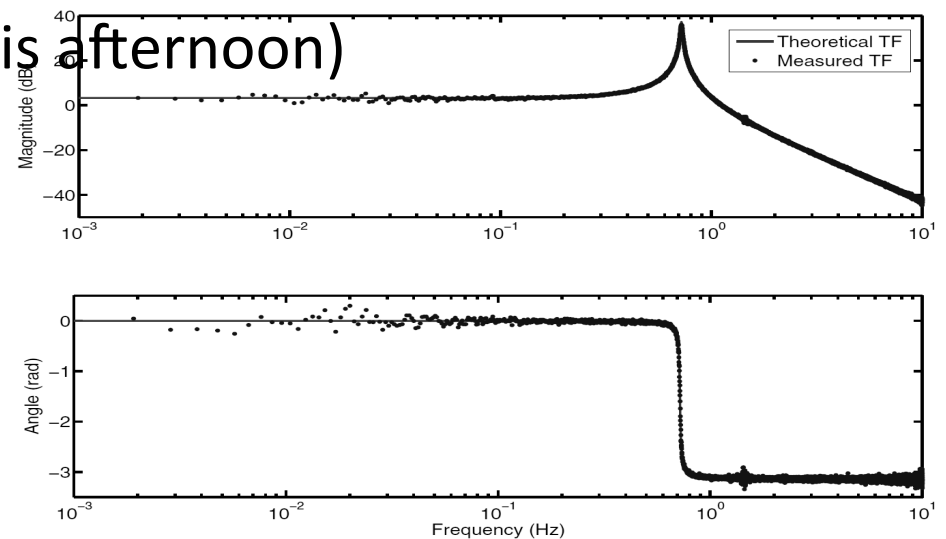
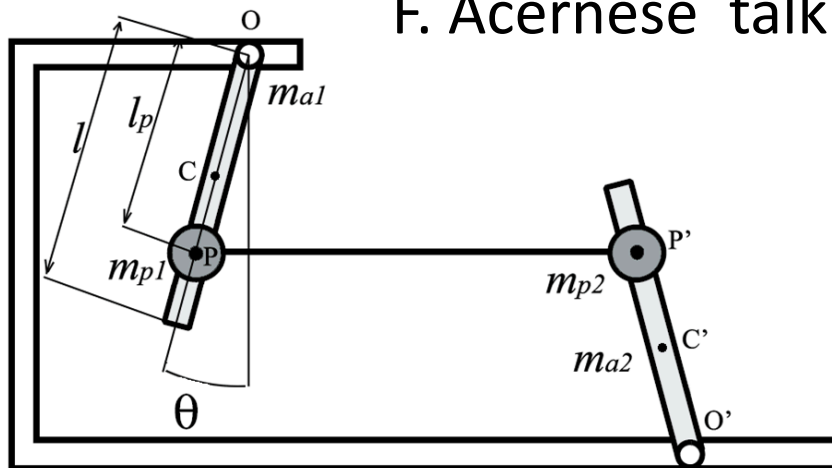
P. Falferi Class. Quant. Grav. in press (2011)

