

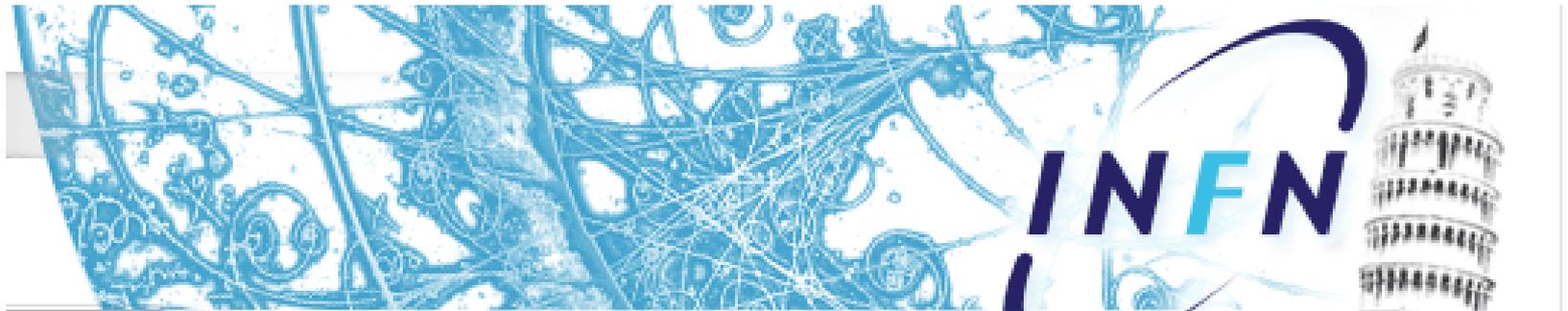


SVT Mechanics

F. Bosi, A. Bernardelli

INFN-Pisa

on behalf of the SuperB SVT Group





Outline



- SVT components design update
- Be pipe (L=390 mm)
- I.R. general layout update (cryostat flanges position)
- LO pixel module support: micro-channel bidirectional flow
- SVT Quick Demounting
- Conclusion



SVT Components Update



1) Difference shift of the L2 and L3 modules position w.r.t. the I.P.

- Reason: better condition for clearance between L2 and L0



Change for the fanout dimensions

2) Be pipe L=390 mm

- Reason -> better clearance Be beam-pipe flanges/L0 striplets cold flanges
- > more space for coolant feeding circuit Be beam-pipe



Reduction of the C.F. semicone nose length

Cooling ring L1/2 forw/back moved outside from I.P. (3.5 mm)

Differente design for L0 striplets Cold flanges



SVT Module Master tablet



Layer Shift in Z direction

SVT - Dimensioni e copertura angolare sensori

A	B	C	D	E	F	G	H	I	J	K	L	M	N	O
Layer	Radius piano y-z sensore barrel SuperB	Radius piano y-z punto estremo sensore wedge SuperB	Radius punto estremo laterale sensore SuperB	Lunghezza orizzontale sensore tangente cono 300 mrad SuperB	Lunghezza sensore barrel SuperB (tabella Londra)	Lunghezza totale sensore barrel SuperB	Lunghezza totale sensore SuperB	Lunghezza estensione sensore oltre 300 mrad SuperB column (G-E)/2	Lunghezza estensione sensore oltre 350 mrad BaBar	Angolo intercettato nel punto ingombro estremo sensore con piano y-z (rad)	Angolo intercettato nel punto ingombro estremo laterale sensore (rad)	Shift Layer asse Z (mm)	Angolo intercettato nel punto ingombro estremo sensore con piano y-z+shift (rad)	Angolo intercettato ingombro fisico sensore estremo laterale +shift (rad)
0	15,10	-	17,30	97,63	-	104,00	104,00	3,19	-	0,283	0,321	0	-	-
1	32,85	-	36,97	212,39	214,78	223,36	223,36	5,48	21,69	0,286	0,320	+2	0,284	0,325
2	39,85	-	44,26	257,65	262,78	265,78	265,78	4,06	2,51	0,291	0,322	-2	0,293	0,326
3	58,85	-	65,28	380,49	385,70	385,70	385,70	2,60	1,41	0,296	0,326	0	-	-
4A	119,85	87,91	90,54	574,60	457,95	457,95	578,23	2,05	1,96	0,295	0,303	+2	0,293	0,293
4B	123,85	91,91	94,42	597,69	479,42	479,42	599,70	1,14	1,07	0,297	0,305	+2	0,296	0,296
5A	139,85	112,18	114,25	732,47	613,04	613,04	737,46	2,72	2,58	0,295	0,300	-2	0,297	0,297
5B	143,85	116,18	118,18	756,53	635,84	635,84	760,26	2,05	1,93	0,297	0,301	-2	0,298	0,298

Modules have sensor in symmetric position but respect I.P. they are shifted along z direction to avoid middle dead space

New Layer Shift in Z direction

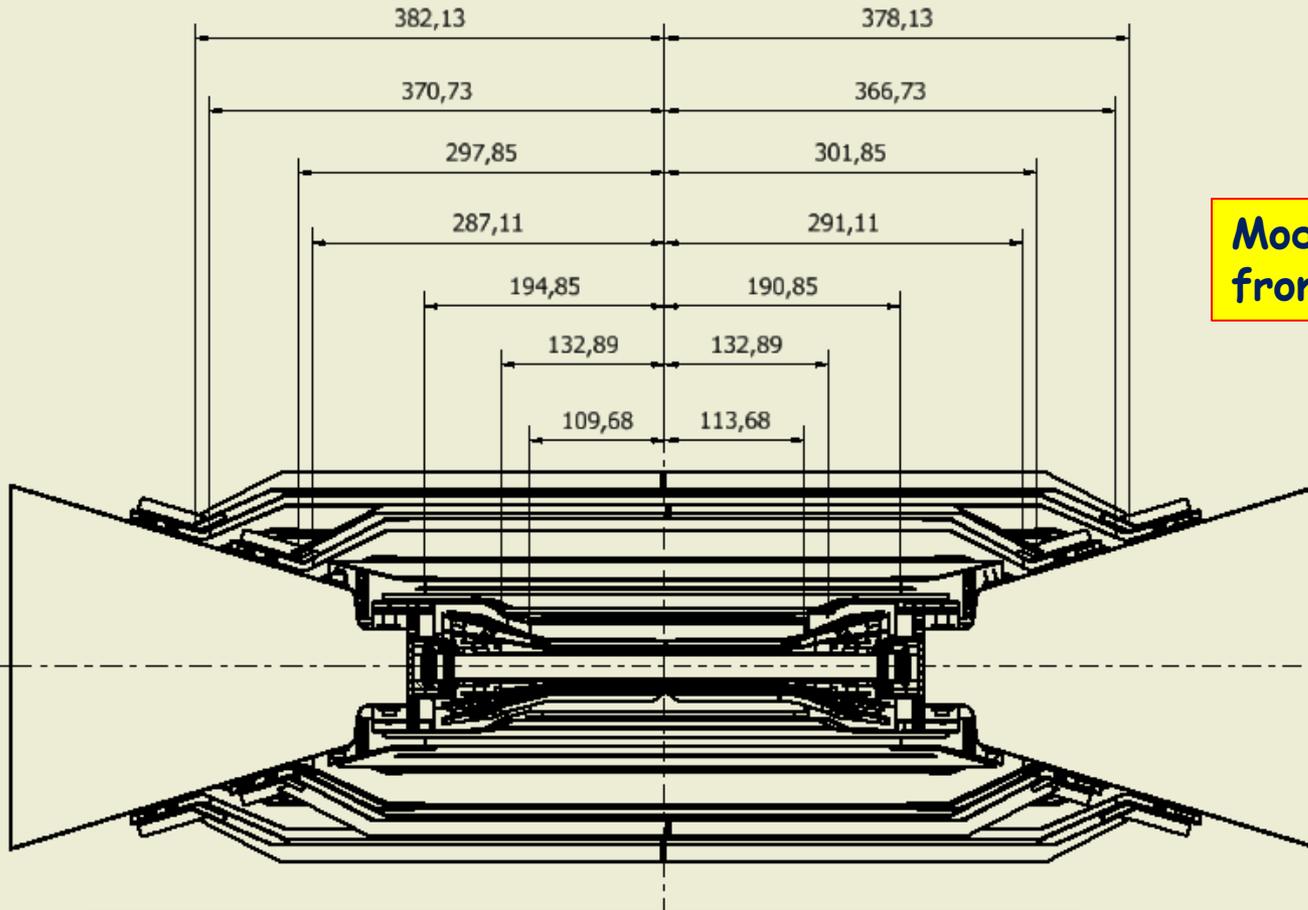
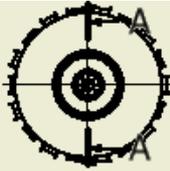
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1	32,85	-	36,97	212,39	214,78	223,36	223,36	5,48	21,69	0,286	0,320	+2	0,284	0,325
2	39,85	-	44,26	257,65	262,78	265,78	265,78	4,06	2,51	0,291	0,322	0	0,293	0,326
3	58,85	-	65,28	380,49	385,70	385,70	385,70	2,60	1,41	0,296	0,326	-2	-	-
4A	119,85	87,91	90,54	574,60	457,95	457,95	578,23	2,05	1,96	0,295	0,303	+2	0,293	0,293
4B	123,85	91,91	94,42	597,69	479,42	479,42	599,70	1,14	1,07	0,297	0,305	+2	0,296	0,296
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5B	143,85	116,18	118,18	756,53	635,84	635,84	760,26	2,05	1,93	0,297	0,301	-2	0,298	0,298

