

Radiative modeling and cryogenic design for LIGO Voyager prototype

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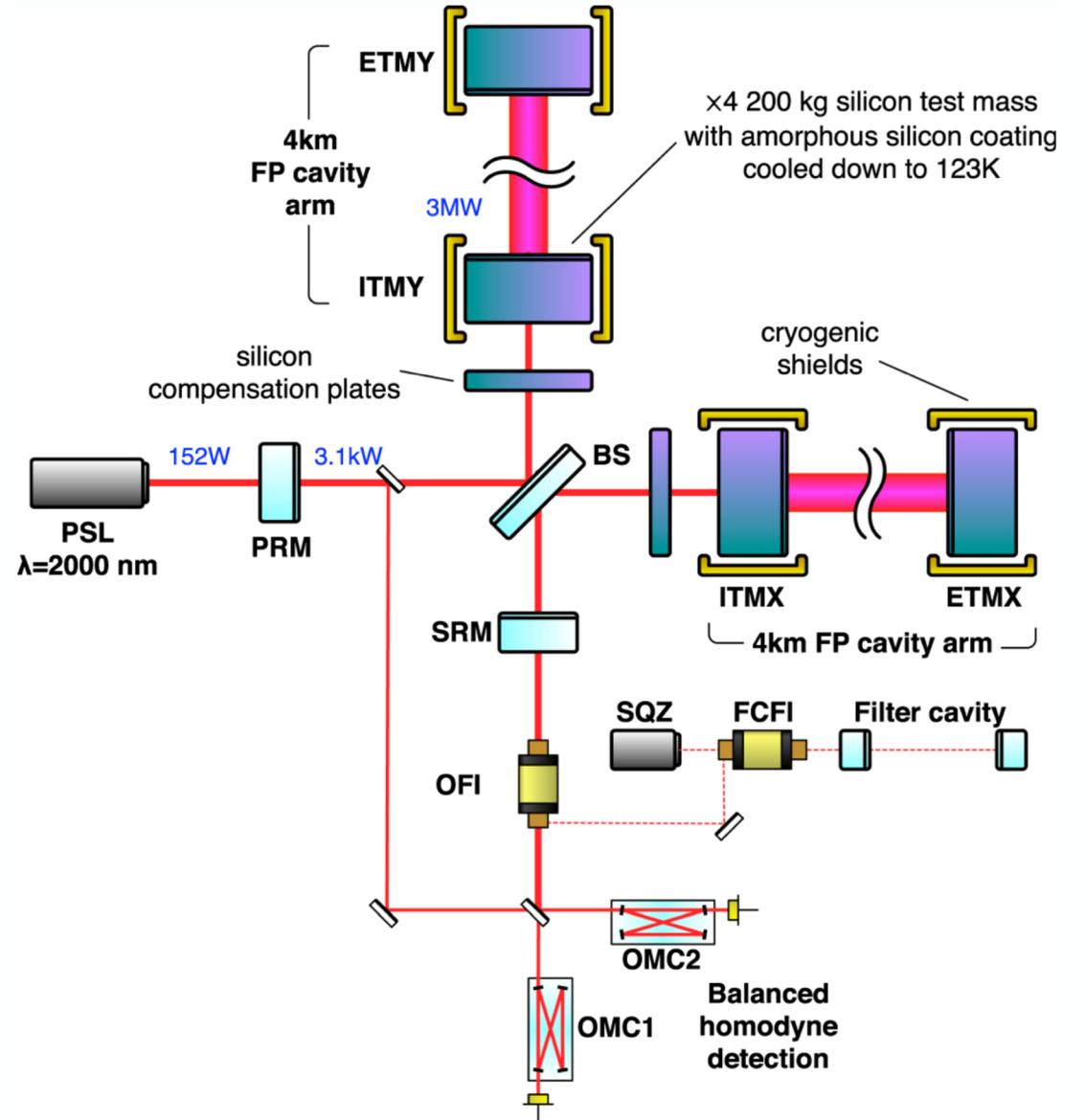
LIGO Voyager



LIGO Hanford Observatory



LIGO Livingston Observatory



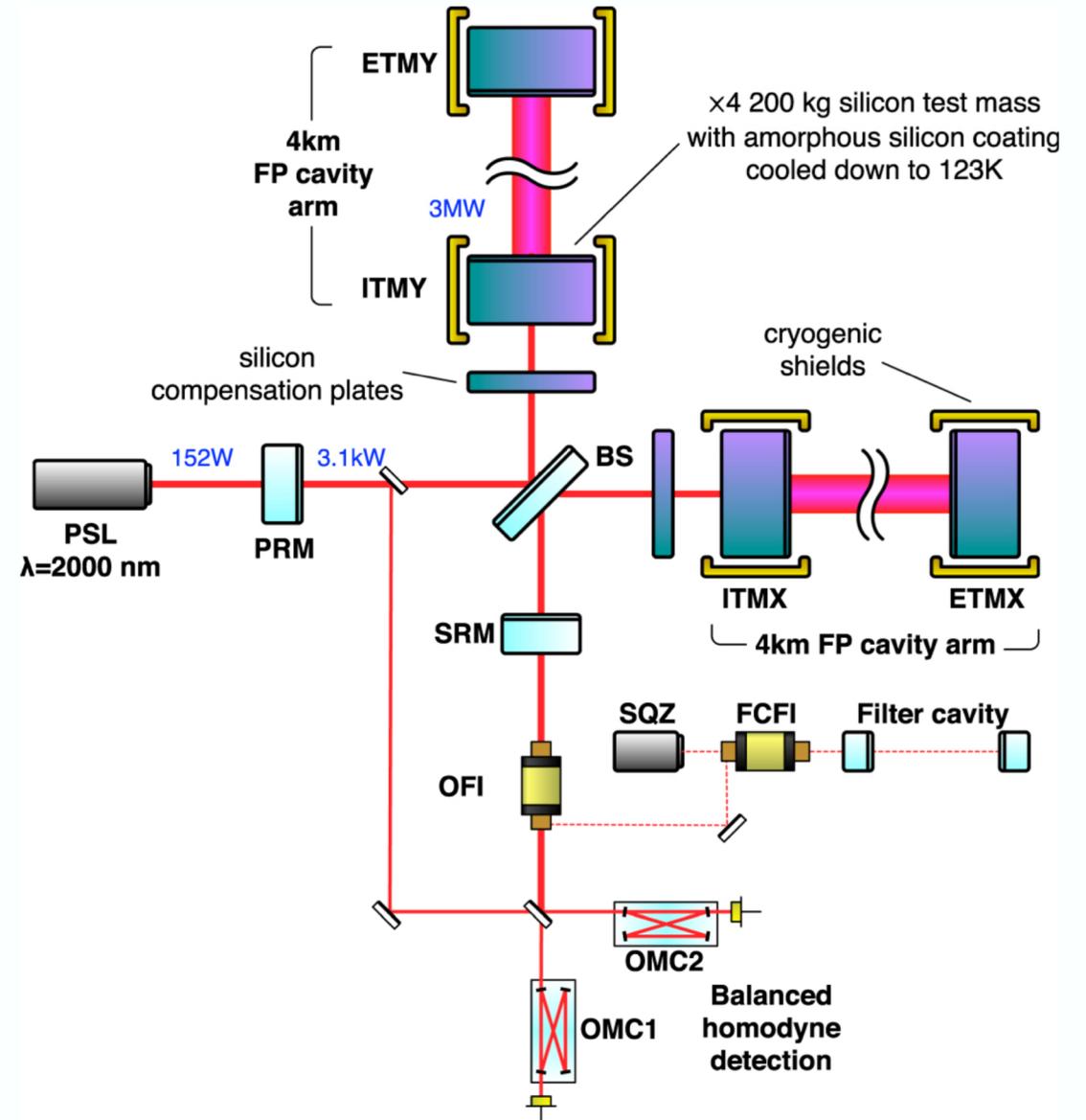
Voyager IFO schematic, from [Voyager White Paper](#)

LIGO Voyager



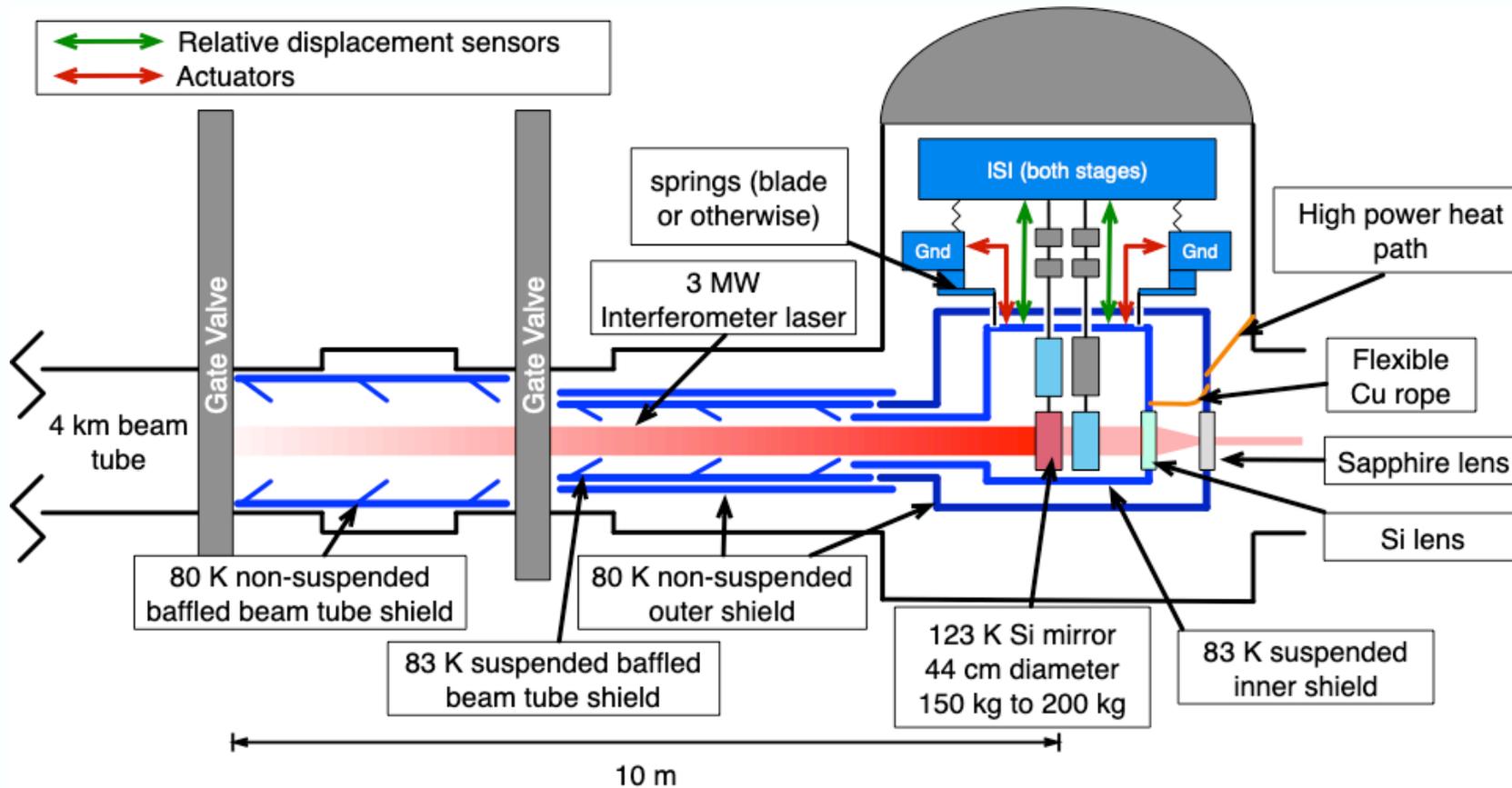
CAD drawing of LIGO India

LIGO Livingston Observatory



Voyager IFO schematic, from [Voyager White Paper](#)