Displacement sensors at low temperature



New Inertial sensors for suspension control

F. Acernese talk (this afternoon)

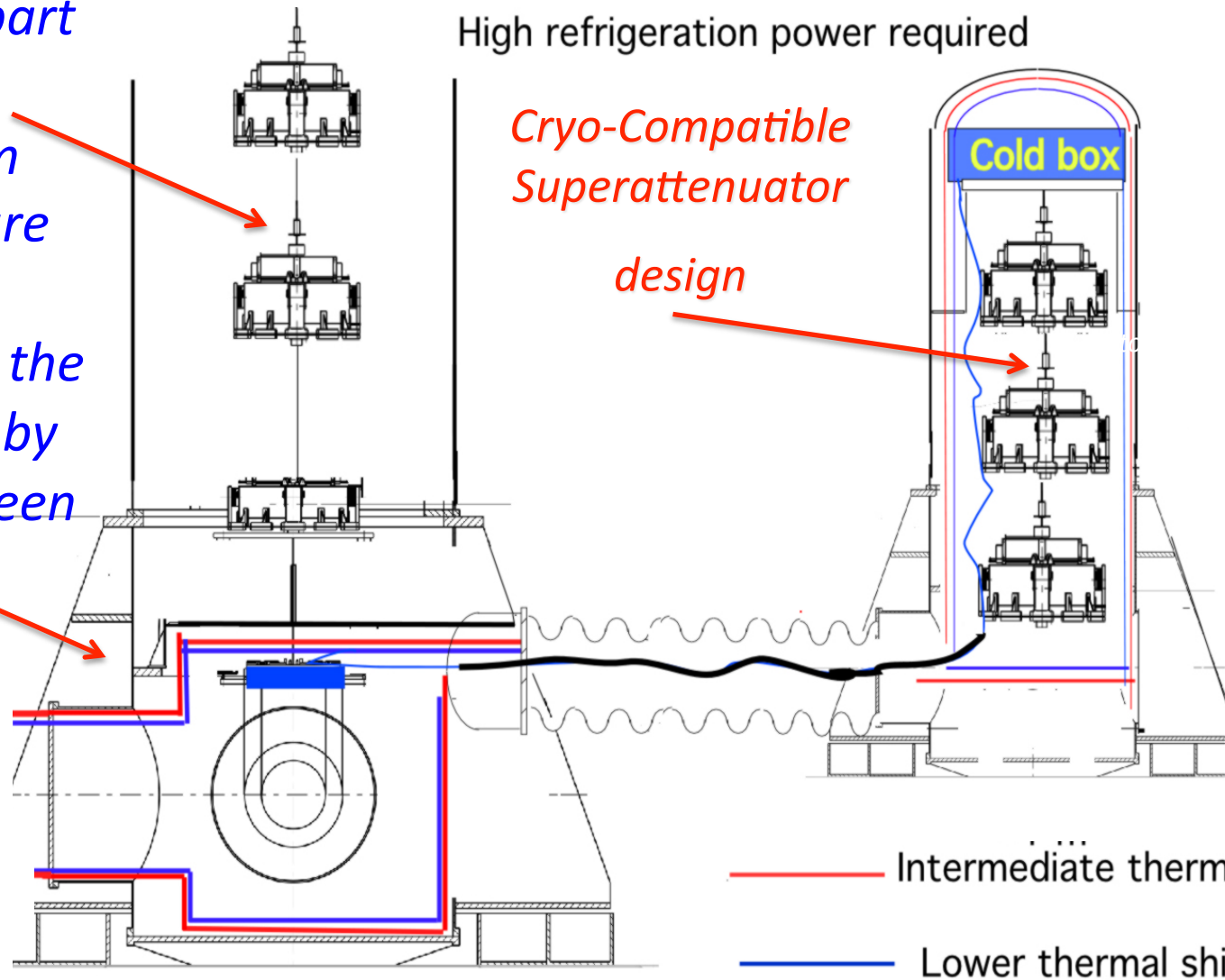


Design of the cooling system

*The upper part
(~15 m)
is at room
temperature
and
insulated to the
lower part by
thermal screen*

High refrigeration power required

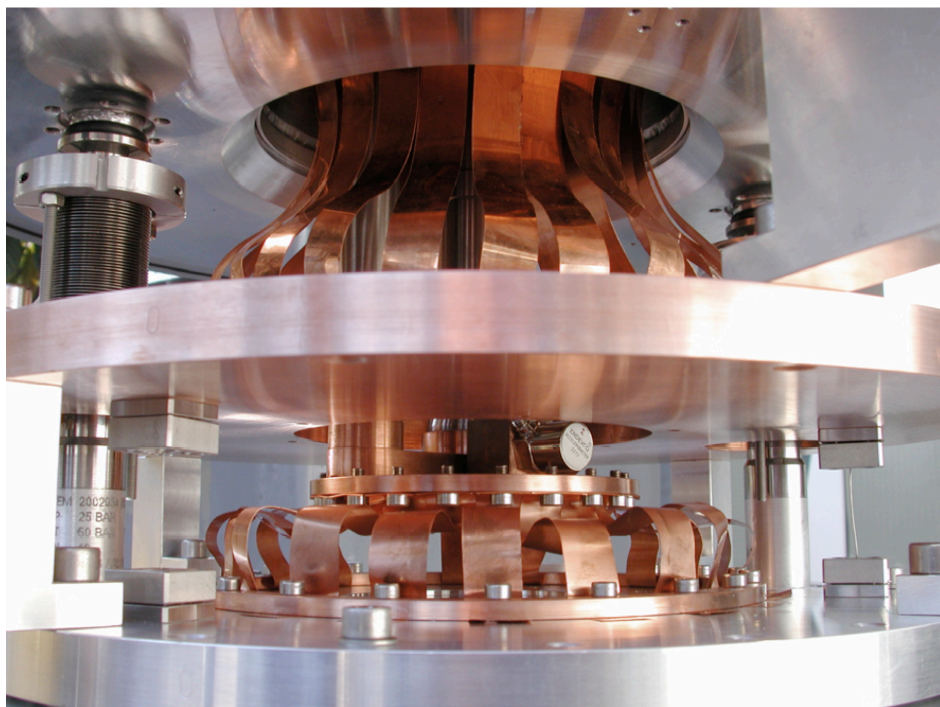
*Cryo-Compatible
Superattenuator
design*