New Layers shift

L3: shifted -2mm in Z direction.
 L2: symmetric w.r.t. I.P. because this layout allow better clearance condition between L0/L2

Modules have sensor in symmetric position but respect I.P. Modules are shifted along z direction to avoid middle dead space

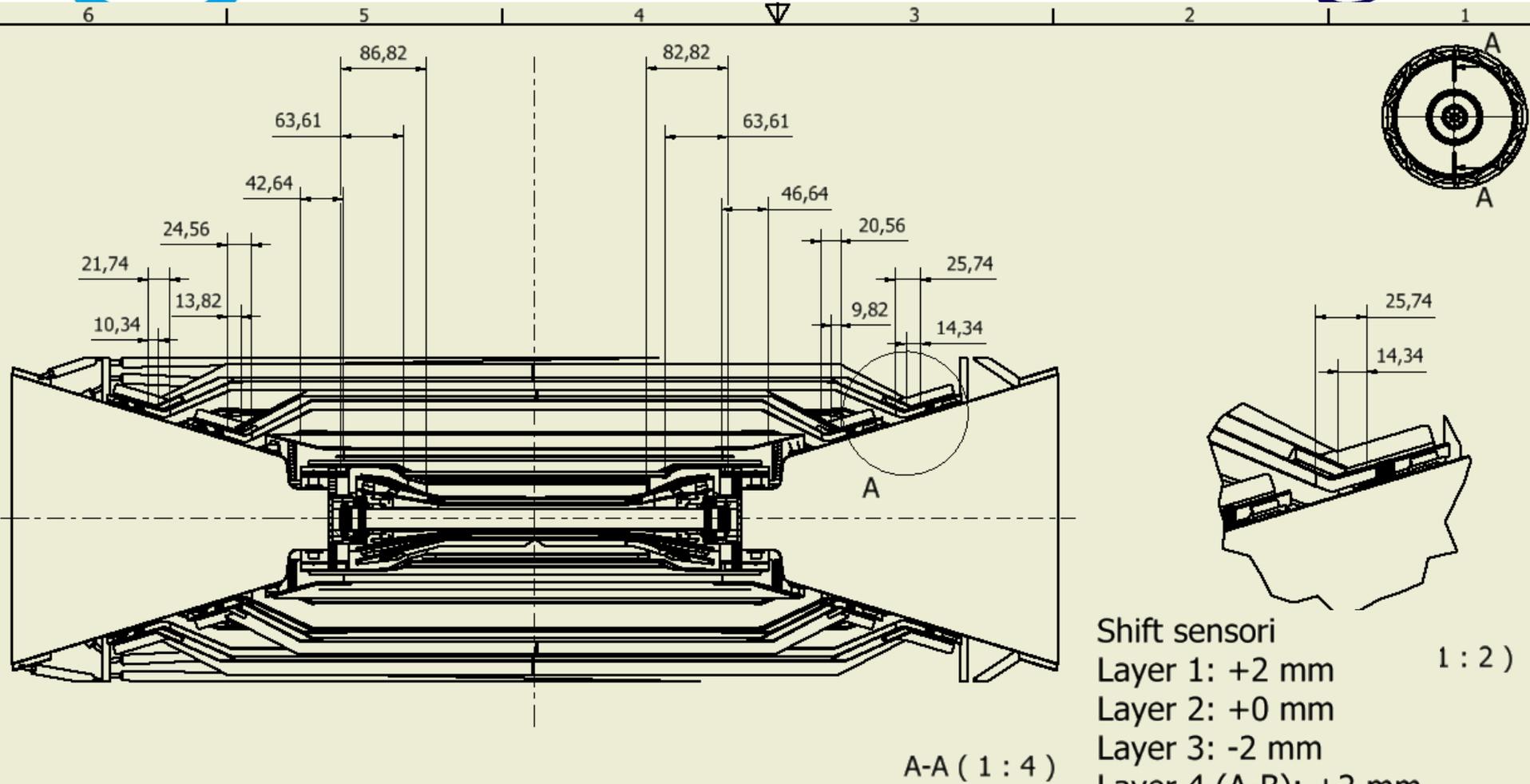


Module Sensors distance from I.P. for layer L1/5

Shift sensori
 Layer 1: +2 mm
 Layer 2: +0 mm
 Layer 3: -2 mm
 Layer 4 (A-B): +2 mm
 Layer 5 (A-B): -2 mm

Reference drawing for module position shift from I.P.

A-A (1 : 4)