Voyager Cryogenics



- Si mirrors
- 2 layers of radiative shielding
- Conductively cooled via heat extraction path
- Shields extend into beam tube
- Radiative cooling of test mass (TM) from inner shield (IS)
- Quasi-monolithic suspension: cold Si blade springs and Si ribbons

Voyager ETM chamber design, from [Voyager White Paper](#)

Mariner

- Voyager-like upgrade to Caltech 40m Lab
- Demonstration of 2- μm cryogenic silicon suspended mirror interferometry
 - ~6 kg Si optics
 - 123 K operation
 - 2050 nm pre-stabilized laser (PSL)



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Phase I	Phase II
Warm PUM (SiO_2)	Cold PUM (Si)
2-stage metal wire suspension	Si springs + ribbons (monolithic)
$\text{SiO}_2/\text{Ta}_2\text{O}_3$ coating	a-Si coating
FPMI configuration	DRFPMI configuration

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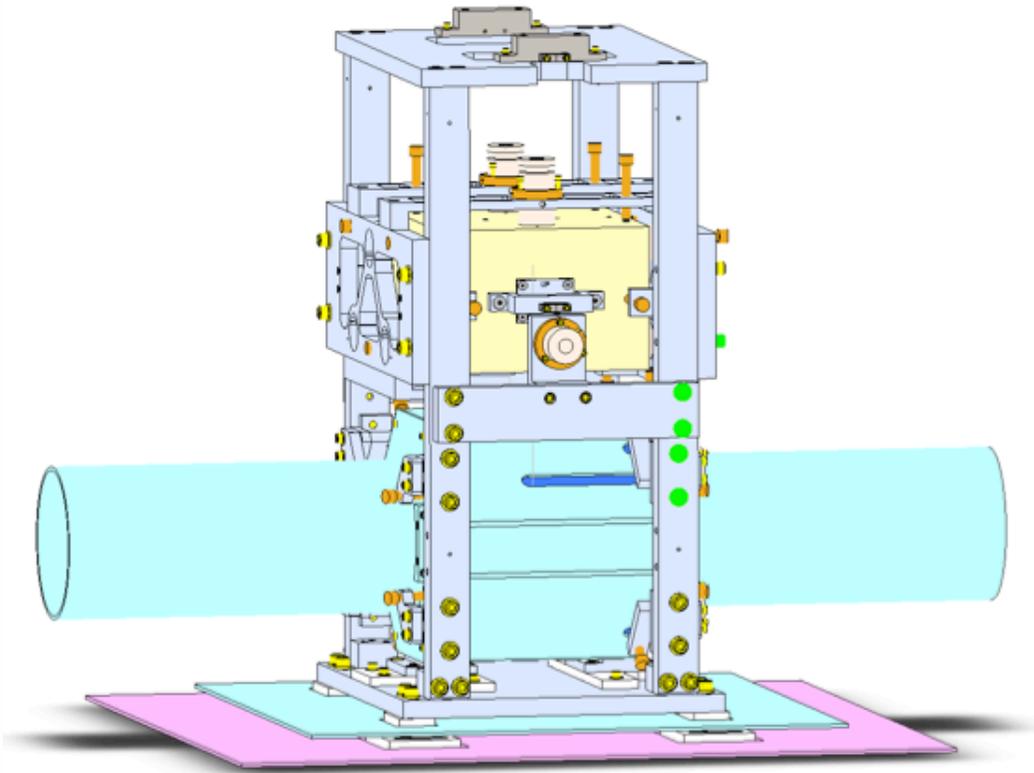


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Why 2 phases?

- Despite best efforts, unexpected challenges can arise with new technologies
 - We don't need DRFPMI to learn things
 - Ex: phase noise in Si substrate
- Integrate Phase I lessons into Phase II \rightarrow optimize for low noise

Phase I



K. Arai

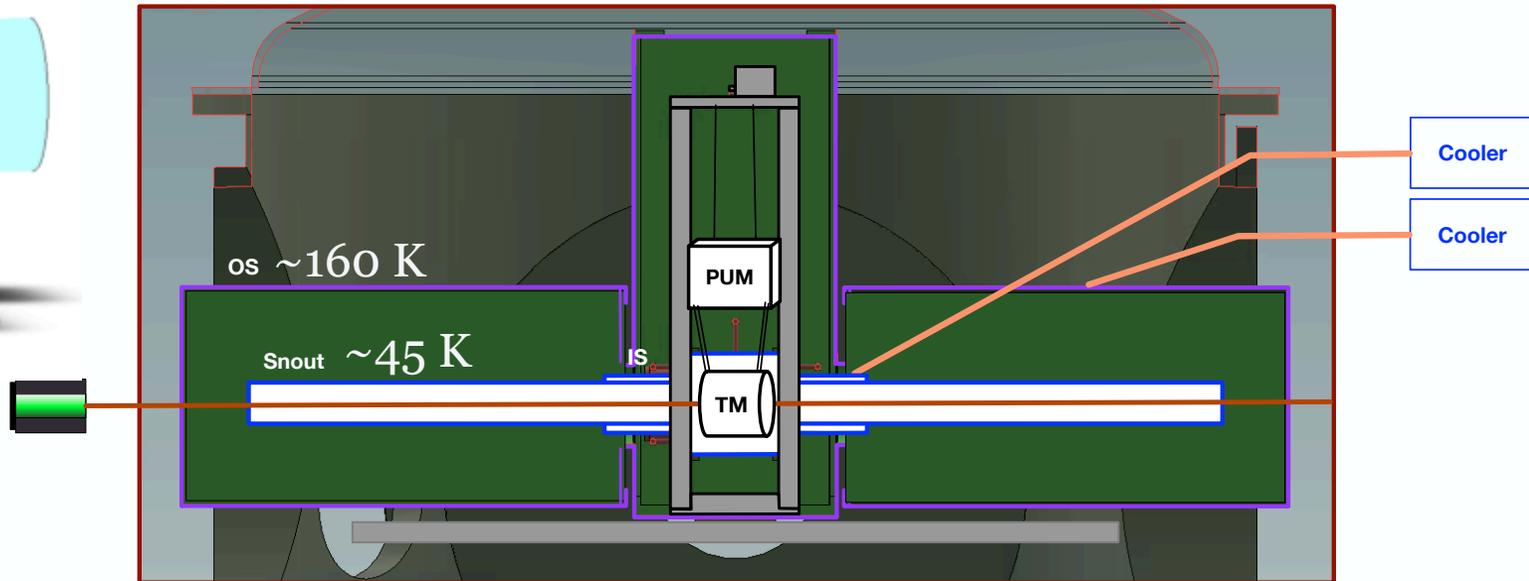
Phase I

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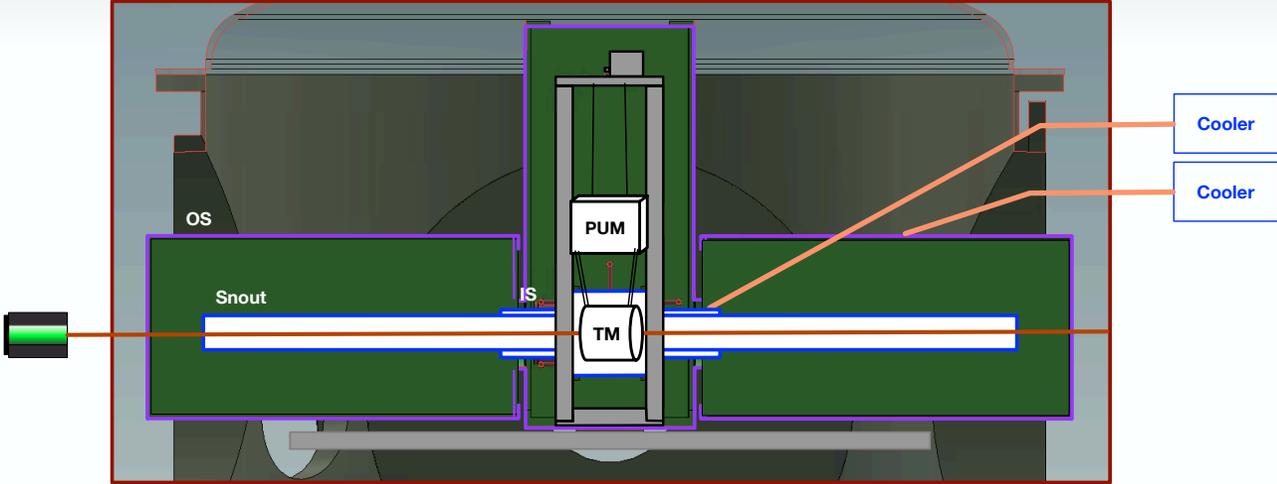
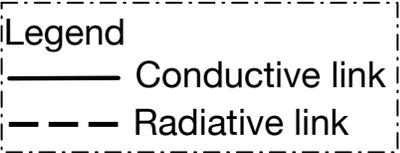
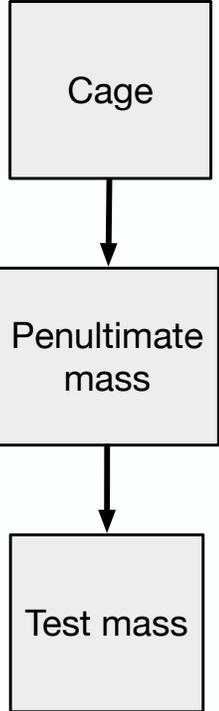
2-stage metal wire suspension