Thermal Links I

Geometries

A corona of thin beams



Long Braids

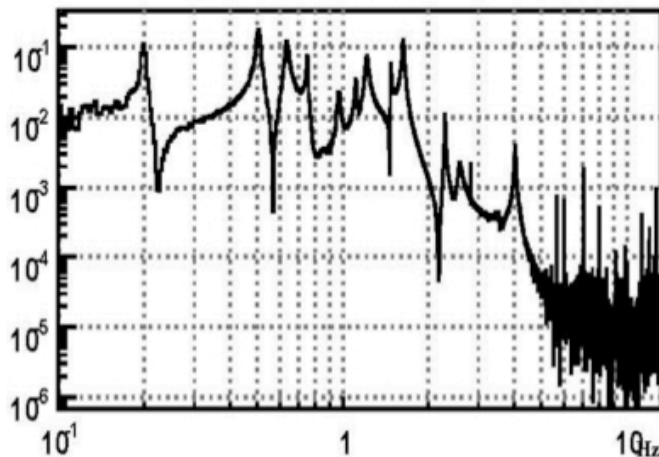


Thermal Links II:

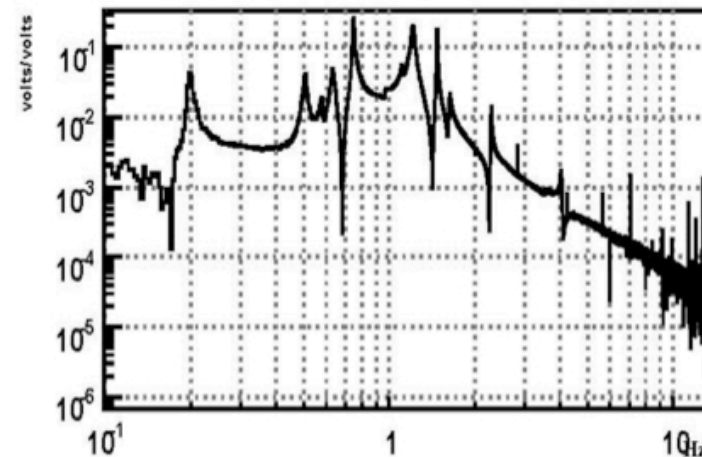
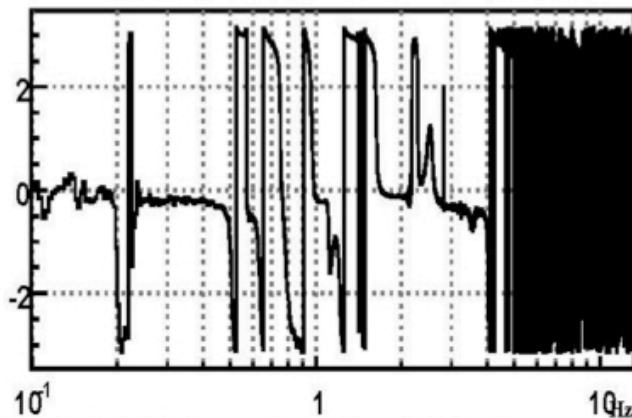
mechanical transfer function measurements at low temperature

P. Puppo talk next Thursday

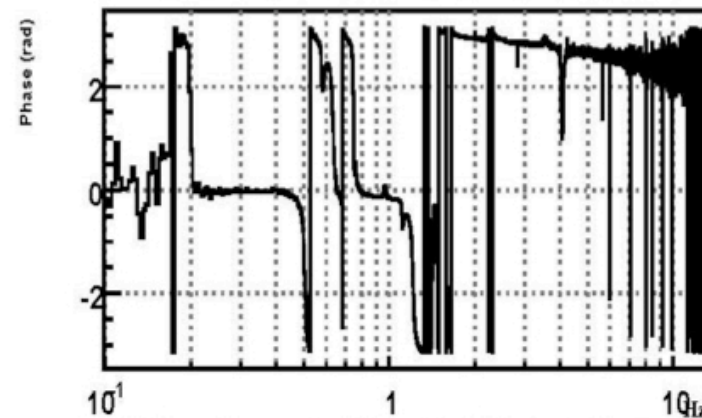
Evidence of a negligible influence of braids in the case of the torsion degrees of freedom