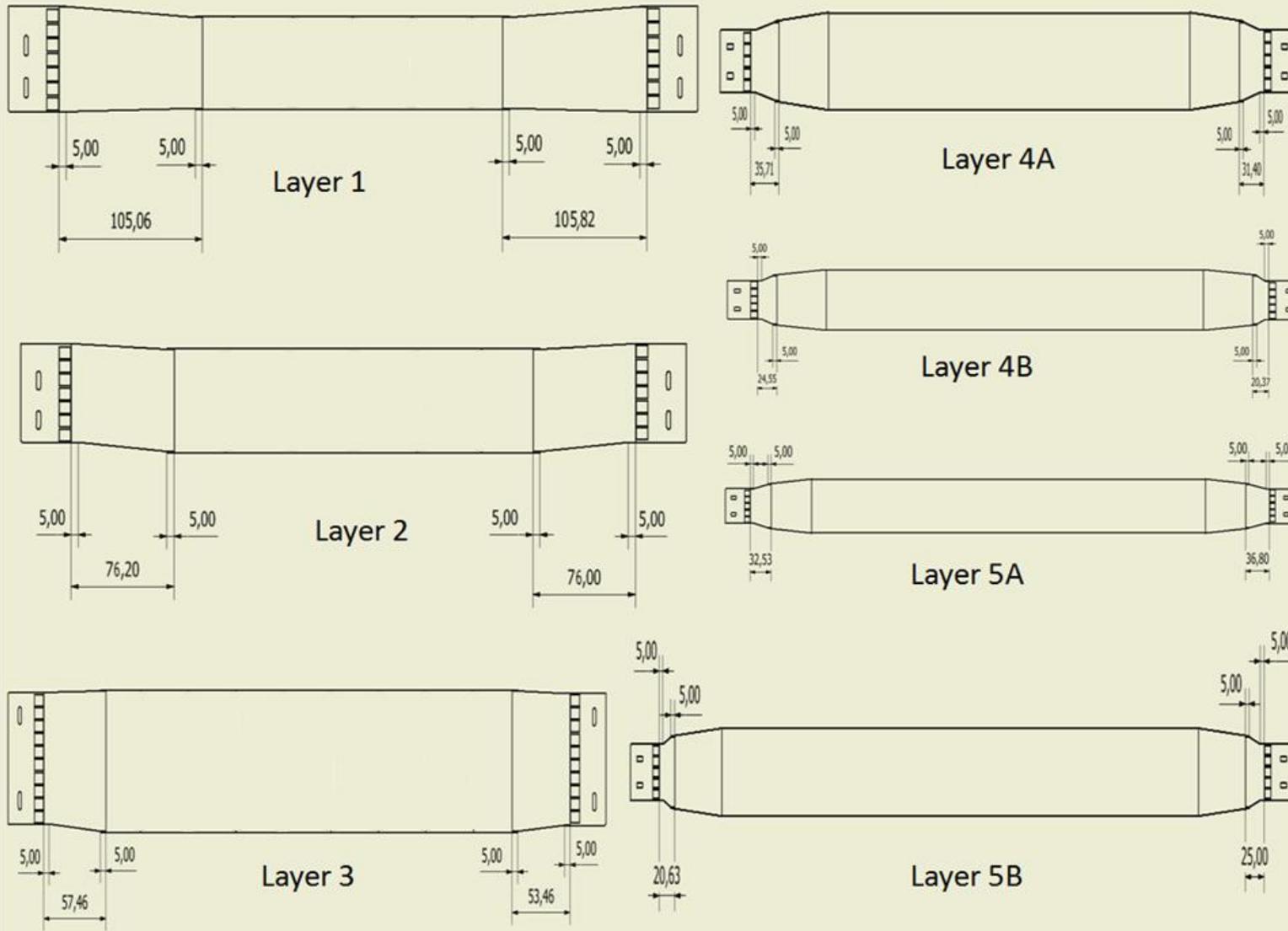


Shift sensori
 Layer 1: +2 mm
 Layer 2: +0 mm
 Layer 3: -2 mm
 Layer 4 (A-B): +2 mm
 Layer 5 (A-B): -2 mm

1 : 2)

Reference drawing for sensor/HDI distance (on Z axis)

Fanout dimension (not in scale)





SVT L1-5 Layout design



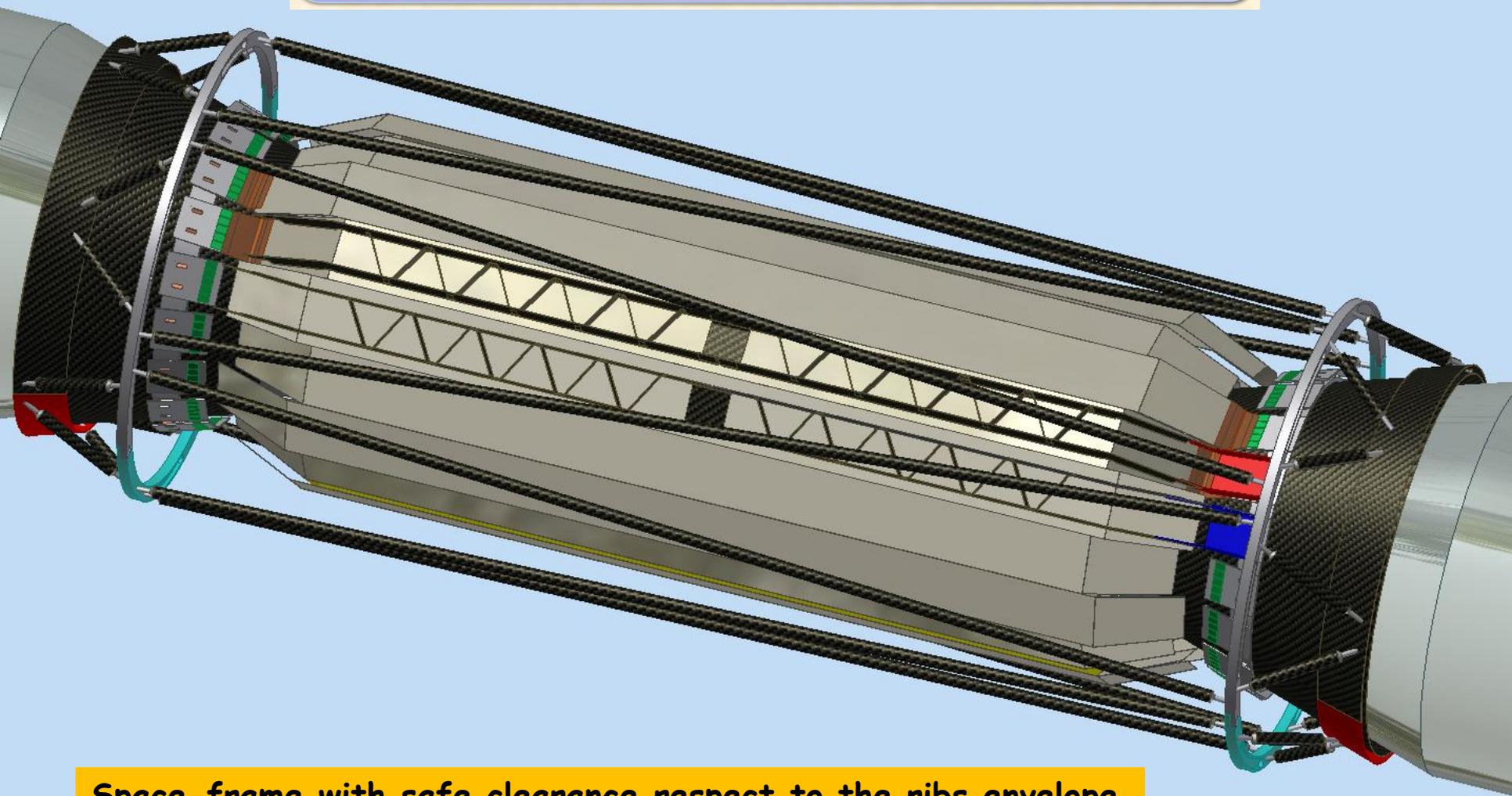
Modelling ribs for Layer 1-5

SuperB modules longer to respect Babar !

For L5B modules about **230 mm extra length** respect Babar dimensions !

(L5B module total length about 760mm).

- Need ribs more height and a reinforced profile along the barrel sensors to be more rigid about flexion /torsion forces.
 - Snake Ribbon or lighter Reticular Structure to add to the ribs:
 - Snake Ribbon: N.1 equivalent rib/module
 - Reticular Structures: N.1/3 equivalent rib/module
- Work in progress for final dimensioning (thermo-structural simulation)



