Thermal Insulation in **Cryogenic Systems**

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Types of insulation used in cryogenic equipment

 \triangleright Expanded closed-cell foams,

- \triangleright Gas-filled powders and fibrous materials,
- \triangleright Aerogel insulation,
- \triangleright Evacuated powders and fibrous materials,
- \triangleright Opacified powders,
- \triangleright Microsphere insulation,
- \triangleright Multi layers Insulations MLIs

Expanded closed-cell foams - I

A foam with a cellular structure produced by evolving gas during its formation

- Heat transfer contributions are due to the convection and radiation within the cells and the conduction through the solid structure.
- Thermal conductivity depends on the density and by the type of gas and the mean temperature used in the manufacturing process.

 $k_t = 0.4(1 - \Phi)k_s + k_g + 4F_e d\sigma T_m^3$

Φ porosity of the foam (volume of void per unit total volume)

- *ks* thermal conductivity of the solid foam material
- k_{s} thermal conductivity of the gas within the cavities
- *d* thickness of the foam

 T_m mean insulation absolute temperature = $_{12}(T_H + T_C)$

 σ = 0.1714 × 10-8 Btu/h-ft₂-°R₄ = 5.669 × 10-8 W/m₂-K₄ (Stefan–Boltzmann constant)

Different Foam types made of

- \triangleright polyurethane
- \triangleright polystyrene
- \triangleright glass foam

 $rac{1}{F_e} = \frac{d}{d_c} \left(\frac{1+r-t}{1-r+t} \right) + 1$

Main use: protect and insulate underground pipes

Fe emissivity factor reflectivity of the foam material transmissivity of the foam material *dc* size of the foam cells

$$
\rho_m = \rho_s (1 - \Phi) + \rho_g \Phi
$$

 ρ_g = gas density $\rho_{\sf s}$ =solid density

Expanded closed-cell foams - II

Insulation of liquid natural gas pipes (higher tolerance of thermal input) Advantage: low cost solution

Disadvantage: large thermal contraction in the case rigid foam insulations.

Thermal Conductivity of Selected Foams and Gas-Filled Insulations

Example: Thermal exp. coeff. 7.2×10^{-5} K⁻¹ glass foam 1.5×10^{-5} K⁻¹ carb. steel $\Delta T = 100$ K, L = 6 m Δ L = 3.6 cm

A cover is needed to prevent diffusion of water vapor and air into the foam. It will reduced also the replacement of the foaming agent with hydrogen and helium gases, which deteriorate the the thermal insulation.

Expanded closed-cell foams - III

Typical values for polyurethane foam

 $r = 0.034$ \rightarrow reflectivity $t = 0.493 \rightarrow$ transmittivity $d_c = 0.25 - 0.50$ mm \rightarrow cell dimensions $\Phi \sim 1 \rightarrow$ porosity

Typical budget of thermal input in the case of polyurethane foam

Conduction via solid \sim 20 % Conduction via gas \sim 70 % Radiation $~\sim$ 10 %

The problem of differential themal contraction: an example: Thermal exp. coeff. 7.2×10^{-5} K⁻¹ glass foam 1.5×10^{-5} K⁻¹ carb. steel $\Delta T = 100$ K, L = 6 m Δ L = 3.6 cm

A cover is needed to prevent diffusion of water vapor and air into the foam. It will reduced also the replacement of the foaming agent with hydrogen and helium gases, which deteriorate the the thermal insulation.

Gas-filled powders and fibrous materials - I

perlite

- \triangleright fiberglass,
- \triangleright perlite (a silica powder),
- Ø silica aerogel (Santocel, Cab-O-Sil),
- \triangleright rock wool
- \triangleright vermiculite.

- d_s is the size of the particles or fibers
- Φ is the porosity
- k_s = thermal conductivity of solid
- k_g = thermal conductivity of gas

Thermal conductivity: main differenece to the respect of the expanded closed celles foama Suppression of gaseous convection due to the small size of the voids within the material.

Gas-filled powders and fibrous materials - II

If k_s >> k_g for example Silica glass $k_s = 0.95 \text{ W/m-K}$, N_2 $k_g = 0.017 \text{ W/m-K}$

The case of fine powder.

 k_g depends on the free path of the gas molecules which is reduced if we use fine powder.

However, this decrease in thermal conductivity is also influenced by the increase in overall contact resistance because more contacts exist in series for small powder particles than for larger particles.

Aerogel insulation - I

AerogeL \rightarrow synthetic material produced from liquid gels by replacing the liquid with a gas through a drying process.