$\text{SiO}_2/\text{Ta}_2\text{O}_3$ coating

FPMI configuration



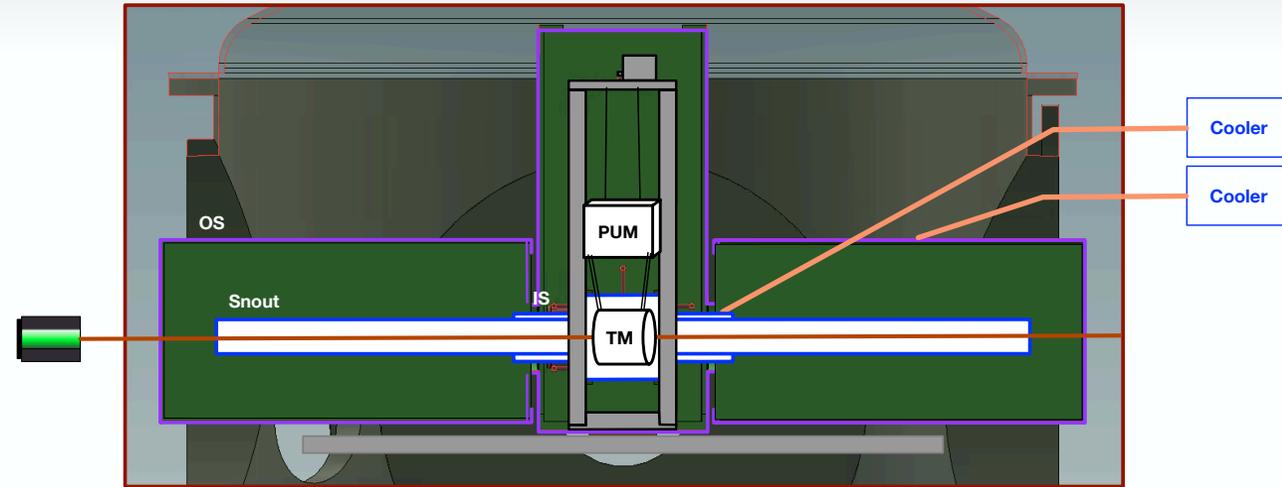
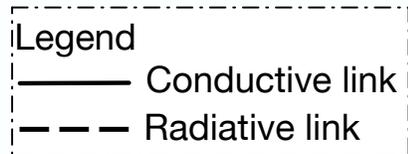
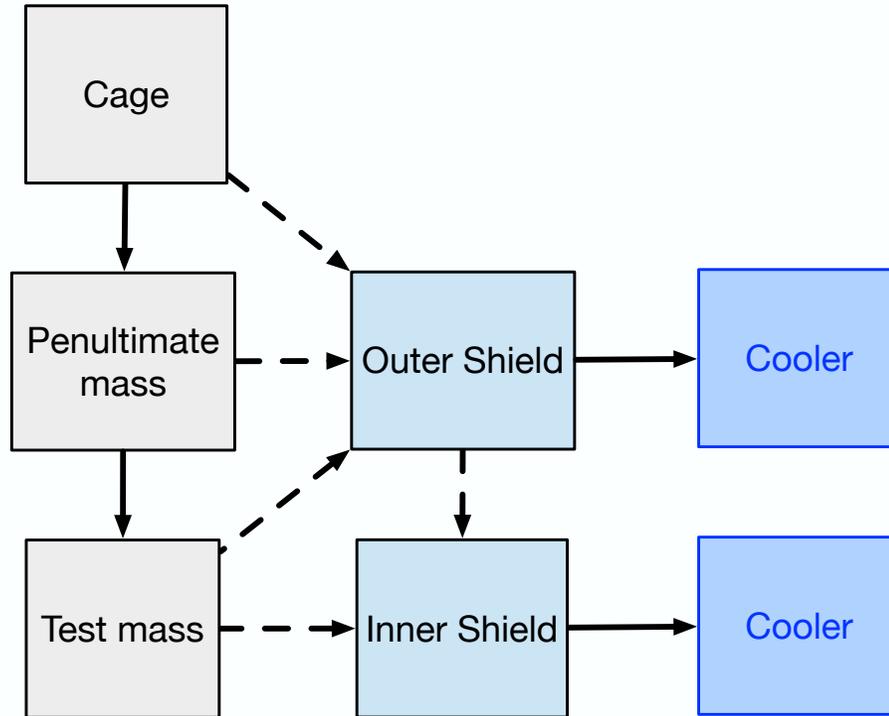
Thermal Modeling



Suspension

- Heat flow along wires to PUM and TM

Thermal Modeling



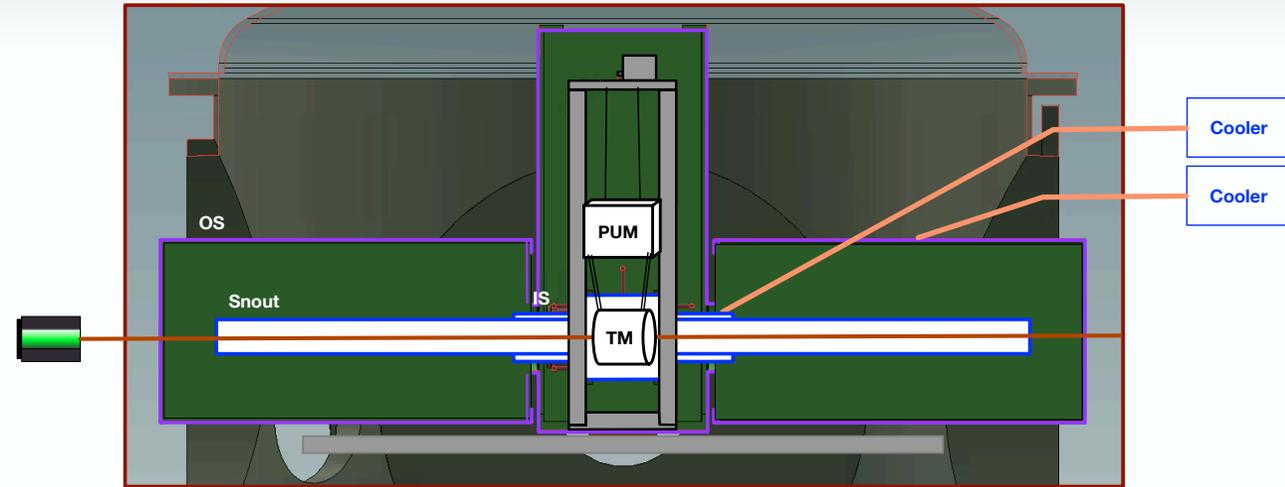
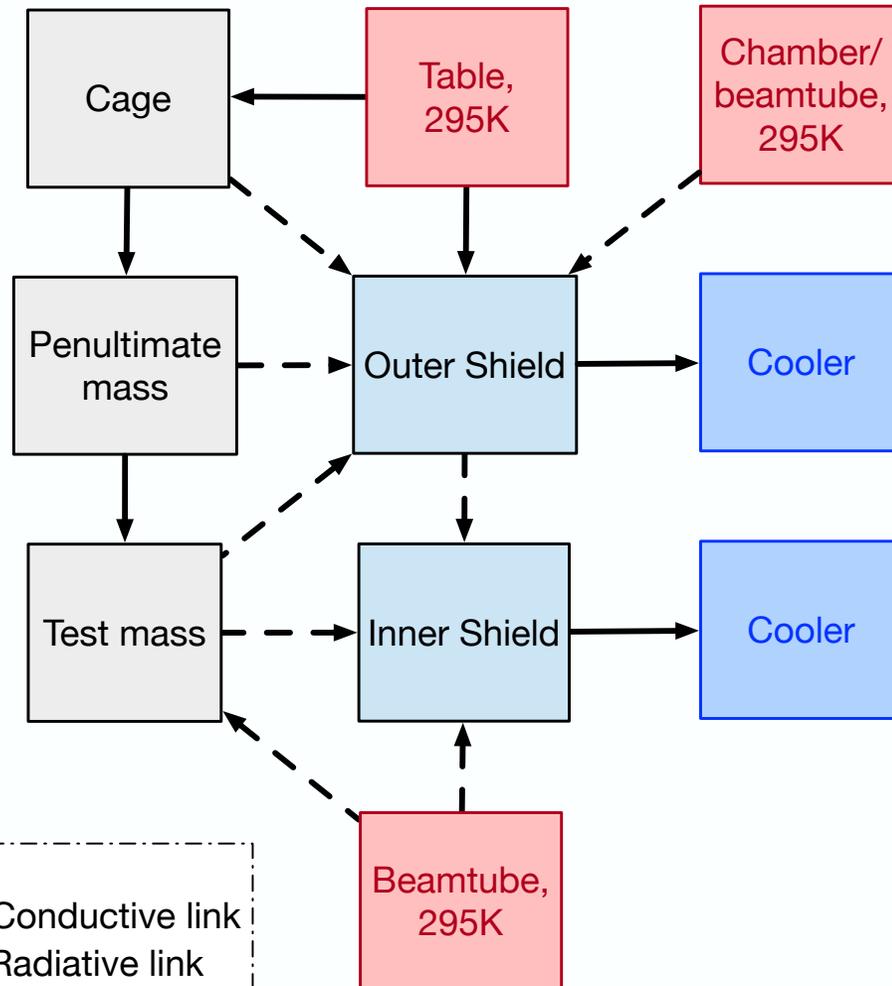
Suspension

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Cooling

- **IS/OS cooled by separate coolers**
- **TM cooled radiatively by IS**
- **View factor from OS to TM via holes in IS**