Space-frame with safe clearance respect to the ribs envelope

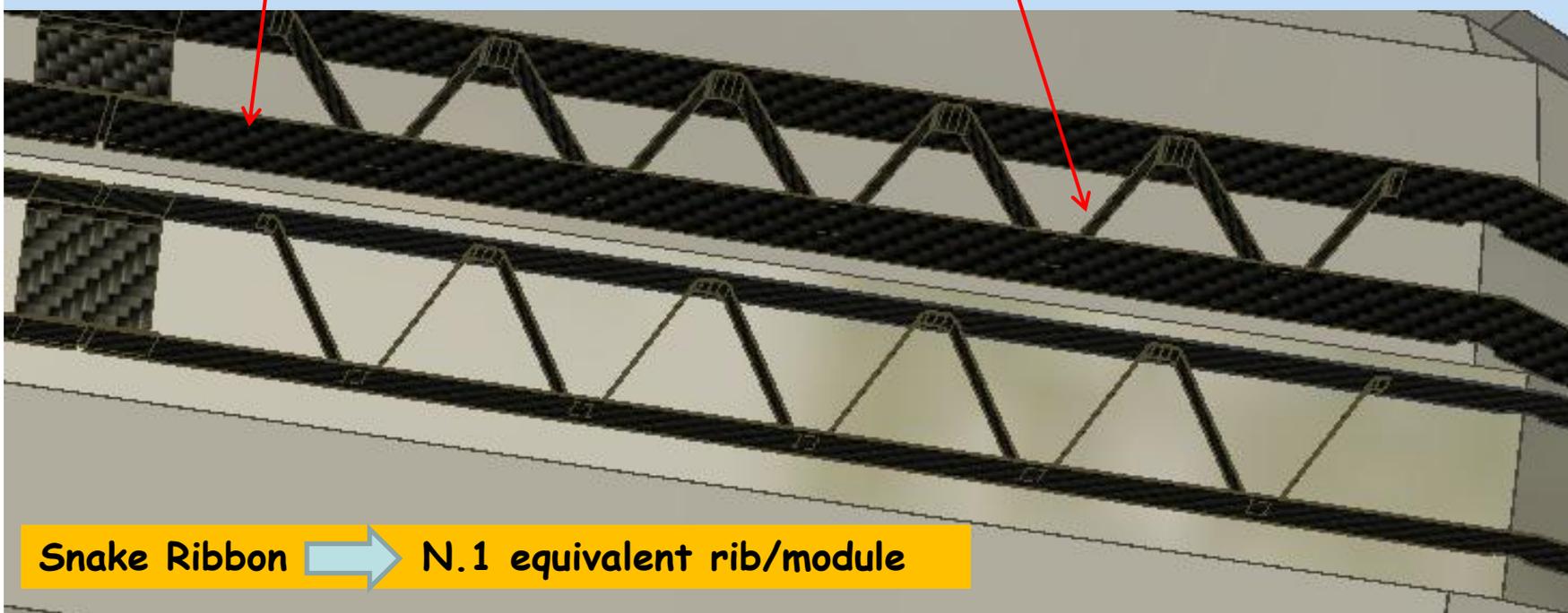
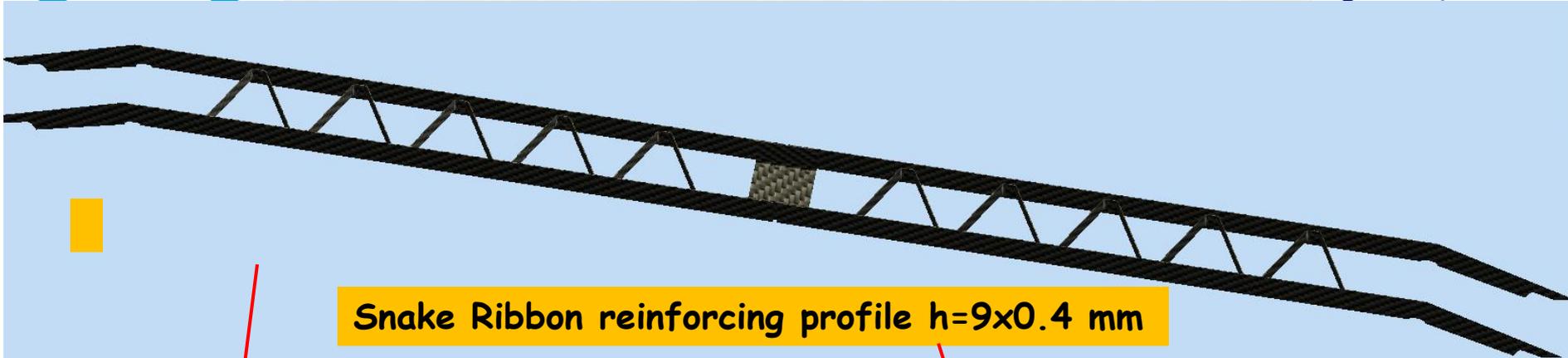