Density $1/10$ of H₂O \rightarrow 80 – 130 kg/m³

Structure: chains of spherical particles (\sim 2-5 mm diameter) fused in cluster and pores of 0.1 mm diameter

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Typical trapped gas : air
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Thermal conductivuty at 200 K \rightarrow 18 mW/m-K

Transparent to infrared radiation \rightarrow radiant heat transfer is the most significant contribution

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k_t = k_0 + \frac{\kappa_g}{1 + 1.6(\lambda/d_s)}
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 k_0 thermal conductivity at low gas pressure $(p \rightarrow 0)$ k_{g} ithermal conductivity of the gas at themean temperature λ mean free path of the gas at the mean temperature, d_1 mean diameter of the micropores

 k_0 *is the sum of the solid conduction and the radiation contributions.*

The case of silica aerogel $k_0 = 1.2$ mW/m-K $d_1 = 9 \mu m$

Aerogel insulation - II

Advantages:

Low density (lightweight) and low thermal conductivity. Flexible aerogel blankets can withstand a load pressure of about 35 kPa for a 10% deformation.

Disadvantages:

It must be protected from moisture in the atmosphere: it will form a gel after prolonged contact with moist air. Therefore, a vapor barrier must be used around the insulation.

Evacuated powders and fibrous materials - I

Obvious method of reducing the heat transfer rate through these insulations is to evacuate the gas from the insulation.

Typical residual pressure 100 mBar range where the thermal conductivity of the gas is practically independent of pressure

Typical gas residual below 10 mbar, thermal conductivity is almost directly proportional to the gas pressure (free path of N₂ at 300 K \sim 5 µm). With gaps between particles of 15 μ m, the heat transfer via molecular conductions decreases with the pressure

At \sim 10⁻³ mbar the only contributions to the heat transfer are due to radiation and solid conduction

Evacuated powders and fibrous materials - II Thermal Conductivity of Selected Evacuated Insulations, Excluding MLIs

Note: The boundary temperatures are 300 K (80°F) and 77 K (-321 °F). The residual gas pressure is below 10^{-3} torr (130 mPa).

Variation of thermal conductivity vs residual nitrogen pressure (powder 30–80 mesh perlite). Boundary temperatures 300 - 78 K

Opacified powders

A technique for reducing the radiant heat transfer through the powder insulation is to add opaque, highly reflecting metal flakes

Example of Thermal conductivity Gain in the case of both perlite and copper flakes loaded by 40 % of opacifier (mass fraction)

1.9 $mW/m-K$ \rightarrow 1.2 $mW/m-K$

Disadvantage: Degradation of ethe performance in the insulation zones where the metallic flakes are concentrated

Microsphere insulation

The microsphere insulation:

- \triangleright packed, hollow glass spheres, size 15 150 µm diameter,
- \triangleright coated on the outer surface with a highly reflective film.

Heat flux (under a vacuum >10−4 mbar) limited by

 \triangleright the resistance to conduction between spheres

 \triangleright radiation attenuation due to the metal coating and scattering between spheres.

$$
k_t = k_s + \left(\frac{4}{3b}\right) \sigma (T_H^2 + T_C^2)(T_H + T_C)
$$

 k_s \rightarrow thermal conductivity associated with conduction through the solid material $b \rightarrow \infty$ extinction (absorption) coefficient for radiant heat transfer through the insulation $\sigma = \rightarrow$ Stefan–Boltzmann constant (5.669 × 10⁻⁸ W/m²-K⁴) T_H and T_C hot and cold boundary absolute temperatures respectively

Examples:

aluminized microspheres with a 20–80 nm coating thickness, $k_s = 1.84 \times 10^{-4}$ *W/m-K, b = 13750 m⁻¹ fully aluminized microspheres* $k_s = 5.17 \times 10^{-4}$ *W/m-K, b = 34000 m⁻¹*

MLIs – Multi Layers Insulation - I

Alternating layers of highly reflecting foil, such as aluminum or copper foil, and a low-conductivity spacer, such as fiberglass paper, Dacron fabric, or silk net.

Foil material are fragile and develops pin holes when crinkled, \rightarrow replaced by aluminized Mylar, Mylar plastic thickness \sim 6 - 10 μ m, Aluminum layer 40 - 60 nm Operation in Vacuum $>8 \times 10^{-5}$ mbar (10 mPa)

Variation of the mean apparent thermal conductivity with residual gas pressure for a typical MLI.