Thermal Modeling



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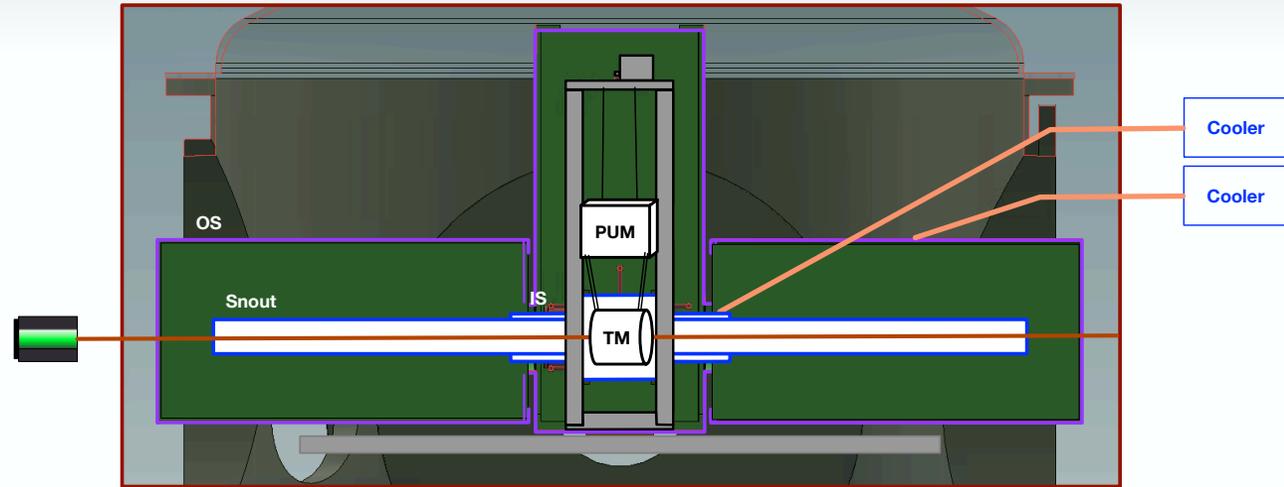
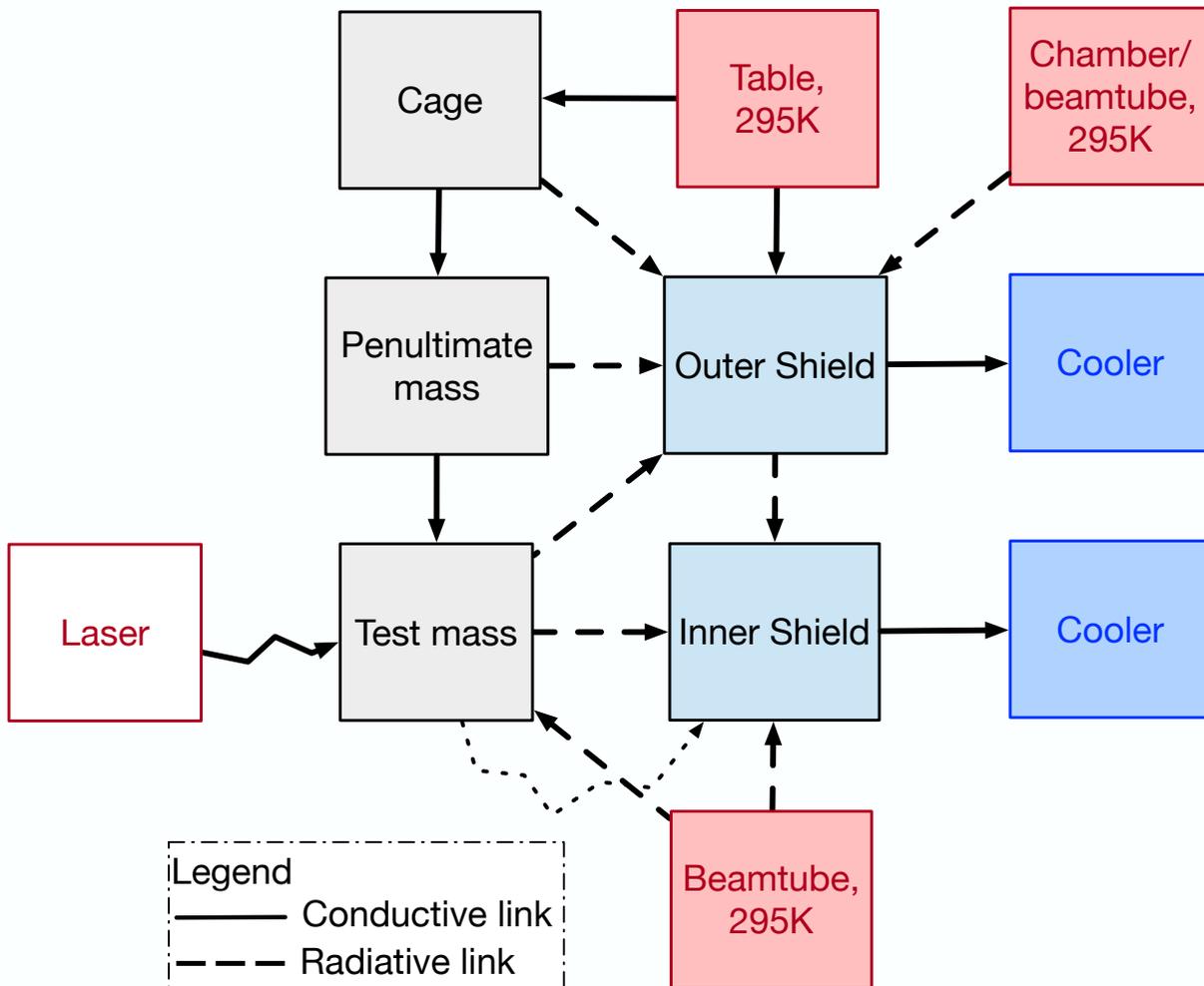
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Heating

- **Chamber and beam tube heat OS/IS**
- **View factor from beam tube to TM through IS snout**

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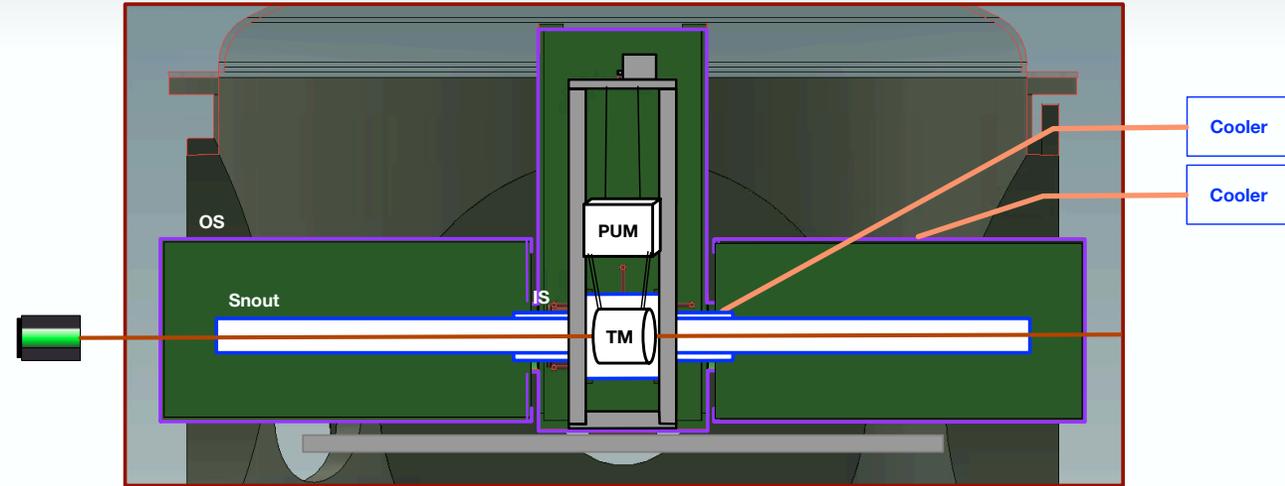
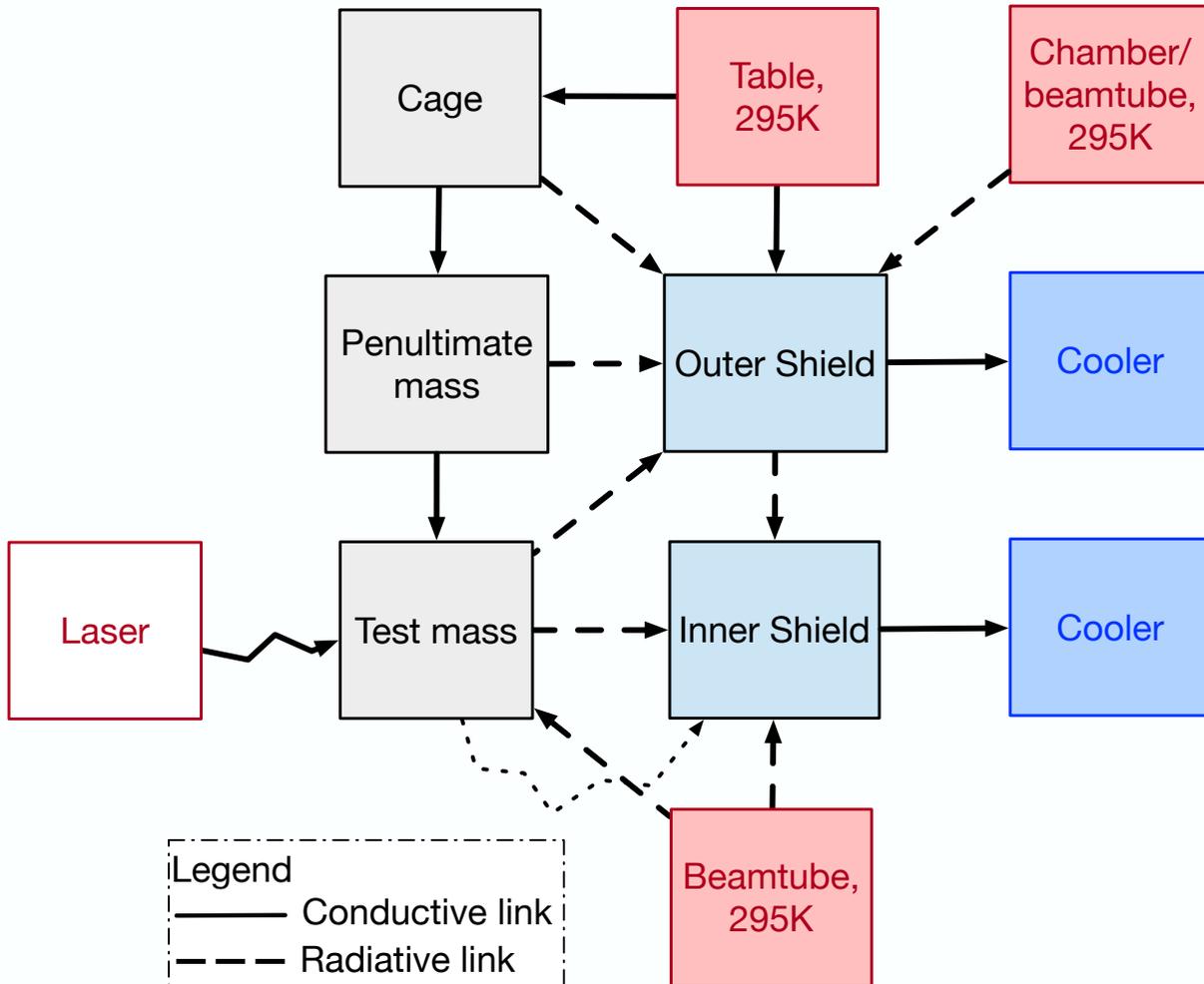
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Laser heating

- **100 mW TM absorption**
- **10 ppm scattering off of TM to IS**

Thermal Modeling



Main heat loads on test mass: **beam tube, laser**

- Snout extensions of rad shields buffer RT radiation from beam tube
 - Tradeoff: longer = better RT shielding; shorter = less thermal mass to cool
 - Shield end-to-end length: 1 m
- Can cool with laser off

What cooler do we select?