RIBS L4/5

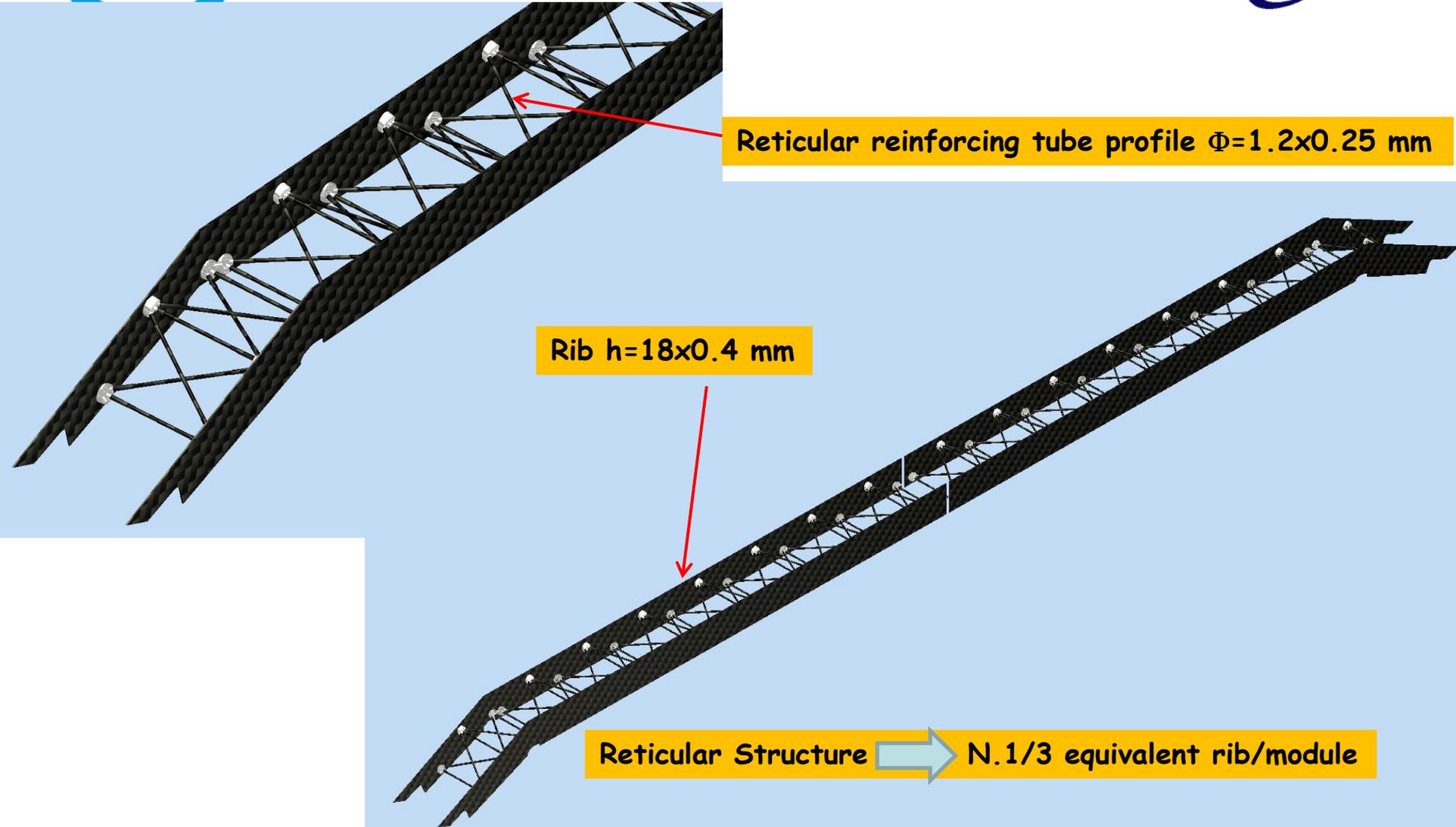


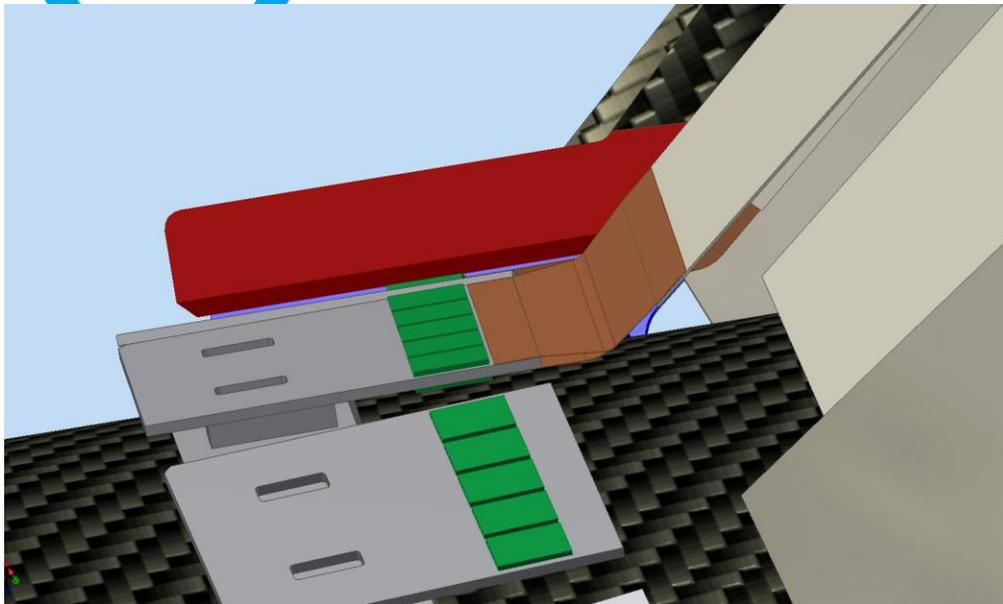
Space-frame/supporting cones and L5



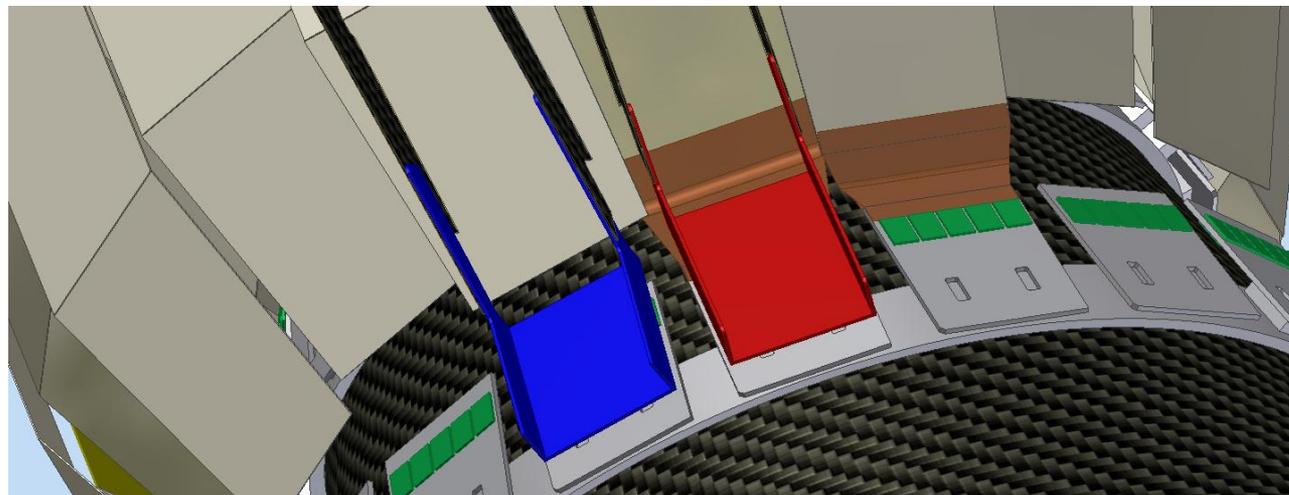


RIBS L4/5



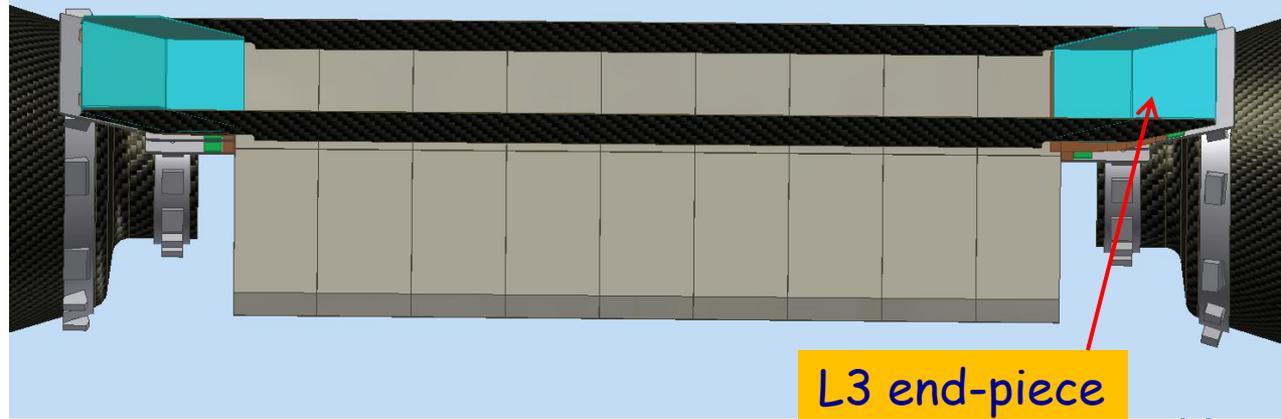


No particular problem to model fanout layer 4-5



SVT L1-5 design

Modelling ribs - fanouts for L1-2-3





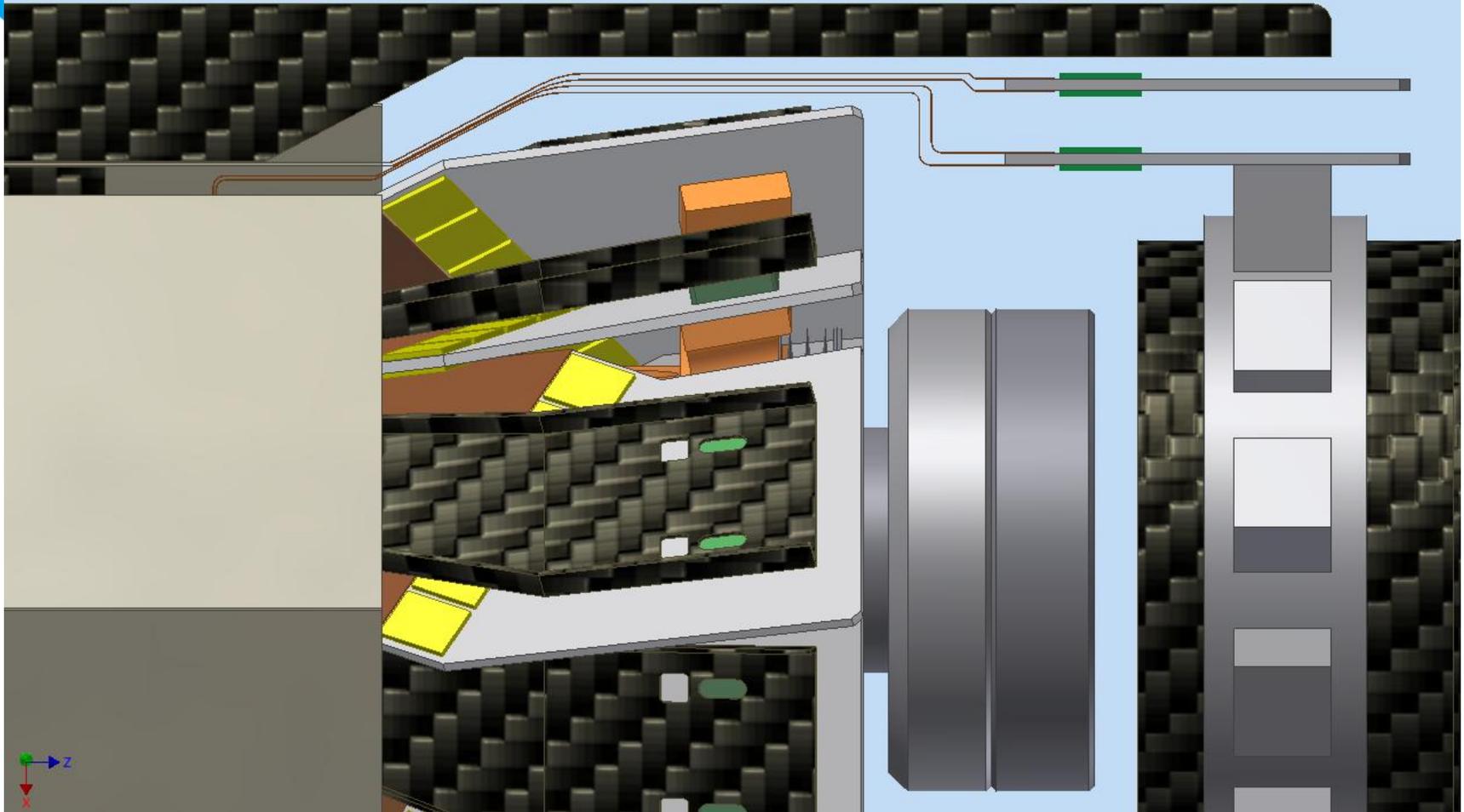