MLIs – Multi Layers Insulation - II

Heat flux reductions:

- \triangleright Solid conduction minimized by the low-conductivity spacers (contacts just in few points).
- Ø Gaseous conduction killed by reducing the residual gas pressure below 10−5 mbar .
- \triangleright Radiation heat transfer is minimized by using many layers of highly reflective metal foil

$$
k_t = \frac{1}{\left(N/\Delta x\right)} \left[h_c + \left(\frac{e}{2-e}\right) \sigma \left(T_H^2 + T_C^2\right) \left(T_H + T_C\right)\right]
$$

- $h_c \rightarrow$ solid conductance of the spacer material
- σ \rightarrow Stefan–Boltzmann constant (5.669 × 10⁻⁸ W/m²-K⁴)
- $e \rightarrow$ the emissivity of the radiation shields

 T_H and T_C \rightarrow boundary temperatures of the insulation in Kelvin

*N/*Δx \rightarrow layer density (1 layer = 1 sheet foil + 1 sheet of spacer)

The case of MLI made of double-aluminized Mylar shields and a double layer of silk net as separator. Empirical formula for h_c

$$
h_c=C_s(N/\Delta x)^{2.56}(T_H+T_C)
$$

MLIs – Multi Layers Insulation - III

MLI Optimum insulation :

- \triangleright Solid conduction minimized by the low-conductivity spacers (contacts just in few points).
- Ø Gaseous conduction killed by reducing the residual gas pressure below 10−5 mbar .
- \triangleright Radiation heat transfer is minimized by using more layers of highly reflective metal foil

 \triangleright Optimization of the Layer Density $\rho_m = (S_s + \rho_r t_r)(N/\Delta x)$

- S_s \longrightarrow mass of spacer material per unit area
- $\rho_r \rightarrow$ density of the shield material
- t_r \rightarrow thickness of the radiation shields
- $N/\Delta x$ \rightarrow layer density (1 layer = 1 sheet foil + 1 sheet of spacer)

Thermal Conductivity of Selected MLIs

Note: The boundary temperatures are 300 K (80°F) and 77 K (-321°F) with residual gas pressures of 1.3 mPa (10⁻⁵).

MLIs – Multi Layers Insulation - IV

Optimisation of the layer density:

If $N/\Delta x$ increase, radiant thermal input decrease If $N/\Delta x$ increase (higher compressed pack of MLI) solid conduction increase

$MLIs - Multi \; Layers \; Insulation - V$

Thermal conductivity of MLI in the direction parallel to the shields

$$
k_{||} = \left(\frac{N}{\Delta x}\right) \left\{ k_r t_r + \left(\frac{2t_p}{N/\Delta x}\right) \left[2\ln\left(\frac{L_p}{2t_p}\right) - 1\right] \sigma \left(T_H^2 + T_C^2\right) (T_H + T_C) \right\}
$$

- $k_r \rightarrow$ thermal conductivity of the reflective radiation shield
- $t_r \rightarrow$ shield thickness
- $L_p \rightarrow$ length of the shield parallel to the shield layers

 t_p \rightarrow space between the shields, given by \rightarrow \rightarrow \rightarrow \rightarrow $t_p = \frac{1 - (N/\Delta x)t_r}{(N/\Delta x)}$

Thermal conductivity of MLI in the direction parallel to the shields may be as much as three orders of magnitude larger than the thermal conductivity normal to the radiation shields

Insula6on performance degraded if the radia6on shields edges contact another surface!!!!

MLIs – Multi Layers Insulation - VI

Residual pressure among the various layers

- \triangleright Effective evacuation of the residual gas from within the insulation blanket is difficult: Mylar is and hydrophilic material!!
- \triangleright Pressure within the insulation may be 2 orders of magnitude higher than the pressure outside the insulation
- \triangleright Outgassing of the shields can also introduce significant quantities of gas within the insulation

Improvements:

- \triangleright Small holes are present in the foil layers to improve the gas removal in the blanket (holes can be made by means of an ordinary sewing needle)
- \triangleright MLI using carbon-filled glass-fiber paper for the spacer material. Carbon acts as "getter" to adsorb the outgassing load at cryo temperatures. Results: Effective Thermal conductivity down to 14 μW/m-K

MLIs – Multi Layers Insulation \rightarrow Conclusion

Complex system involving three heat transfer processes

 $Q_{M1} = Q_{rad} + Q_{sol} + Q_{res}$

– With n reflective layers of equal emissivity,

Q rad ~ 1/(N+1)

– Due to parasitic contacts between layers, Q_{sol} increases with layer density

- *Q res* due to residual gas trapped between layers, scales as *1/N* in molecular regime
- Non-linear behaviour requires layer-to-layer modelling

In practice

- Typical data available from (abundant) literature
- Measure performance on test samples

My personal experience: *Minimal cover 30 layers at least*