→ Consider **cooling and vibration requirements**

What's the best we can do?

- Radiative cooling limit:
 - $\varepsilon=1$
 - 0 K environment fully enclosing sample

$$mC_p \frac{dT}{dt} = -\sigma AT^4$$

T = temperature (K)

A = area (m^2)

m = mass (kg)

C_p = heat capacity ($J/kg K$)

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- Voyager time to 123 K ~68 h (< 3 days)

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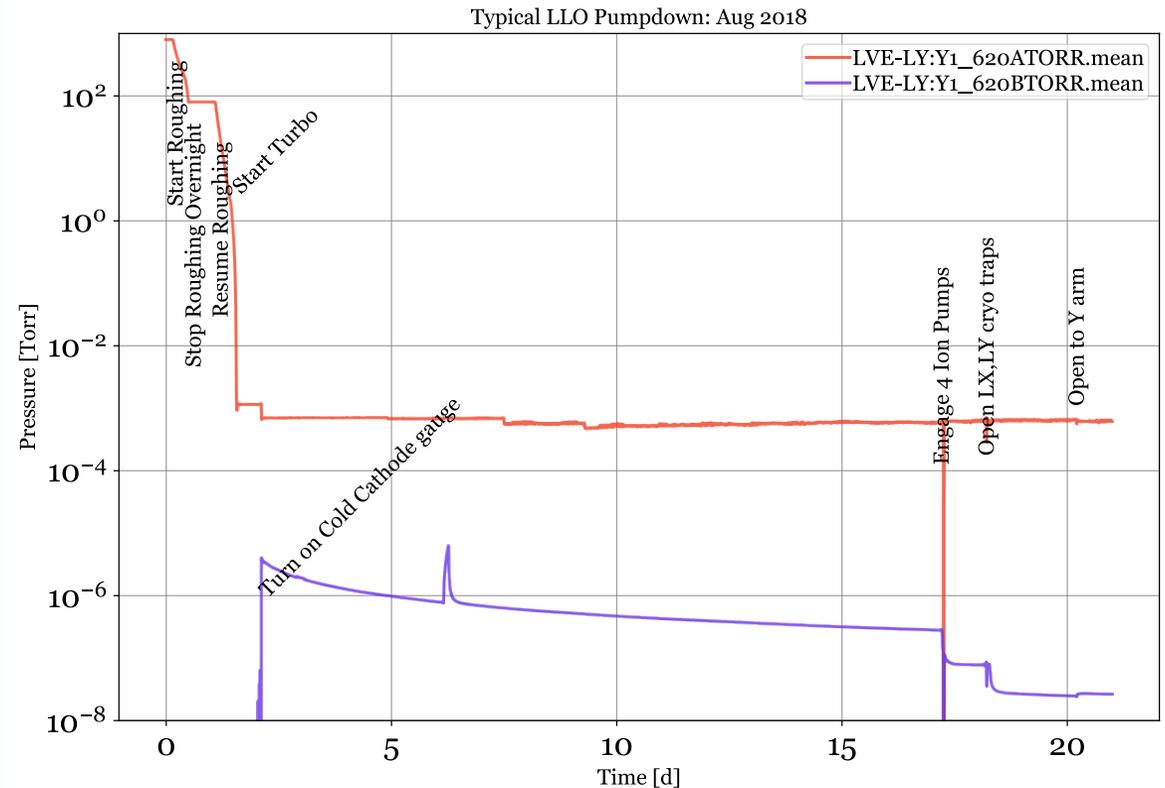
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LLO pumpdown: ~18 days



Cooling Requirements

1. Time to cool Si optic to 123 K should be comparable to vacuum pump down time

Mariner Goals

Goal #1

$\tau_{123\text{K}} \sim 24 \text{ h}$

Cooling Requirements

1. Time to cool Si optic to 123 K should be comparable to 40m vacuum pump down time
 - Need excess cooling power at 123 K to offset TM heat loads
2. Cooling power delivered to TM should be ~2x the heating power at 123 K

Mariner Goals

Goal #1	$\tau_{123K} \sim 24 \text{ h}$
Goal #2	$P_{cool, 123K} \geq 0.7 \text{ W}$

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Voyager Goals

Goal #1	$\tau_{123K} \sim 18 \text{ d}$
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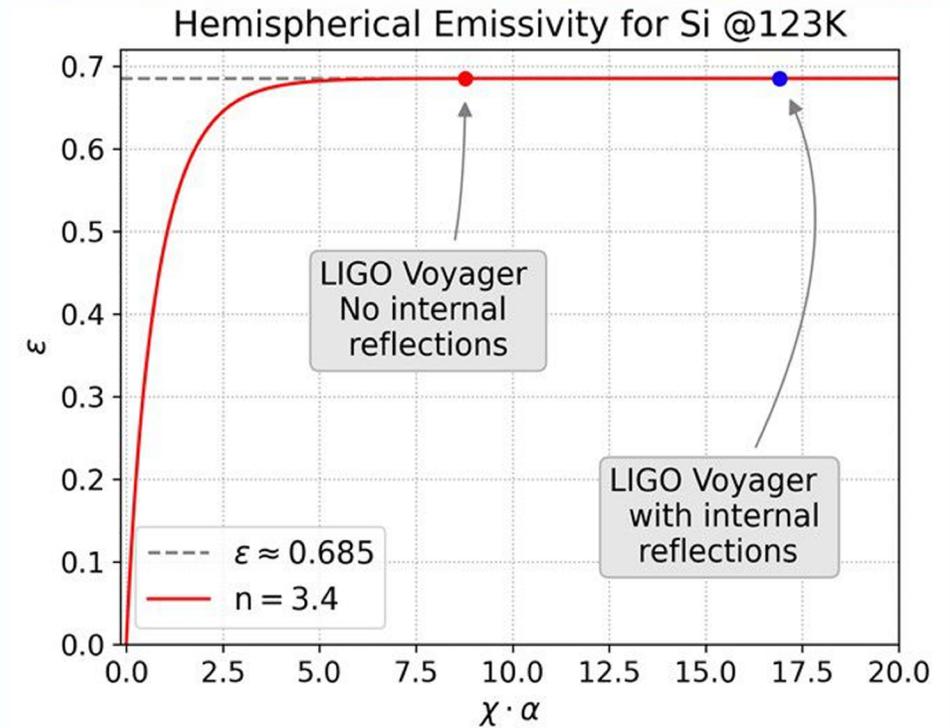
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TM Barrel Emissivity

- Is bare Si enough?
 - Voyager TM effective emissivity: $\varepsilon \approx 0.685$

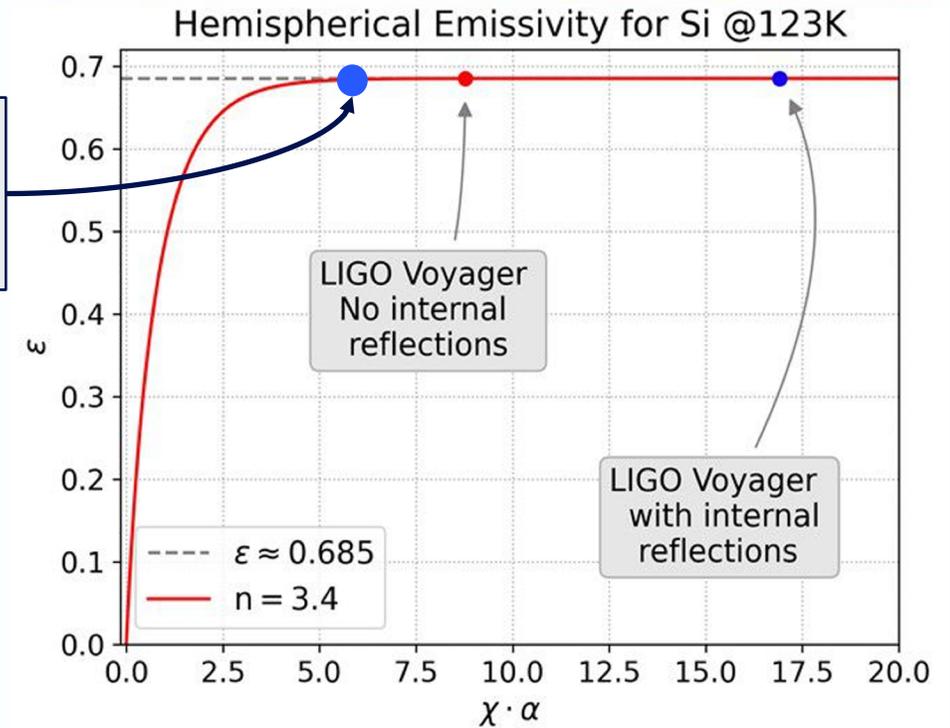


J. Reis, O. Aguiar [G2101151](#)

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 - Mariner TM effective emissivity: $\varepsilon \approx 0.685$

Mariner
with internal
reflections

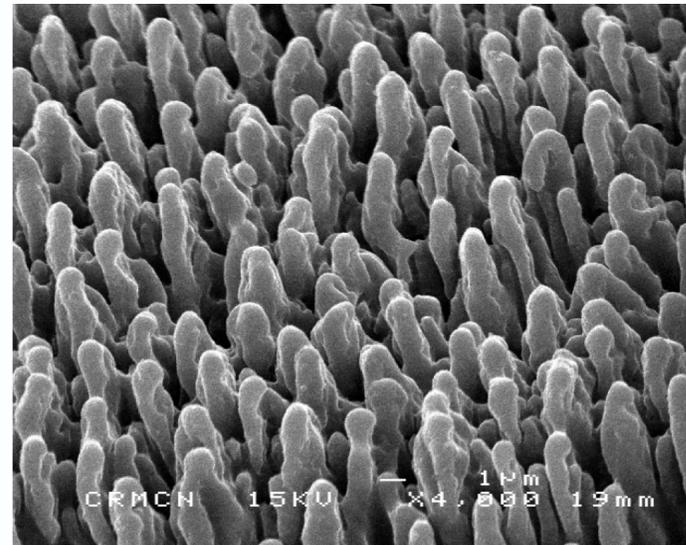
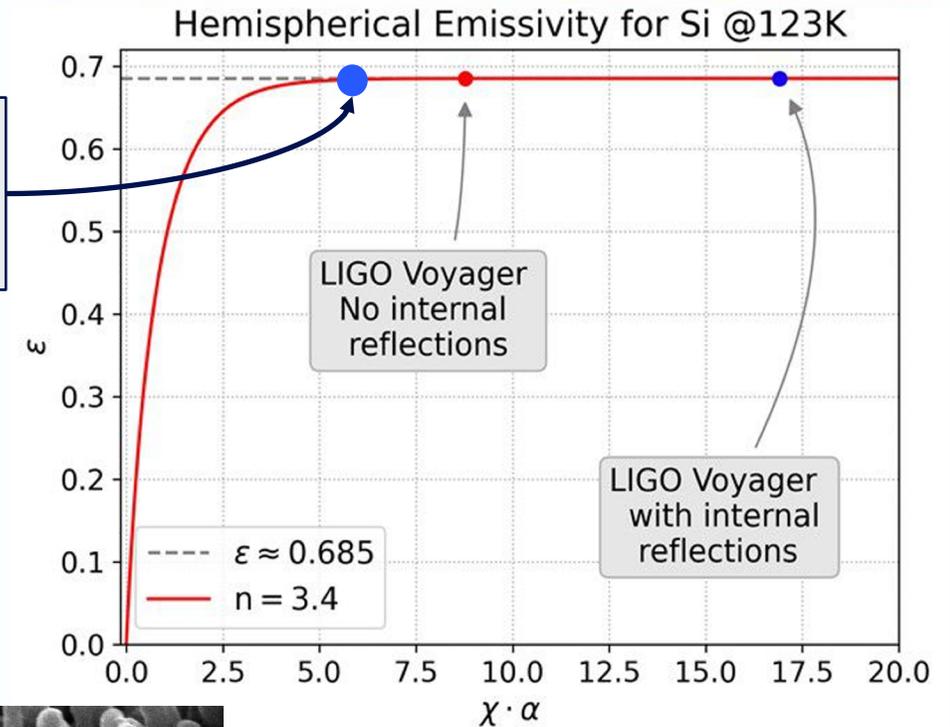