SVT L1-5 Layout design

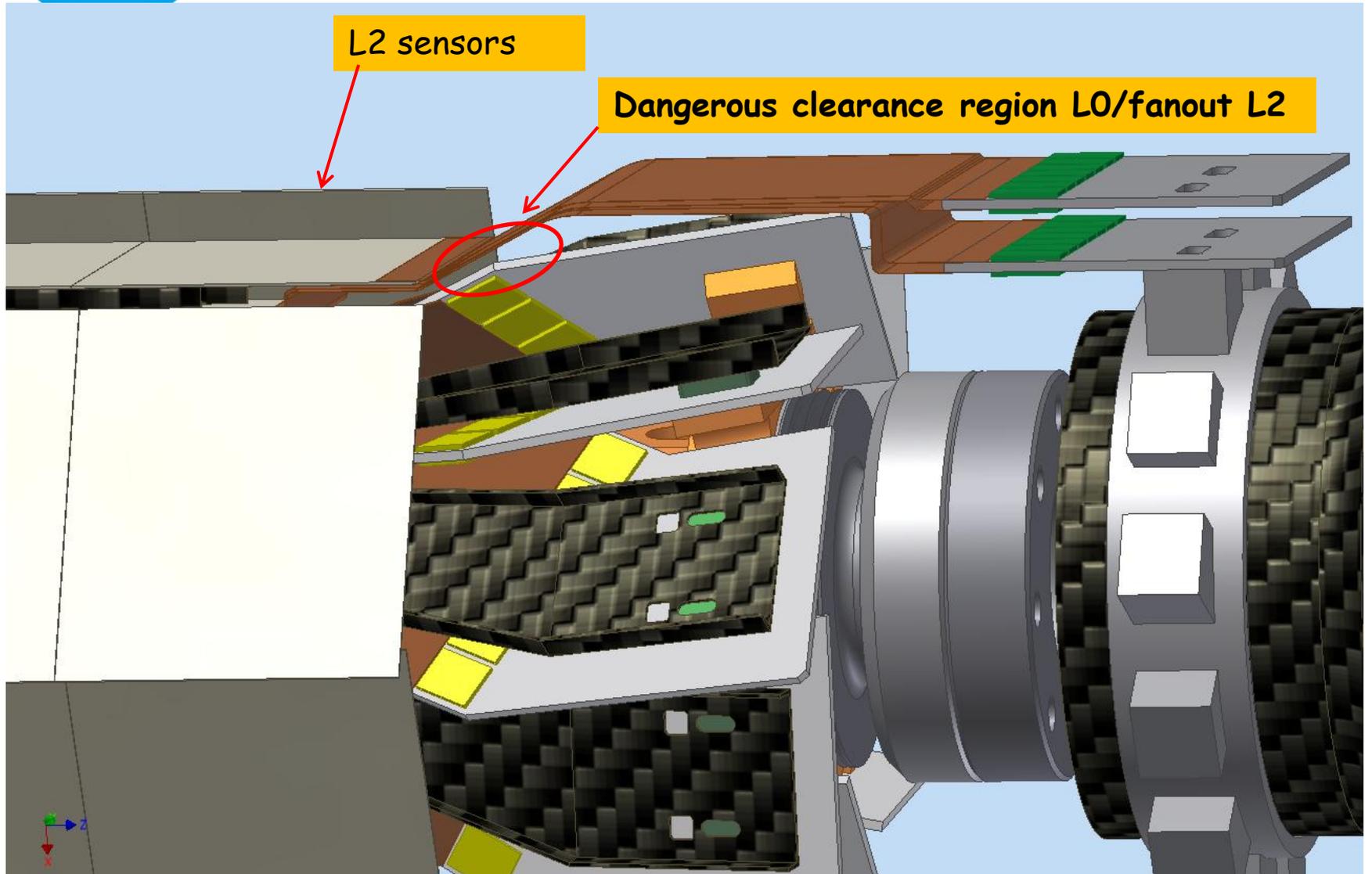


Modelling ribs -fanouts for L1-2 in update models

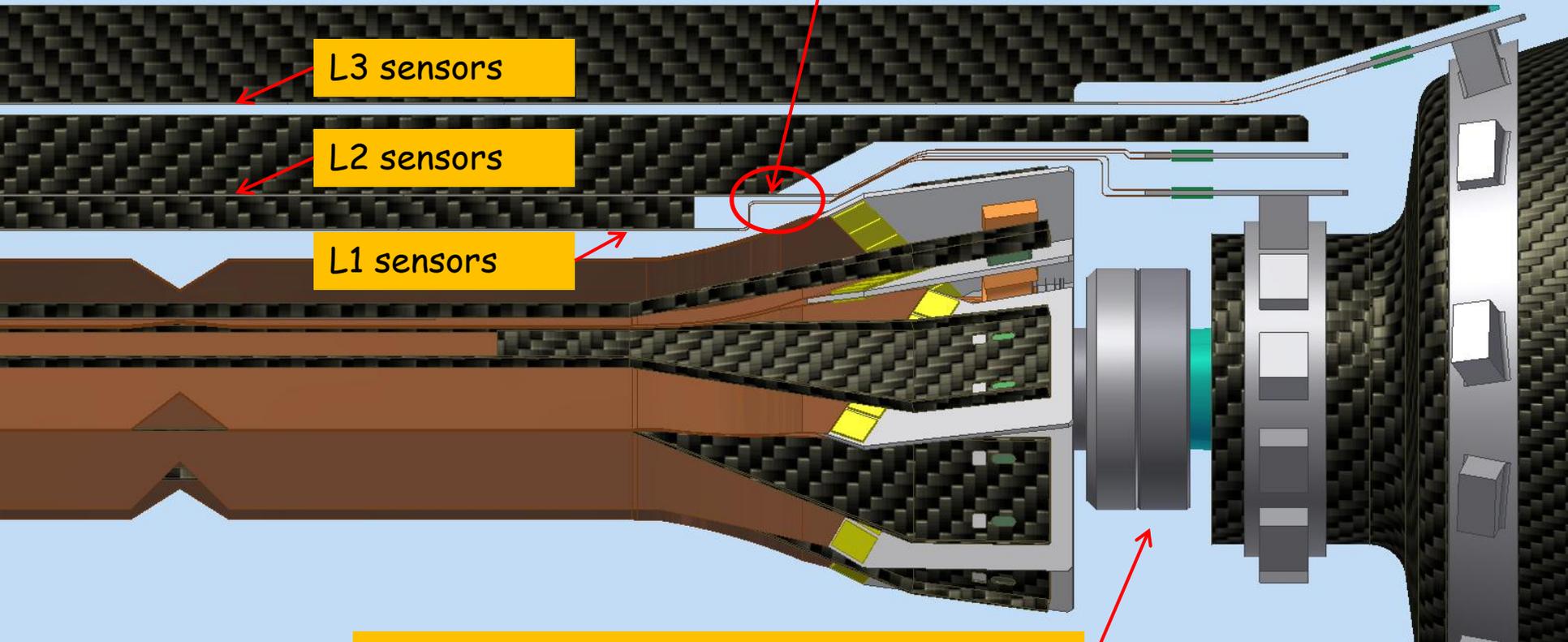
1) L1-2-L3 fanouts shape very peculiar :

- They need to round around the LO Hybrid : ribs will be used like constrain to hold the fanout on the right shape
- Region very crowded with small clearance between components!