V1:Sc_CP_bf_RM_50Hz.over.V1:Sc_CP_noise_50Hz_TRFCT



V1:Sc_CP_bf_MA_50Hz.over.V1:Sc_CP_noise_50Hz_TRFCT

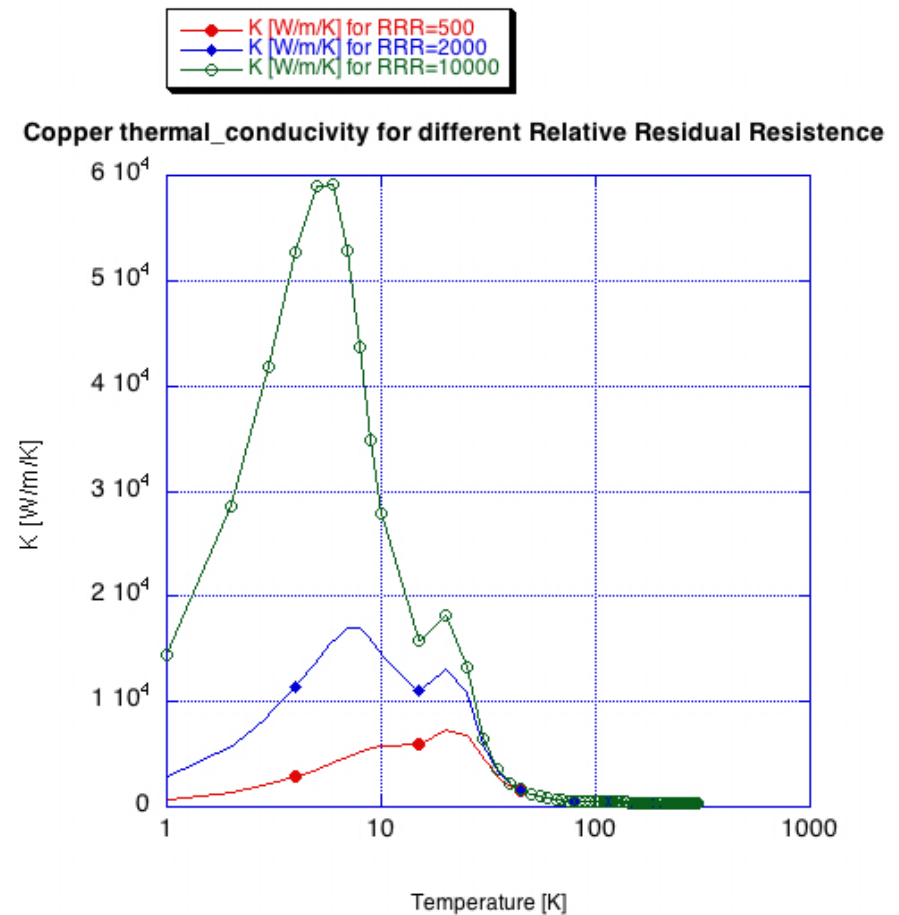
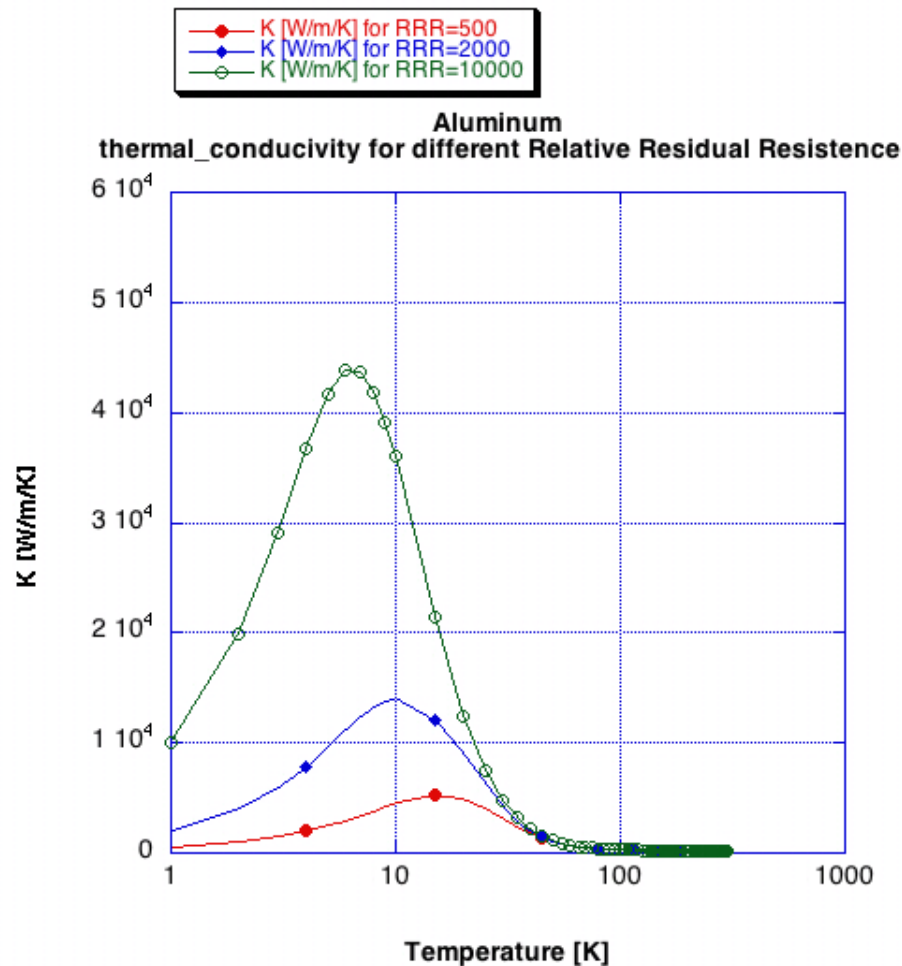


