

# DEVELOPMENT AND EXPERIMENTAL CHARACTERIZATION OF PROTOTYPES FOR LOW MATERIAL BUDGET SUPPORT STRUCTURE AND COOLING OF SILICON PIXEL DETECTORS, BASED ON MICROCHANNEL TECHNOLOGY



F.Bosi\*, M.Massa\*\* – INFN Pisa

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\*filippo.bosi@pi.infn.it - \*\*maurizio.massa@pi.infn.it

## Introduction

Pixel detectors at future colliders will need to match very stringent requirements on position resolution. The support structure and cooling add important contributions to the total material in the active area.

Advantages of the MICROCHANNELS technology:

- due to the high surface/volume ratio, heat exchange through forced convection of a liquid coolant is taking place efficiently, obtaining high thermal conductivities without affecting the stiffness of the structure
- the contiguity between the fluid and the circuit dissipating power reduces thermal resistances
- uniform distribution of the passive material and uniform temperature of the surface covered by the sensors can be obtained.

Several prototypes with different geometries of micro-machined channels have been realized in ceramics (AlN) and composite materials (CFRP). FEA simulations have been validated by the experimental tests performed in the thermofluidodynamics test-bench, recently assembled at the INFN-Pisa laboratory.

## The key concept

In a thermal convective exchange the film coefficient is:

$Nu = \text{Nusselt number}$

$K = \text{Conductive heat transfer coefficient of the liquid}$

$D_h = \text{Hydraulic Diameter of the cooling channel}$

$$h = \frac{Nu \cdot k}{D_h}$$

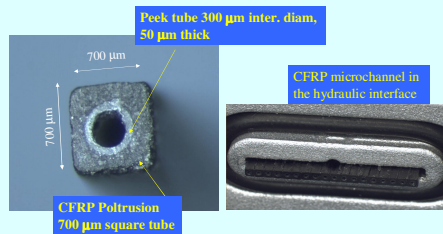
To maximize the h value it is important to **minimize the hydraulic diameter**: this remark points us to the **microchannel technology**.

Minimize  $D_h \rightarrow$  high pressure drop (needed a compromise between pressure drops and film coefficient value).

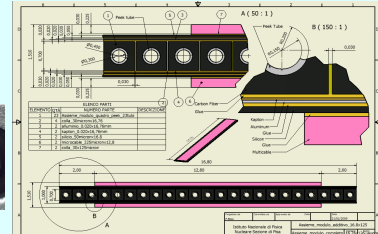
## The specifications for the pixel support (layer0 SuperB)

- evacuate the heat dissipated by the electronics (specific power  $\sim 2 \text{ W/cm}^2$ ), keeping the temperature of the sensors below  $50^\circ\text{C}$
- material budget: the pixel support structure (w/o cables and sensors) has to remain below  $0.30\% X_0$ .

### Microchannel CFRP single unit

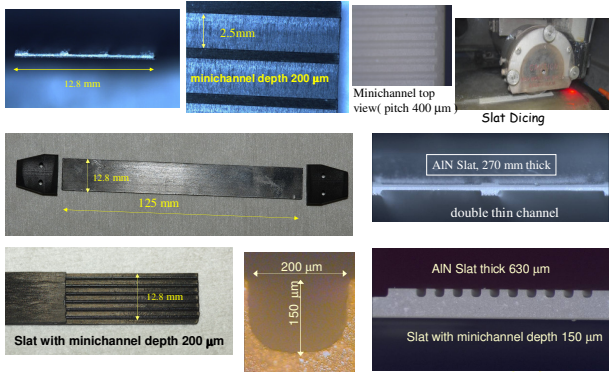


### Microchannel module assembly (additive method)



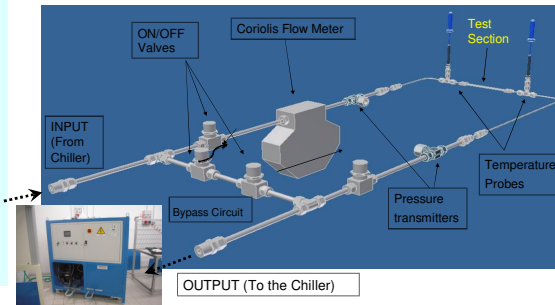
## Prototype production

### CFRP Subtractive Method and AlN Micromachining

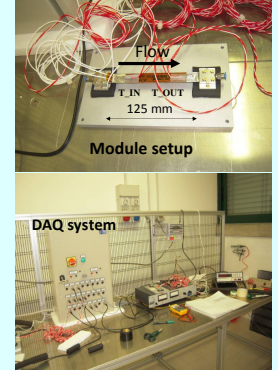


## Test-bench hydraulic circuit

Liquid coolant: water +  $\text{C}_2\text{H}_6\text{O}_2$  - 50% @  $10^\circ\text{C}$   
Specific Power density:  $2 \text{ W/cm}^2$  (using a kapton heater)