Fanout fixed to the ribs to hold the right shape



L3 sensors

L2 sensors

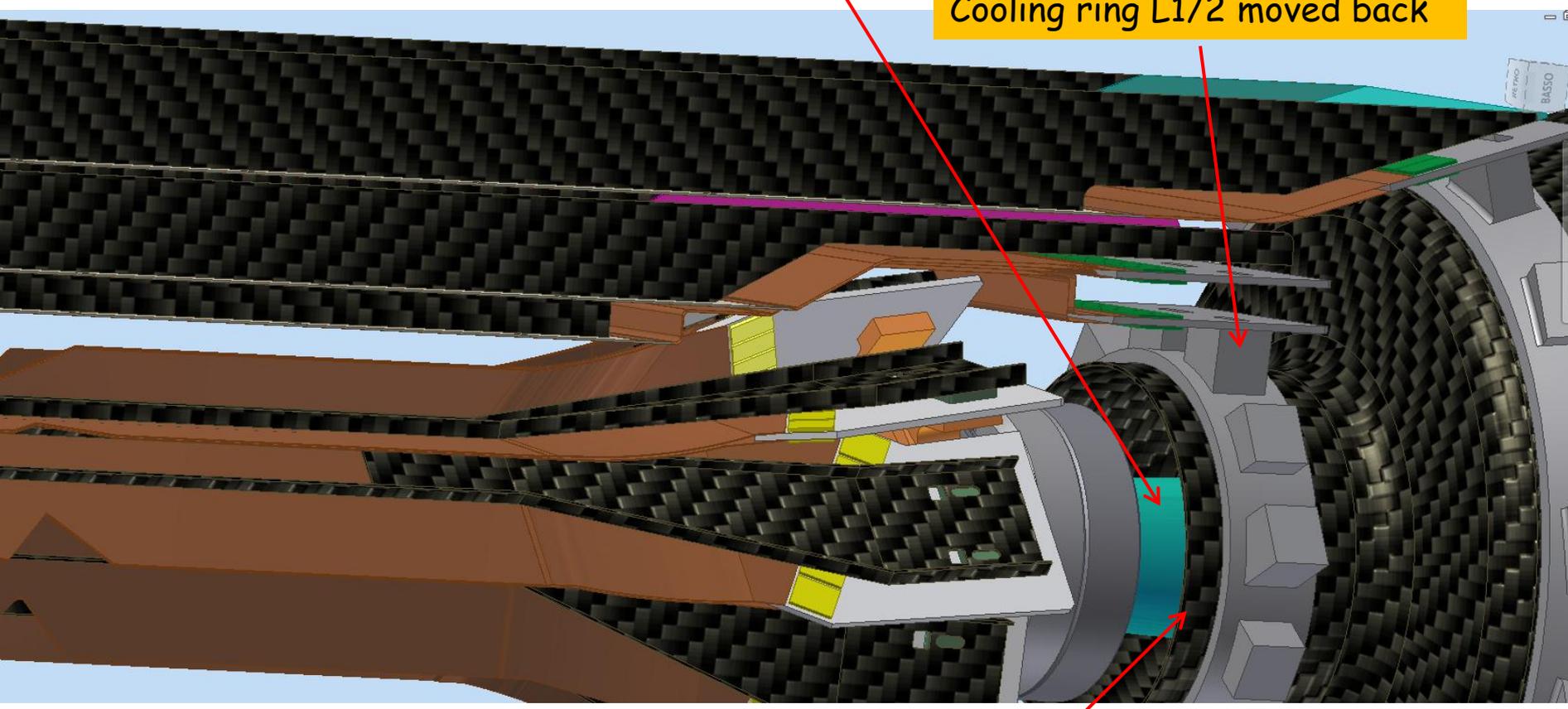
L1 sensors

New Be beam-pipe flanges position: very useful for the nregion clearance

L0/L1-2-3 fanouts routing

Cryostat beam-pipe

Cooling ring L1/2 moved back



Reduced semicone nose C.F. support

Reduction of the nose of C.F. semicone to allow easier passing of the L0 cables towards the transition Card

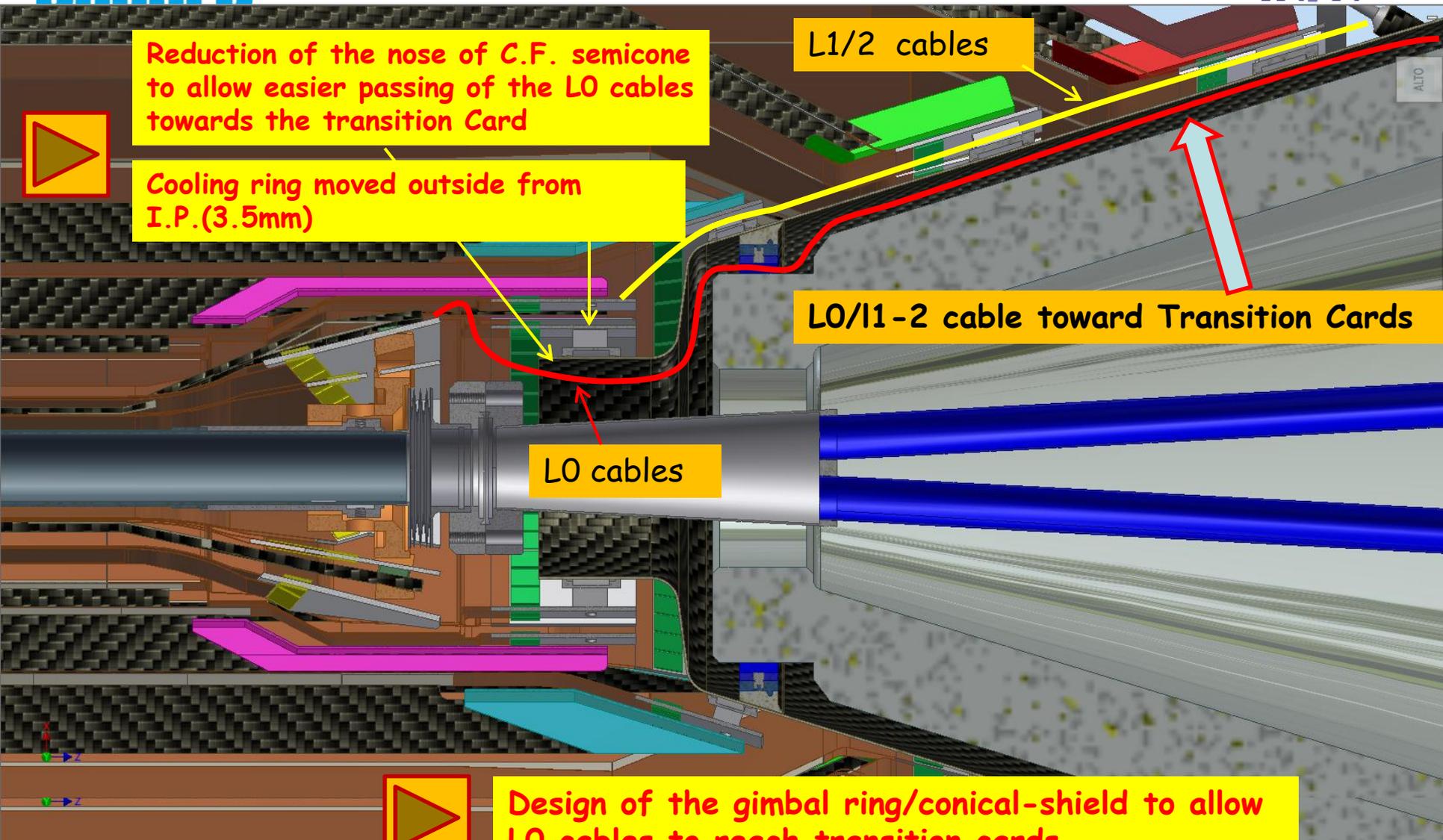
Cooling ring moved outside from I.P. (3.5mm)