Thermal Links III

Pure Materials as aluminum and copper

$$RRR = \rho_{\text{room temperature}} / \rho_0$$

where ρ_0 resid. resist. at $T \sim 0$ K



Thermal Links IV

Solution for the stationary state

Use of a high purity material

$k \sim 2000 \text{ W/m/K}$ in the range 1-10 K

Thermal link length 20 m

Thermal difference at the link ends $\sim 1\text{K}$

LF Int. $\sim 200 \text{ mW}$ ~ 8 wires $r \sim 1 \text{ mm}$

Cooling strategy: two solution analyzed

- Cryo – coolers:

- the solution adopted in LCGT

- reduced manpower

- easy to use in an underground laboratory

- higher vibration and sound noise in the lab*

- lower refrigeration power*

- Cryo-fluids (Helium):

- technique adopted in the GW resonant detectors

- higher refrigeration power

- less noisy system

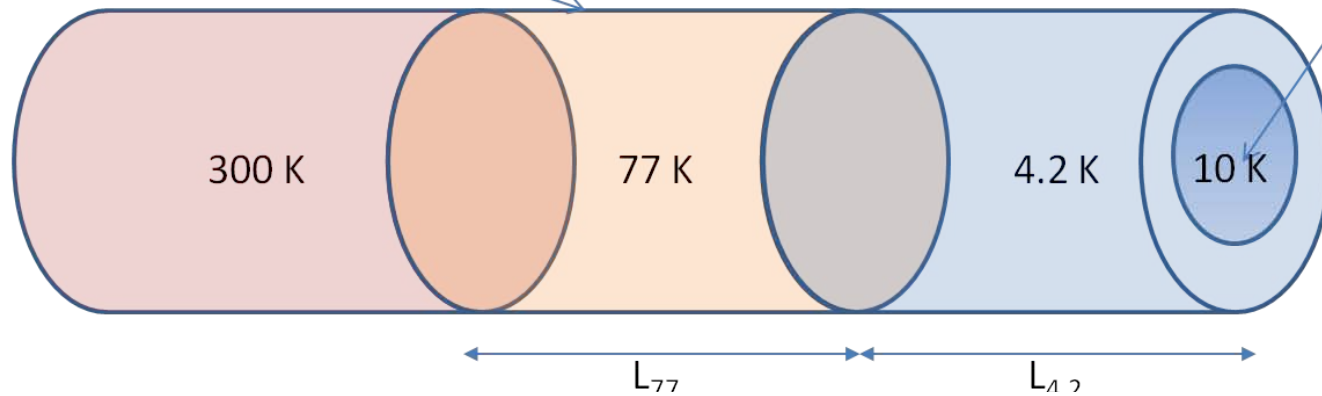
- higher manpower*

- safety issue in the underground laboratory*

Cryotraps

Inner Diameter 0.75 m

Mirror
Diameter 45 cm

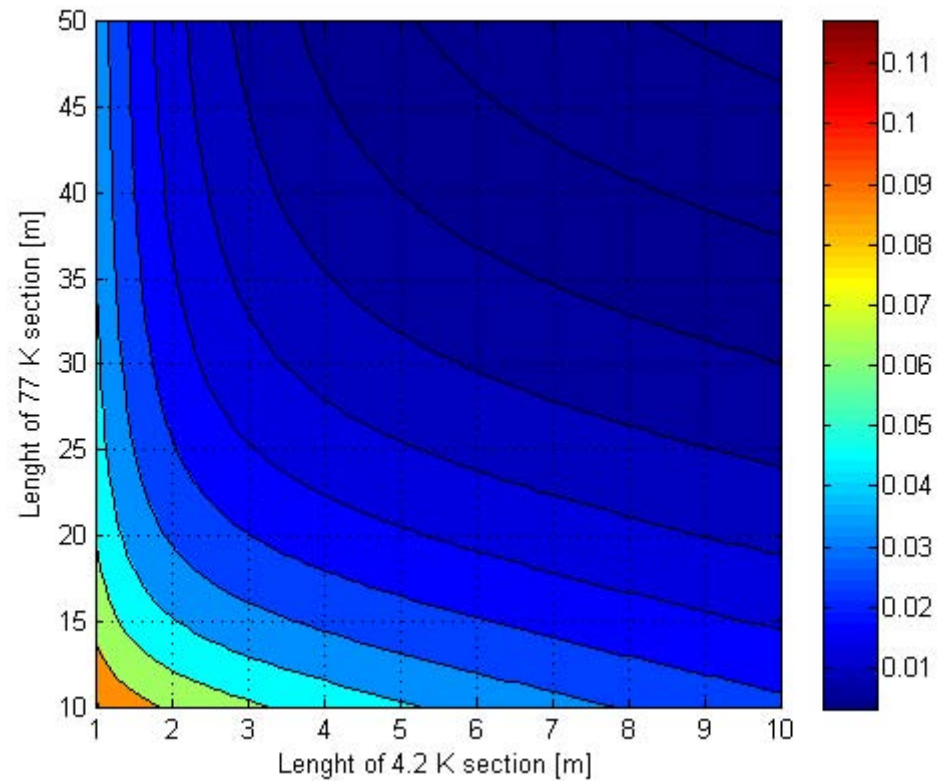


Heat sink
@ T=5 K

Silicon fibres
 $K \sim 10^3 \text{ W/(m K)}$

Laser power
(20 mW) +
radiation

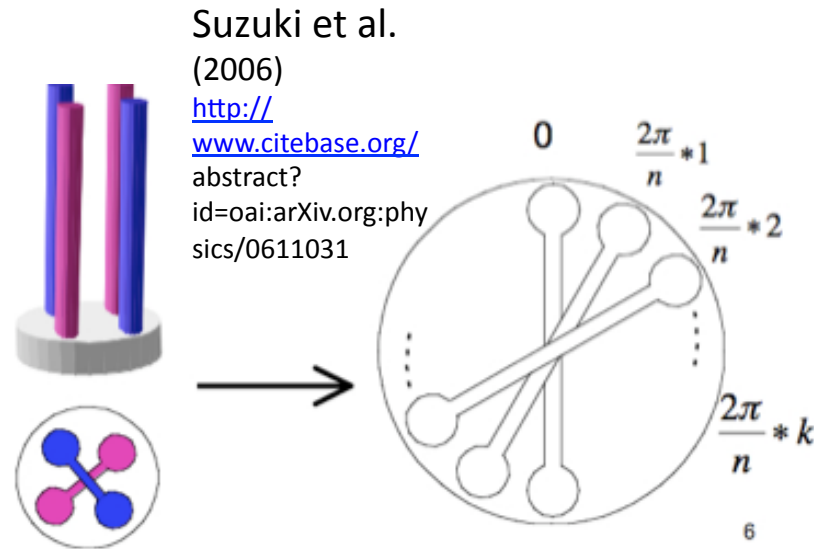
Heat exchange
by radiation with
ambient @ 4 K