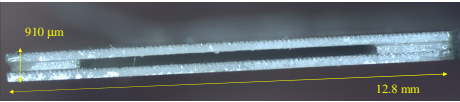


## TFD Lab @INFN-Pisa

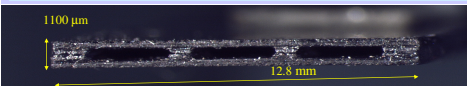


## Test results

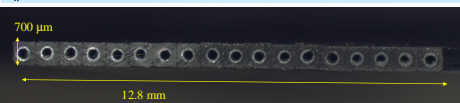
Single Channel AlN additive method: Thickness =  $0.65\% X_0$   
 $D_h = 0.48 \text{ mm}$



Triple channel CFRP subtractive method: Thickness =  $0.40\% X_0$   
 $D_h = 0.84 \text{ mm}$



Microchannel CFRP additive method: Thickness =  $0.28\% X_0$   
 $D_h = 0.3 \text{ mm}$

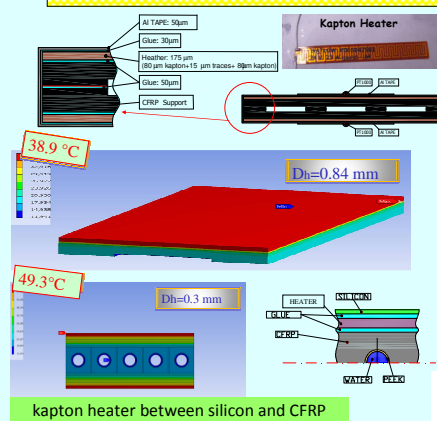


AlN single channel measurement	
$T_{\text{FLUID}}$	$9.5^\circ\text{C}$
Power Density $\delta_p$	$2 \text{ W/cm}^2$
Flow Rate	$0.45 \text{ Kg/min}$
$T_{\text{IN\_AVERAGE}}$	$32.4^\circ\text{C}$
$T_{\text{OUT\_AVERAGE}}$	$42.8^\circ\text{C}$
$P_{\text{IN}}$	$1.9 \text{ bar}$

CFRP triple channel measurement	
$T_{\text{FLUID}}$	$9.5^\circ\text{C}$
$\delta_p$	$2 \text{ W/cm}^2$
Flow Rate/ch	$0.24 \text{ kg/min}$
$T_{\text{IN\_AVERAGE}}$	$41.1^\circ\text{C}$
$T_{\text{OUT\_AVERAGE}}$	$43^\circ\text{C}$
$P_{\text{IN}}$	$2.6 \text{ bar}$

CFRP Microchannel measurement	
$T_{\text{FLUID}}$	$9.5^\circ\text{C}$
$\delta_p$	$2 \text{ W/cm}^2$
Total Flow Rate	$0.26 \text{ Kg/min}$
$T_{\text{IN\_AVERAGE}}$	$45.1^\circ\text{C}$
$T_{\text{OUT\_AVERAGE}}$	$47.2^\circ\text{C}$
$P_{\text{IN}}$	$3.6 \text{ bar}$

## FEA studies



### Triple Channel FEA Simulation

(1/6 of the total structure)

Thermal conductivities:  
CFRP support:  
 $K_x = K_y = 75 \text{ W/mK}$ ;  $K_z = 3 \text{ W/mK}$   
Aluminum Tape:  $K = 180 \text{ W/mK}$   
Glue:  $K = 0.15 \text{ W/mK}$   
Heater:  $K = 0.15 \text{ W/mK}$   
Boundary conditions:  
Power density:  $2 \text{ W/cm}^2$   
 $h = 15000 \text{ W/m}^2\text{K}$ , turbulent flow  
(total flow rate =  $1 \text{ kg/min}$ ,  
Fluid  $\text{H}_2\text{O} + \text{C}_2\text{H}_6\text{O}_2$  - 50%)  
 $T_{\text{fluid}} = 9.5^\circ\text{C}$   
Results:  $T_{\text{max}} = 38.9^\circ\text{C}$

Substantial agreement between the test results and FEA studies.

## Conclusions & perspectives

- The microchannel CFRP prototype best matches the requirements on  $X_0$  with efficient heat evacuation.
- Room for further optimization and reduction in material
- Future plan: evaporative cooling in microchannels to reach lower temperatures.