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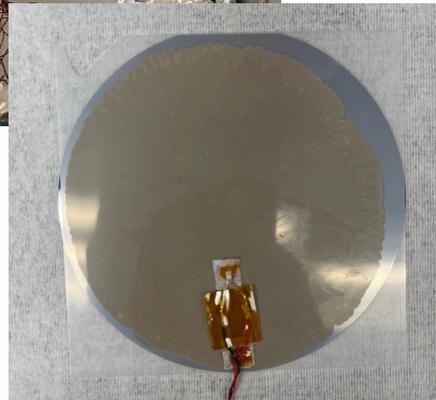
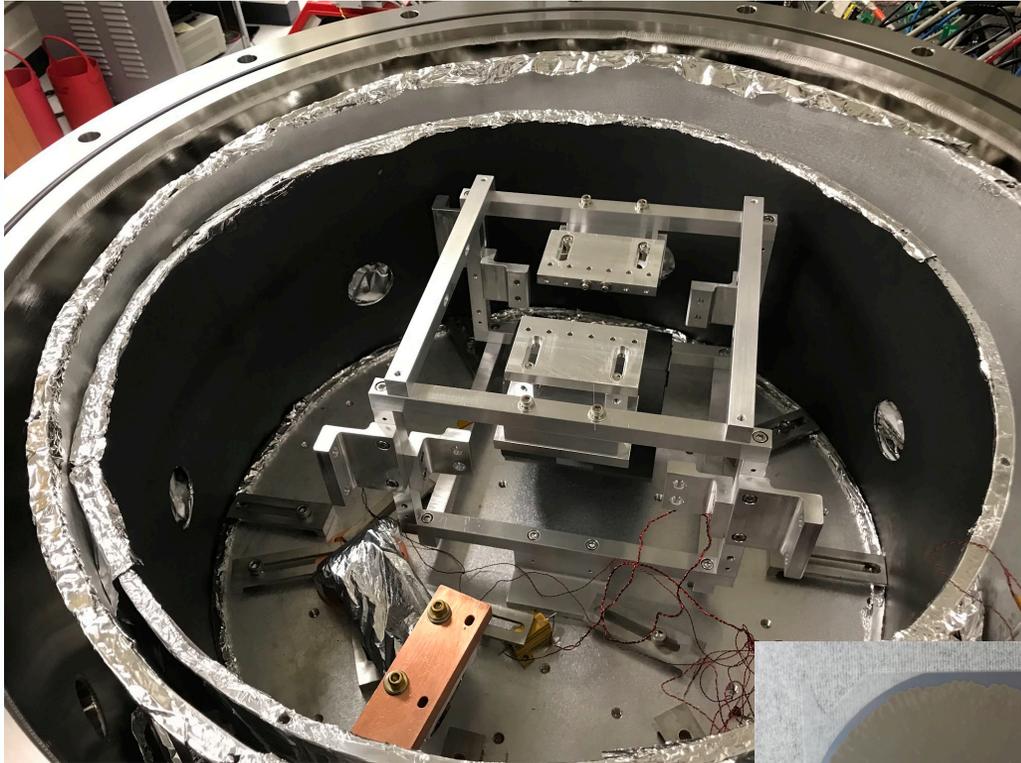
- Is bare Si enough?
 - Voyager TM effective emissivity: $\epsilon \approx 0.685$
 - Mariner TM effective emissivity: $\epsilon \approx 0.685$
- High emissivity barrel coating
- \rightarrow maximizes radiative coupling between TM and cold shielding
- Black Si
 - Surface modification of Si – creates needle-shaped structure
 - $\epsilon \sim 0.8$

Mariner with internal reflections



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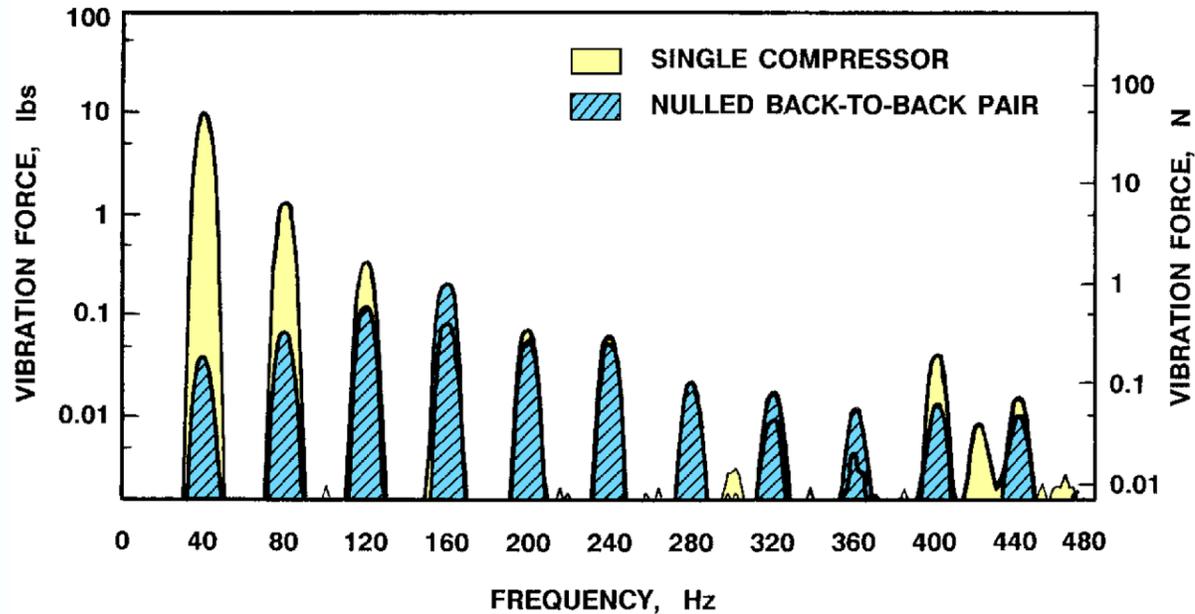
Cooling Tests



- Cool Si samples in tabletop cryostat
 - Extract cooldown times to 123 K
- Estimate emissivities
- Test bonding methods

→ Cryostat upgrades in progress to simplify thermal modeling and increase cooling efficiency

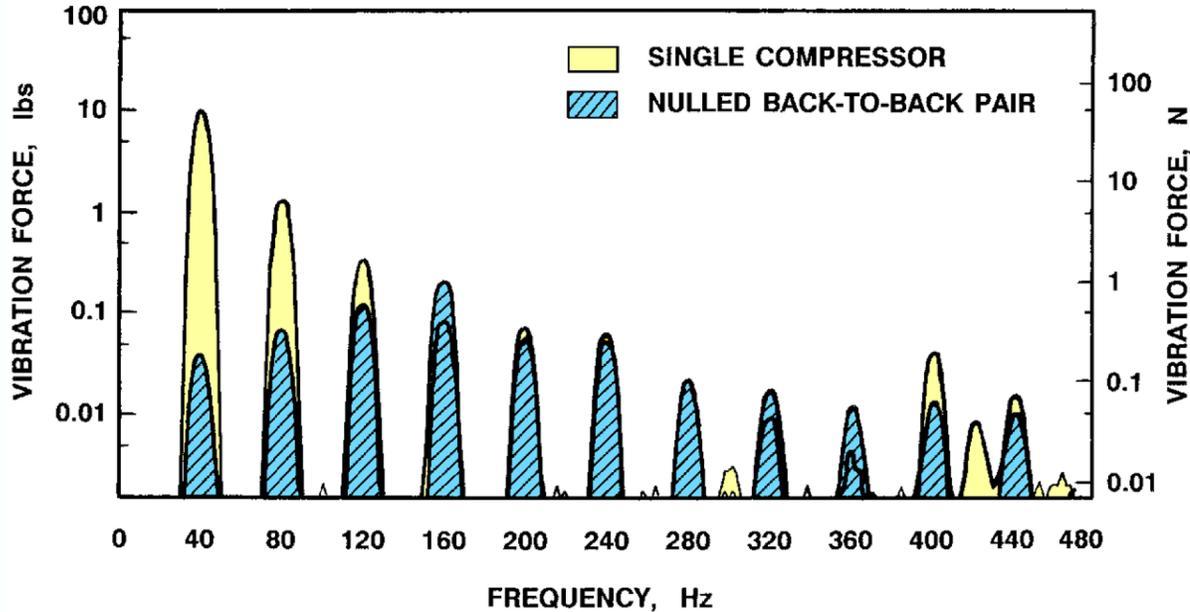
Vibration Considerations



Vibration spectrum generated by a BAe 80K Oxford-style, linear-drive compressor with a 40-Hz drive frequency.