Credits A. Chincarini, G. Gemme

R&D for reducing the cryo cooler vibration

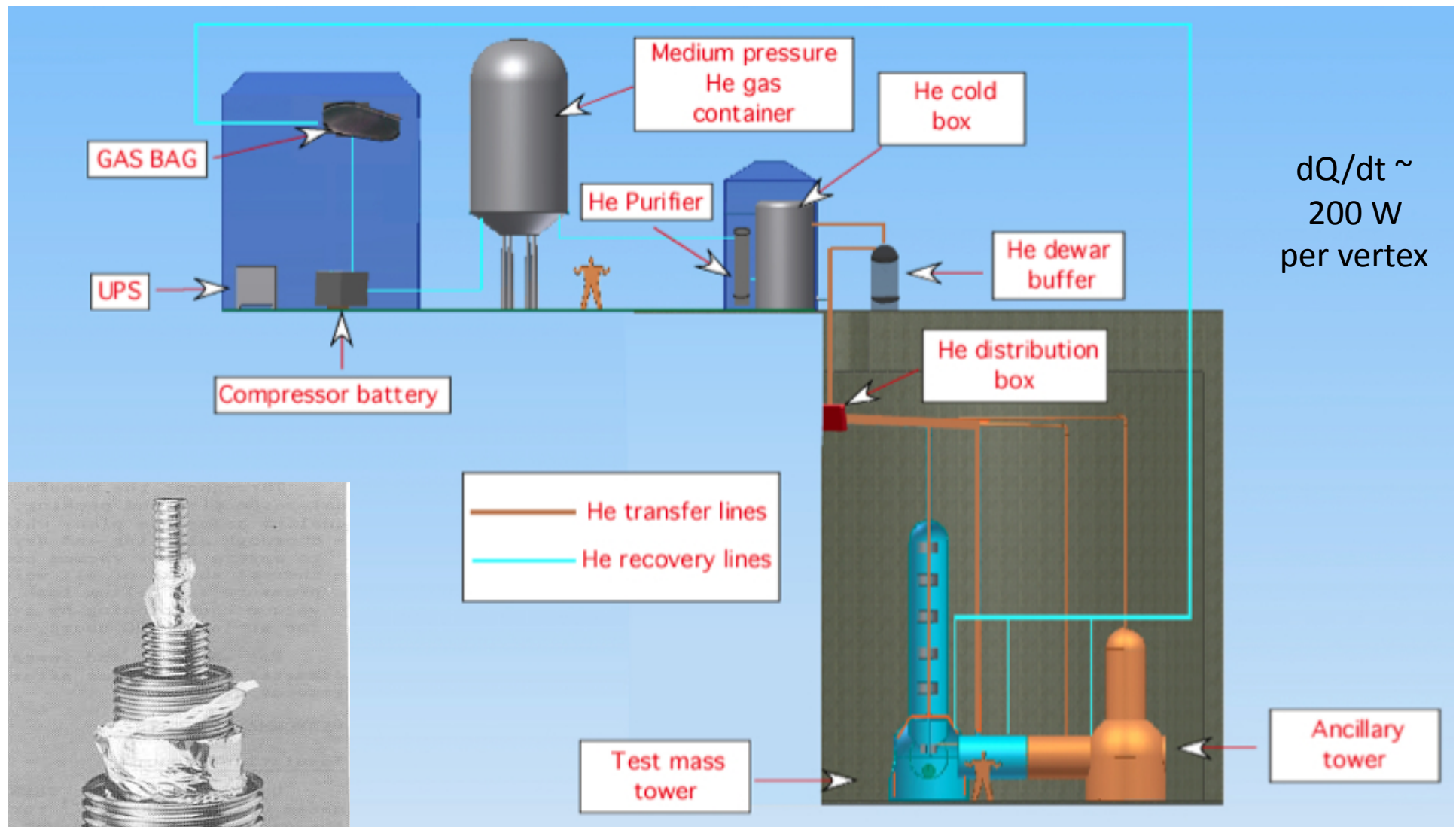
- Vibration noise of the refrigeration system ($\sim 0.01 - 0.03 \text{ mm}/(\text{Hz})^{1/2}$) kept under control by an active system.



- Improved attenuation is possible by adding to the first PT an other cryo-cooler, which operates @ 180° of phase



Cryofluid solution: a He plant in each vertex



- Low loss He transfer line ($\sim 30 \text{ mW/m}$): length $\sim 200 \text{ m}$
- Payload and ancillary tower cryostats
- He Cryotrap

Cryofluid: R&D on the boiling problem

(absent in the superfluid case)

