# Square cavity alignment

- Each mirror has a 3-axial PZT movement.
- Globally there are 12 degrees of freedom, which can be expressed as linear combination of:

 $X_{A},\,X_{B},\,X_{C},\,X_{D},\,Y_{A},\,Y_{B}$  ,  $Y_{C}$  ,  $Y_{D},\,Z_{A},\,Z_{B},\,Z_{C},\,Z_{D}$  .

• The 3 rigid translations can be expressed by linear combinations of the motion of the 4 mirrors:

 $\begin{array}{l} ({\rm X}_{\rm A} + {\rm X}_{\rm B}) + ({\rm X}_{\rm C} + {\rm X}_{\rm D}) \\ ({\rm Y}_{\rm A} + {\rm Y}_{\rm B}) + ({\rm Y}_{\rm C} + {\rm Y}_{\rm D}) \\ ({\rm Z}_{\rm A} + {\rm Z}_{\rm B}) + ({\rm Z}_{\rm C} + {\rm Z}_{\rm D}) \end{array}$ 

- The 3 rigid rotations can be expressed as:  $(X_B - X_D) + (Y_A - Y_C)$   $(Y_A - Y_C) + (Z_B - Z_D)$  $(Z_A - Z_C) + (X_B - X_D)$
- 2 degrees of freedom are freezed assuming that the length of the two Fabry-Pérot (FP1 and FP2) between the opposite mirrors are measured at  $10^{-10}$  $X_A - X_C = Y_B - Y_D = d$



- the residual 4 degrees of freedom :
- $(X_A + X_C) (X_B + X_D)$ : FP1 orthogonal to FP2; the centre of FP2 is translated by  $\delta_1$ along FP1
- $(Y_B + Y_D) (Y_A + Y_C)$ : FP1 orthogonal to FP2; the centre of FP1 is translated by  $\delta_2$ along FP2
- $(Y_A Y_C) (X_B X_D)$ : FP1 axis is rotated of  $\pi/4 + \epsilon$  with respect to FP2 axis
- $(Z_A + Z_C) (Z_B + Z_D)$ : FP1 orthogonal and centred to FP2, but with a vertical displacement  $\zeta$ .

# Square cavity alignment

- For all the kind of deformations, keeping fixed the diagonals, perimeter and area values approach quadratically the case of perfect square. The value of the perimeter when optimized must be 2d V2.
- Around the best geometry, the perimeter is a quadratic function of the deviations. The motion, combined as indicated above, are mutually orthogonal.
- It is then possible to optimize one degree of freedom at time. It can be achieved by a parabolic fit or, as an error signal by modulating the PZT. In principle, it should be possible to modulate each degree of freedom at different frequencies, and to detect on-line the error signals.



- If we neglect the 6 rigid body degrees:
  6 PZT are sufficient: 3 in A, 1 in B, C, D(along the diagonal)
- Diagonal length:  $X_A X_C$  and  $Y_B Y_D$
- Planarity control: Z<sub>A</sub>
- Angle between FP1 and FP2 axis: Y<sub>A</sub>
- Crossing point:  $(X_A + X_C)$  and  $(Y_B + Y_D)$

- Each mirror has a 3-axial PZT movement.
- Globally there are 18 degrees of freedom, which can be expressed as linear combination of:

 $X_A, X_B, X_C, X_D, X_E, X_F;$   $Y_A, Y_B, Y_C, Y_D, Y_E, Y_F;$  $Z_A, Z_B, Z_C, Z_D, Z_E, Z_F.$ 

$$\boldsymbol{\Omega}_{obs}^2 = \boldsymbol{\Omega}^2 \cdot \sum_{i=1}^3 \left( \hat{\boldsymbol{\Omega}} \cdot \hat{\boldsymbol{n}}_i \right)^2$$



- Diagonal deplacements (fixed length; no rotation) 3x3 degrees of freedom:
- Rotations of one diagonal with respect to the other two (fixed length; no deplacements) 3x2 degrees of freedom
- Diagonal stratching 3 degrees of freedom

- Diagonal deplacements (fixed length;
  no rotation) 3x3 degrees of freedom
  - 3: Rigid body translations along *x*, *y*, *z*
- Do not affect the orientation of the rings
  - The effect on the scale factor has been already calculated to be quadratic

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$$\boldsymbol{\Omega}_{obs}^2 = \boldsymbol{\Omega}^2 \cdot \sum_{i=1}^3 \left( \hat{\boldsymbol{\Omega}} \cdot \hat{\boldsymbol{n}}_i \right)^2$$



- Diagonal deplacements (fixed length; no rotation) 3x3 degrees of freedom:
- Diagonal stratching 3 degrees of freedom
- Rotations of one diagonal with respect to the other two (fixed length; no deplacements) 3x2 degrees of freedom

- Diagonal stratching
  - Fixed by servos!

- Each mirror has a 3-axial PZT movement.
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- Diagonal deplacements (fixed length; no rotation) 3x3 degrees of freedom:
- Diagonal stratching 3 degrees of freedom
- Rotations of one diagonal with respect to the other two (fixed length; no deplacements) 3x2 degrees of freedom



- Rotations (6):
  - around x-axis:  $R_x^z$ ;  $R_y^z$

 $R_y^x + R_z^x$  $R_x^y + R_z^y$ 

 $R_{x}^{z} + R_{y}^{z}$ 

 $R_y^x - R_z^x$ 

 $R_{x}^{y} - R_{z}^{y}$ 

- around y-axis:  $R_{x}^{y}; R_{z}^{y}$
- around z-axis:  $R_v^x$ ;  $R_y^x$
- Rigid rotations :

Angular deformation

$$\begin{split} \Omega_{obs}^2 &= \sum_{i=1}^3 \left( \vec{\Omega} \cdot \hat{n}'_i \right)^2 = \sum_{i=1}^3 \left( \vec{\Omega} \cdot \left( \hat{n}_i + \lambda_{ij} \hat{n}_j \right) \right)^2 = \sum_{i=1}^3 \left( \vec{\Omega} \cdot \hat{n}_i + \lambda_{ij} \vec{\Omega} \cdot \hat{n}_j \right)^2 \\ &= \sum_{i=1}^3 \left( \vec{\Omega} \cdot \hat{n}_i \right)^2 + \sum_{i=1}^3 \left( \lambda_{ij} \vec{\Omega} \cdot \hat{n}_j \right)^2 + 2 \sum_{i=1}^3 \left( \vec{\Omega} \cdot \hat{n}_i \right) \left( \lambda_{ij} \vec{\Omega} \cdot \hat{n}_j \right) \\ &= \Omega^2 + 2 \lambda_{ij} \Omega_i \Omega_j \end{split}$$

• Angular motion 
$$R_y^x - R_z^x$$
  
deforms the geometry of the ring *yz*  
and rotates the direction of the normal  
to the other two rings.

$$\hat{n}'_{x} = \hat{n}_{x}$$
  
 $\hat{n}'_{y} = \hat{n}_{y} + \lambda \hat{n}_{z}$   
 $\hat{n}'_{z} = \hat{n}_{z} + \lambda \hat{n}_{y}$ 

$$\Omega_{obs}^2 = \Omega^2 + 4\lambda \Omega_y \Omega_z$$

# ELUCUBRAZIONI FANTASIA

#### Cavità ottiche triangolari

*Come fissare la geometria* 