R. Ross, [Vibration suppression of advanced space cryocoolers](#)

Vibration Considerations



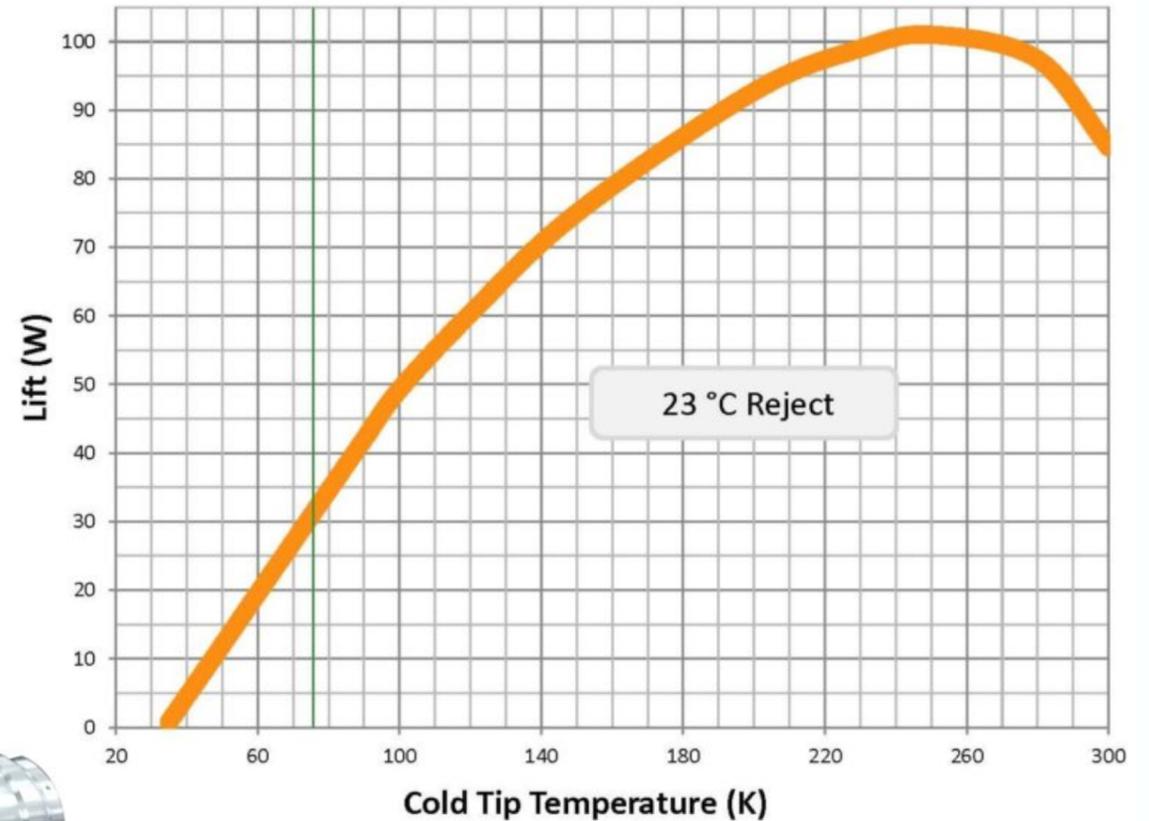
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Sunpower CryoTel DS 30

Nominal Performance



[Sunpower Inc.](#)

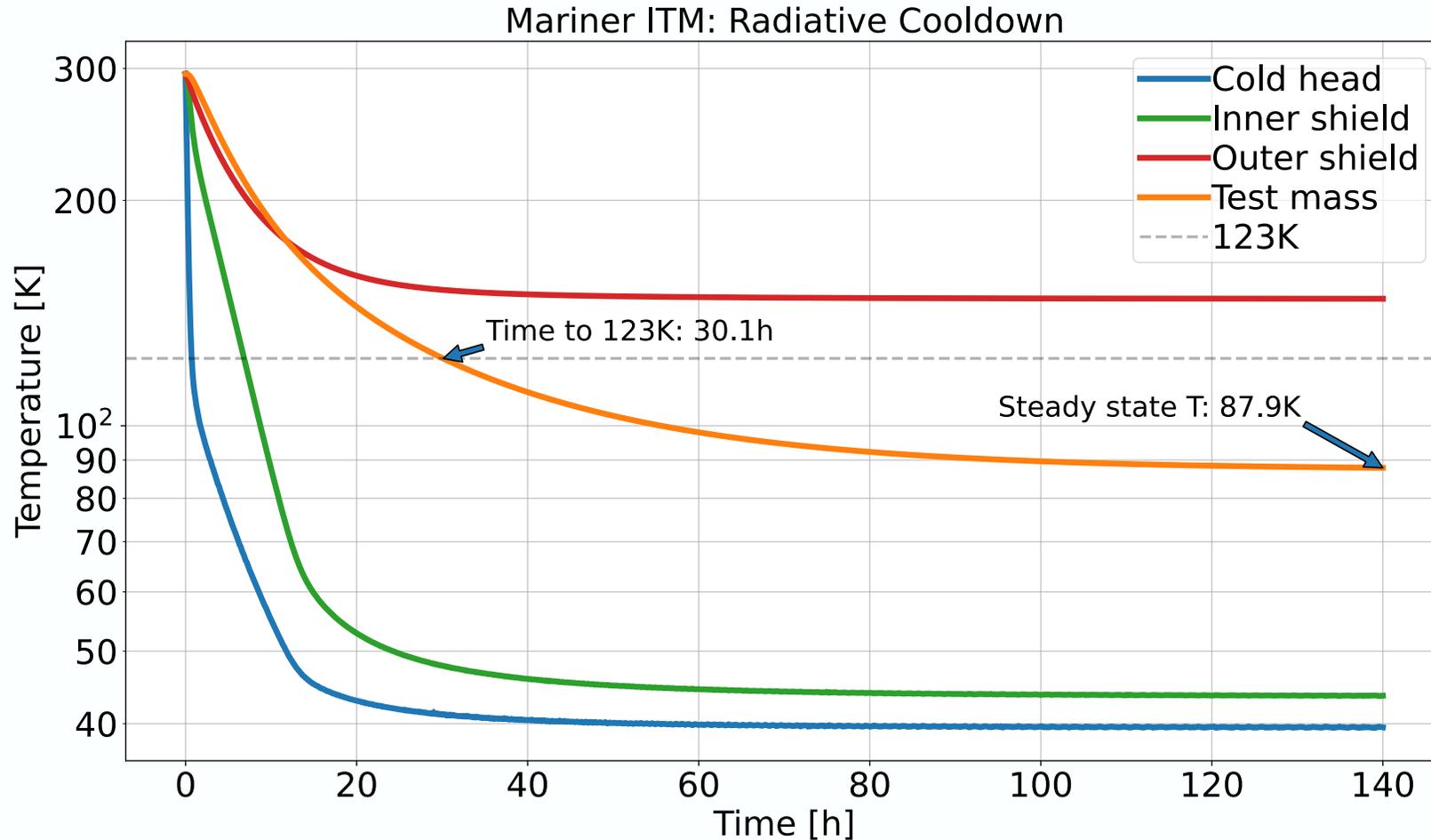
Mariner Cooldown

Goal #1

$$\tau_{123K} < \sim 24 \text{ h}$$

Goal #2

$$P_{cool, 123K} \geq 0.7 \text{ W}$$



- Sunpower DS 30 (Stirling) cryocooler
- High emissivity (0.9) coating on Si barrel
- High emissivity (0.9) coating on inner shield (inner surface)
- Laser off during cooldown
- $\tau_{123K} \sim 30 \text{ h}$
- $P_{cool, 123K} = 1 \text{ W}$