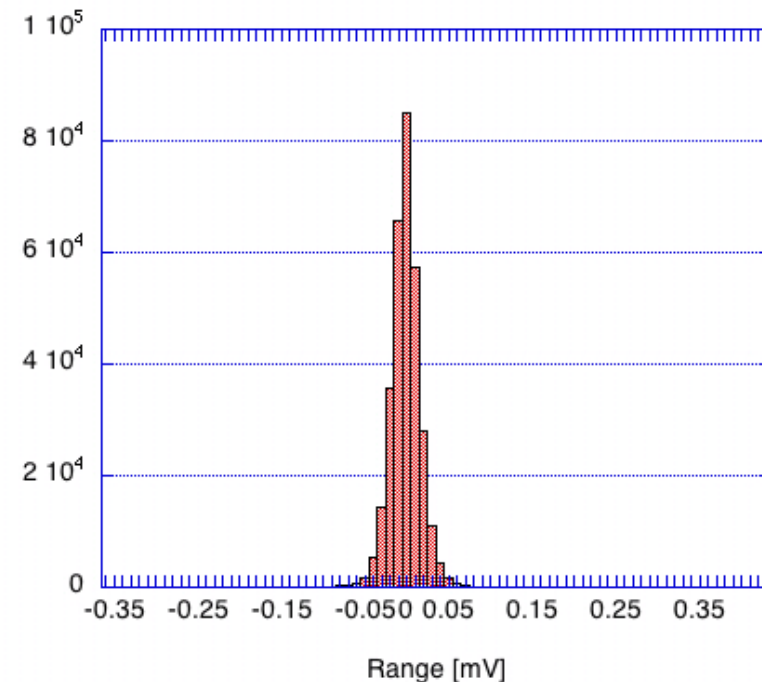
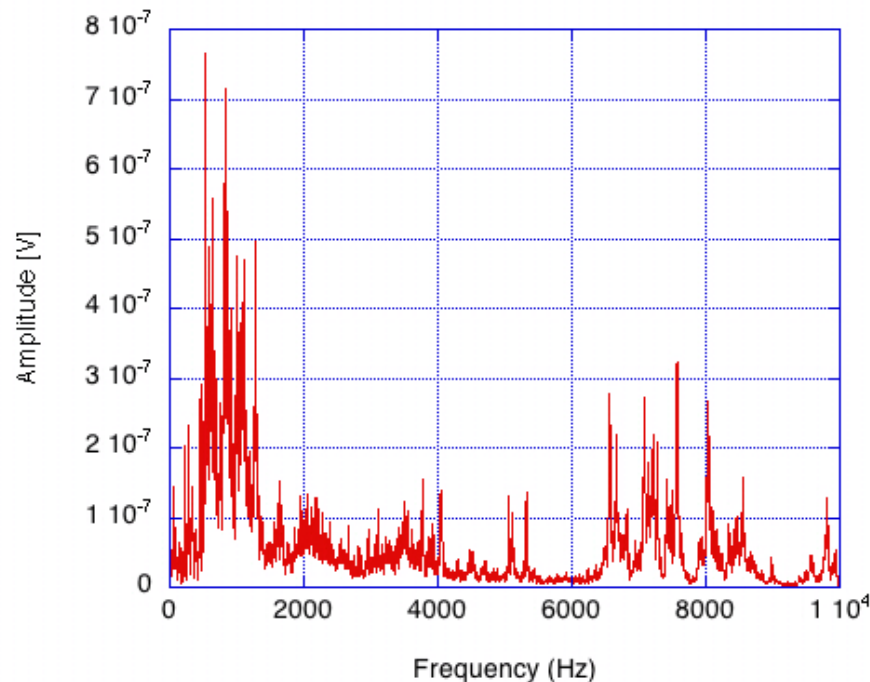
Displacement amplitude and frequency spectrum shape depend on the tank material and geometry: typical pressure fluctuation 20 dBa $\rightarrow 2 \cdot 10^{-4}$ Pa.

For example in the case of the GW resonant antenna Explorer $x_{\text{rms}} \sim 10^{-10}$ m @ 4K with an evaporation rate of a liquid Helium ~ 2 lt/h

Example of the noise characteristics of a boiling fluid in cylindrical container



A simplest approach is to use supercritical helium at $T > 5.19$ K (single phase status)



Conclusion

At the conclusion of the FP7- ET design study we are confident that ET is not any more a dream, it is a challenging project which requires R&D activity for

- ✓The material characterization,
- ✓The suspension improvement and its compatibility with the cryo temperatures
- ✓The development of new auxiliary sensors and actuators
- ✓The noiseless cooling process

This presentation is dedicated to the memory of our friend *Stefano Braccini*

It has been prepared thanks to the contributions of several physicists of the ET team

F Acernese, F Barone, S Braccini, A Chincarini, R De Rosa, P Falferi, J Franc, F Frasconi, G Gemme, A Gennai, M Lorenzini, E Majorana, R Nawrodt, G M Perciballi, R Poggiani, P Puppo, D Rabeling, S Reid, S Rowan, M Tonelli, J F J van den Brand.....and many others