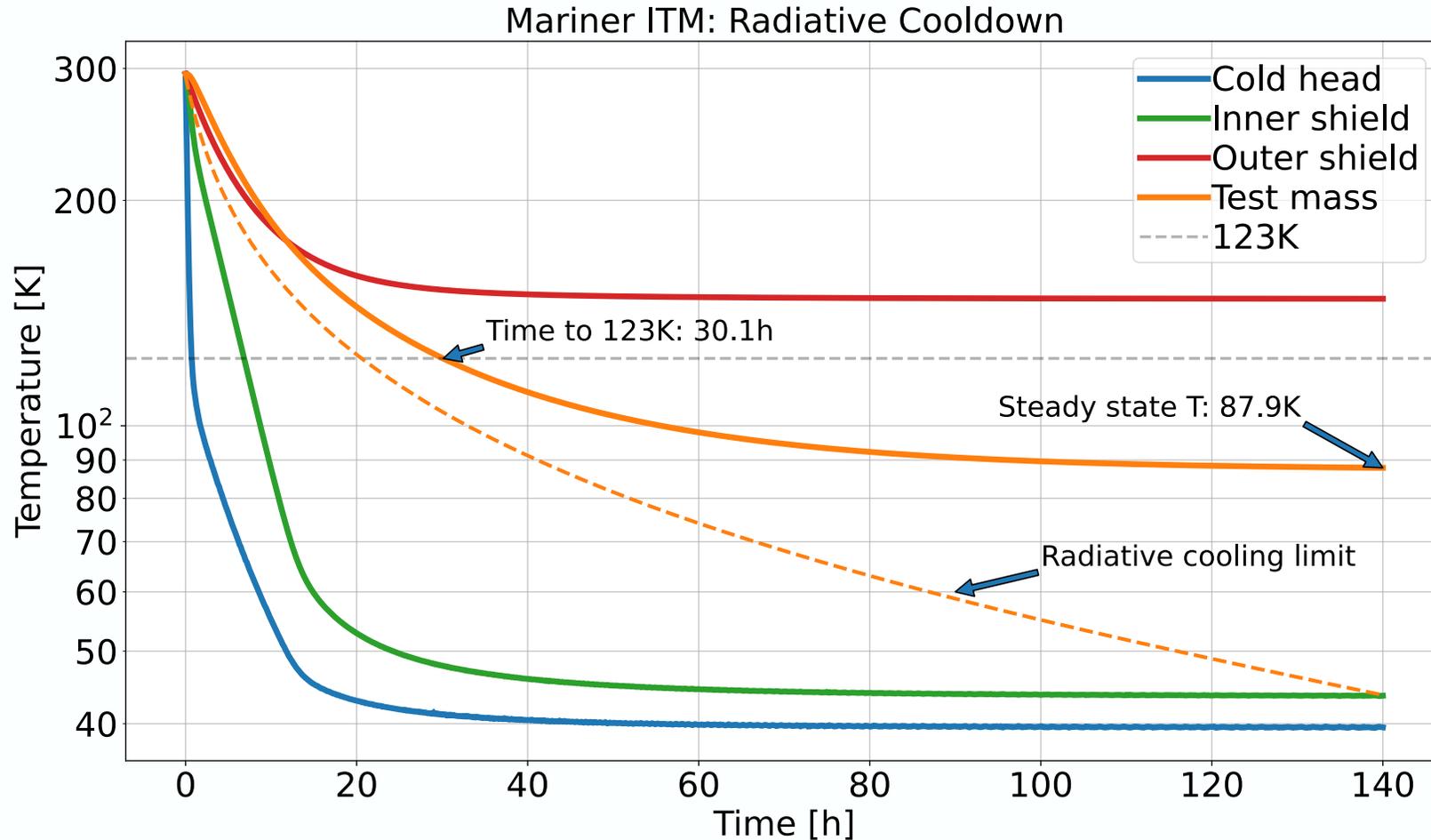
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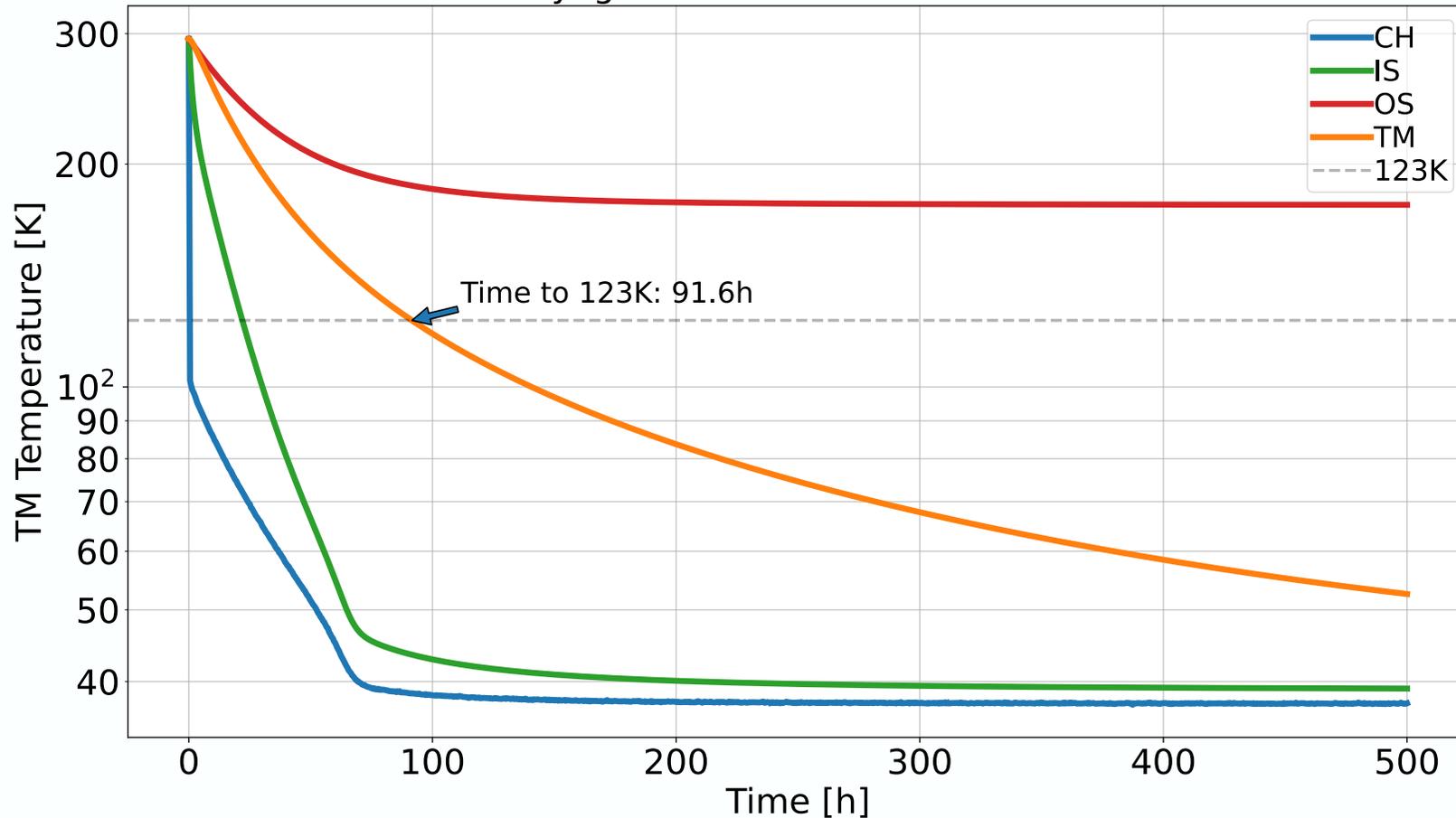


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Voyager Cooldown

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Voyager ITM: Radiative Cooldown

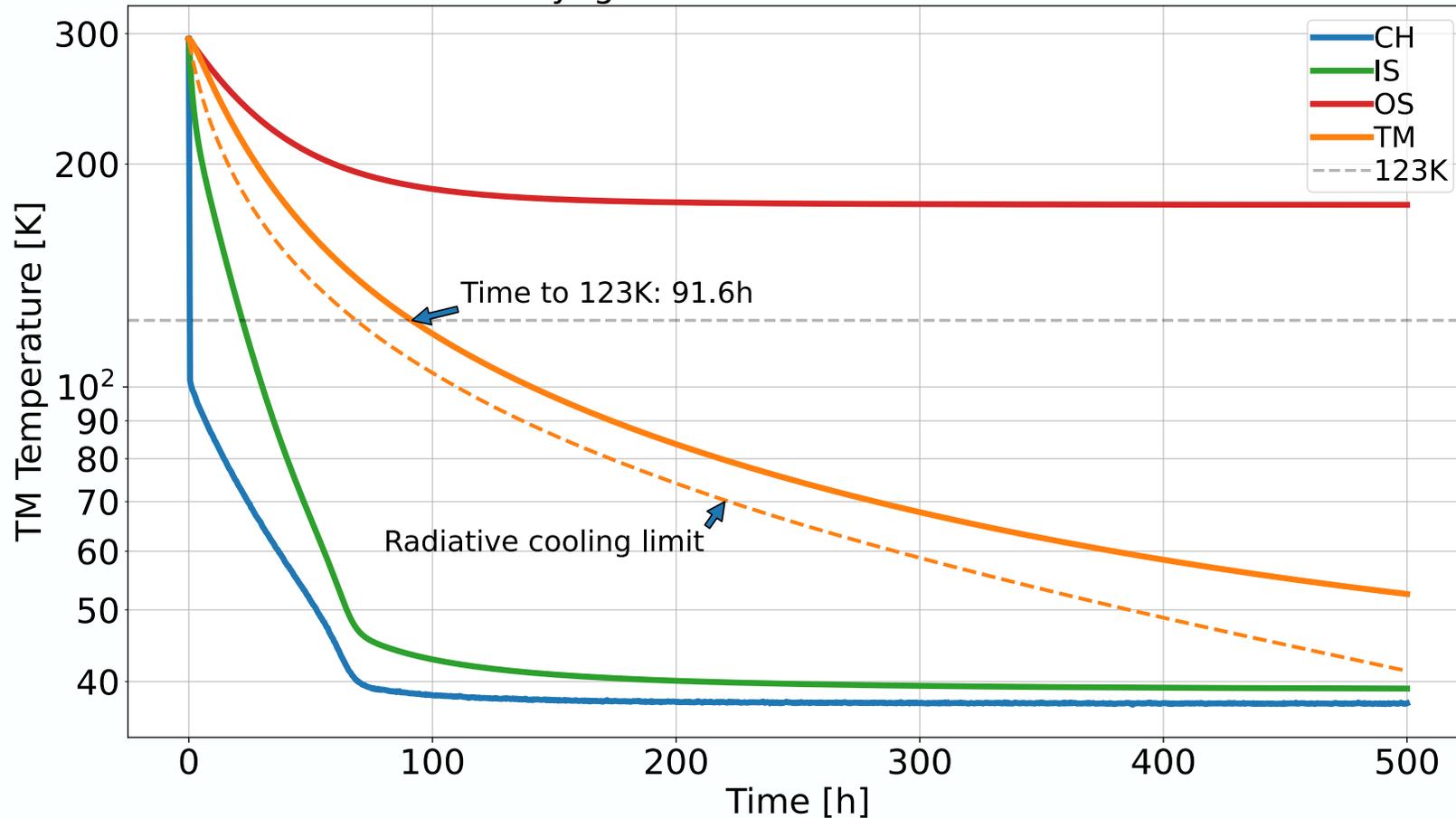


- Need $\sim 1 \text{ kW}$ cooling power at 123 K
 - Multiple coolers?
 - LN2?
- High emissivity (0.9) coating on Si barrel
- High emissivity (0.9) coating on inner shield (inner surface)
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- $\tau_{123K} \sim 92 \text{ h}$
- $P_{cool, 123K} = 11 \text{ W}$

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Future Work

- Thermal modeling in COMSOL
- Testing of more high-emissivity coatings necessary for optimal cooldown of Mariner/Voyager TMs
- Mariner Phase I/II will inform Voyager cryogenic design



Backup Slides

Si Properties

Property	Silicon at 123 K	Silicon at 295 K	Fused silica at 295 K
Thermal expansion (1/K)	1×10^{-9}	2.6×10^{-6}	3.9×10^{-7}
Specific heat (J/kg/K)	300	700	739
Thermal conductivity (W/m/K)	700	130	1.38
Refractive index	3.5	3.45	1.438
dn/dT	1×10^{-4}	1.78×10^{-4}	1.1×10^{-5}

[Voyager White Paper](#)

Voyager SUS Parameters

Parameters	aLIGO	150 kg test mass	200 kg test mass
P , Main chain payload (kg)	124	470	470
m_1 (kg)	22	148.8	116.5
m_2 (kg)	22	101.7	87.6
m_3 (kg)	40	69.5	65.9
m_4 (kg)	40	150	200
Total length to optic center (mm)	1642	1642	1642
L_1 (mm)	422	410.5	410.5
L_2 (mm)	277	410.5	410.5
L_3 (mm)	341	410.5	410.5
L_4 (mm)	602	410.5	410.5
fiber/ribbon material	silica fiber	silicon ribbon	silicon ribbon
σ_4 , fiber/ribbon stress (MPa)	774	150	150
PUM springs	no	yes	yes
10 Hz seismic noise (m/ $\sqrt{\text{Hz}}$)	1.7e-19	8.8e-20	1.9e-19
10 Hz long. thermal noise (m/ $\sqrt{\text{Hz}}$)	6.5e-19	1.4e-19	1.6e-19
10 Hz damping noise est (m/ $\sqrt{\text{Hz}}$)	1.7e-20	5.1e-21	1.7e-20

Thermal Modeling

Radiation:

$$P_{ij} = \frac{\sigma(T_i^4 - T_j^4)}{\frac{1 - \varepsilon_i}{A_i \varepsilon_i} + \frac{1 - \varepsilon_j}{A_j \varepsilon_j} + \frac{1}{A_i F_{ij}}}$$

Conduction:

$$P_{ij} = \frac{\kappa A}{l} (T_i - T_j)$$

P = power (W)

T = temperature (K)

ε = emissivity

A = area (m²)

l = length (m)

F = geometric view factor

κ = thermal conductivity (W/mK)

Current thermal model (python) integrates a system of ODEs to model temperature vs. time of major chamber components

- Estimates radiative view factors
- Temperature-dependent heat capacity, thermal conductivity