L1/2 cables

L0/L1-2 cable toward Transition Cards

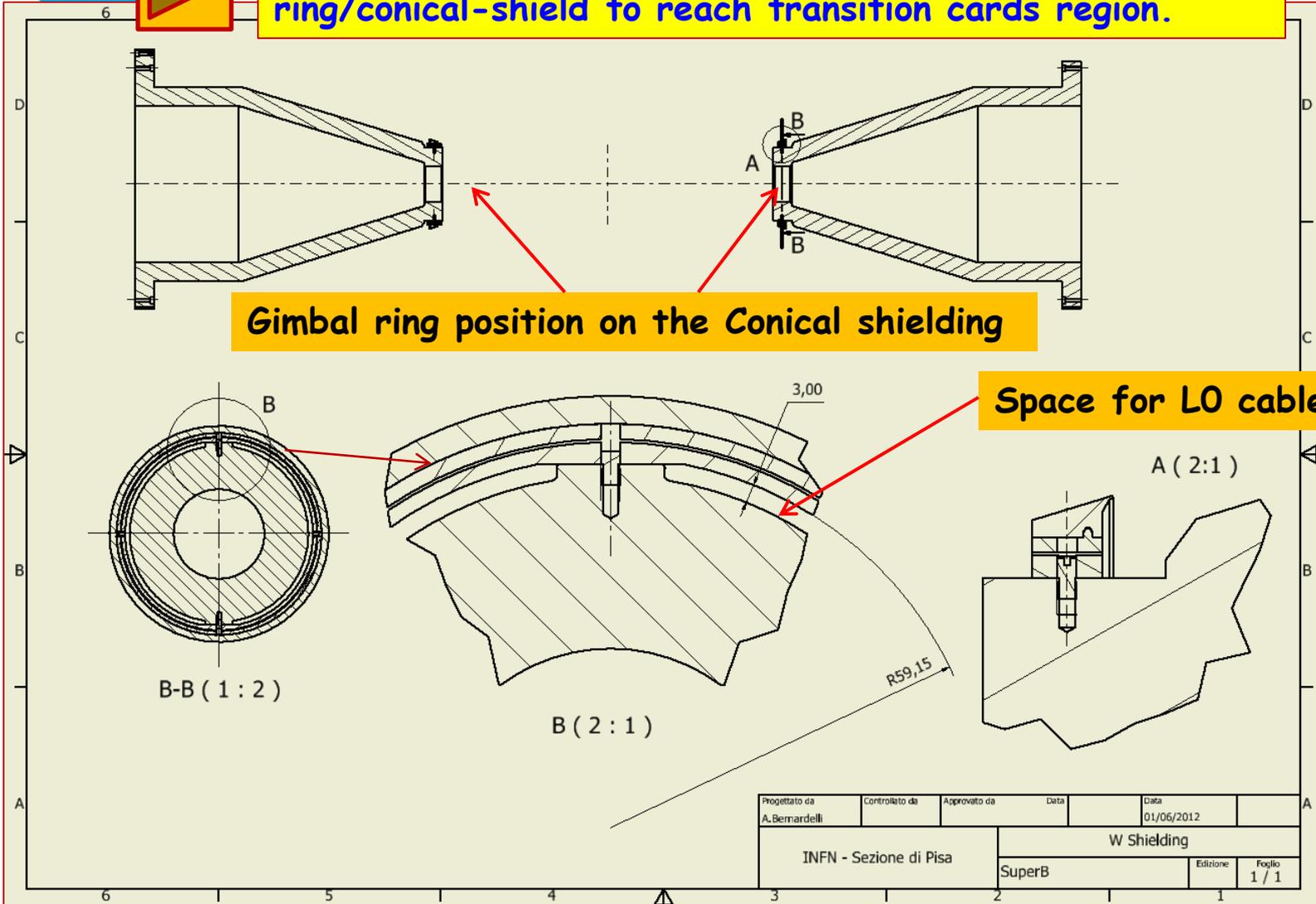
L0 cables

Design of the gimbal ring/conical-shield to allow L0 cables to reach transition cards





Design to allow L0 cables to pass-through gimbal-ring/conical-shield to reach transition cards region.

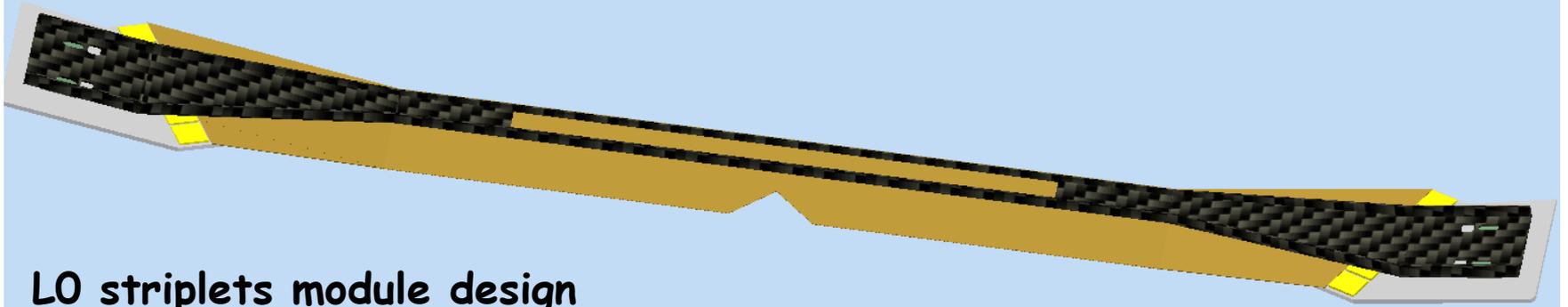


Gimbal ring position on the Conical shielding

Space for L0 cables

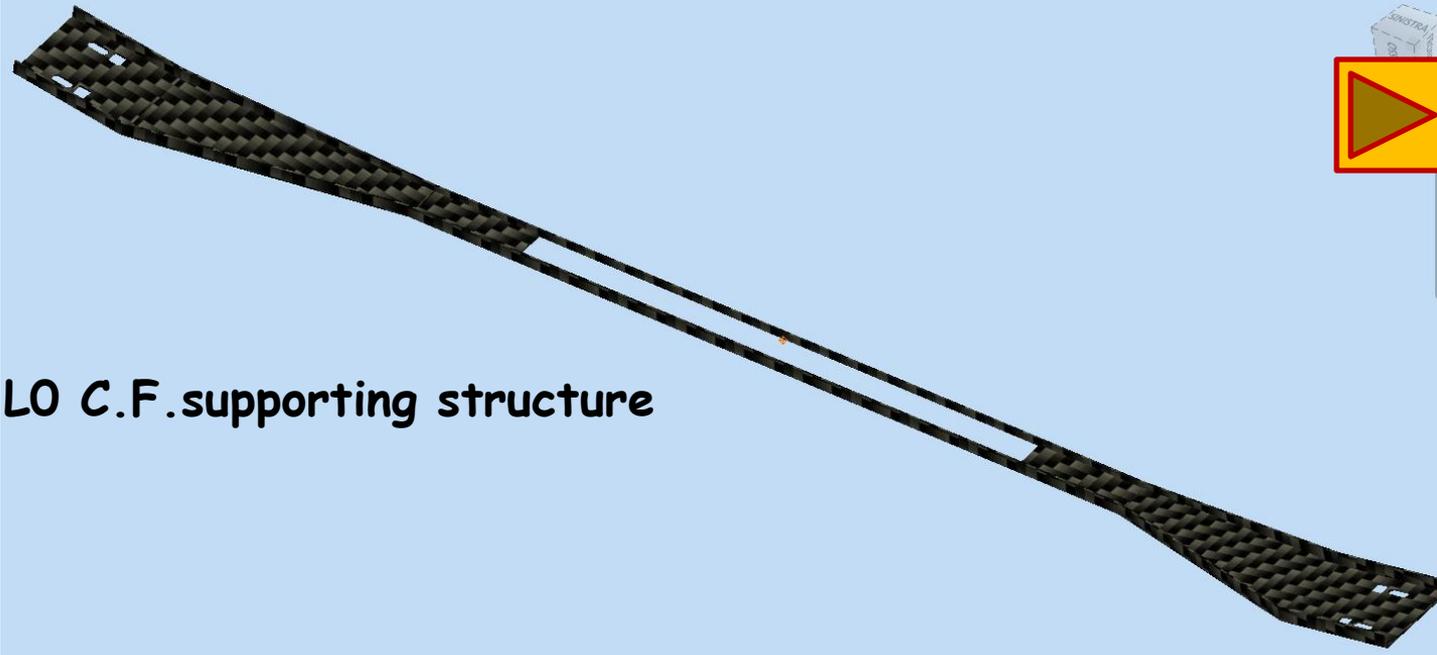
Progettato da A. Bernardelli	Controllato da	Approvato da	Data	Data 01/06/2012
INFN - Sezione di Pisa		W Shielding		
SuperB		Edizione	Foglio 1 / 1	

Striplets L0 module design



L0 striplets module design

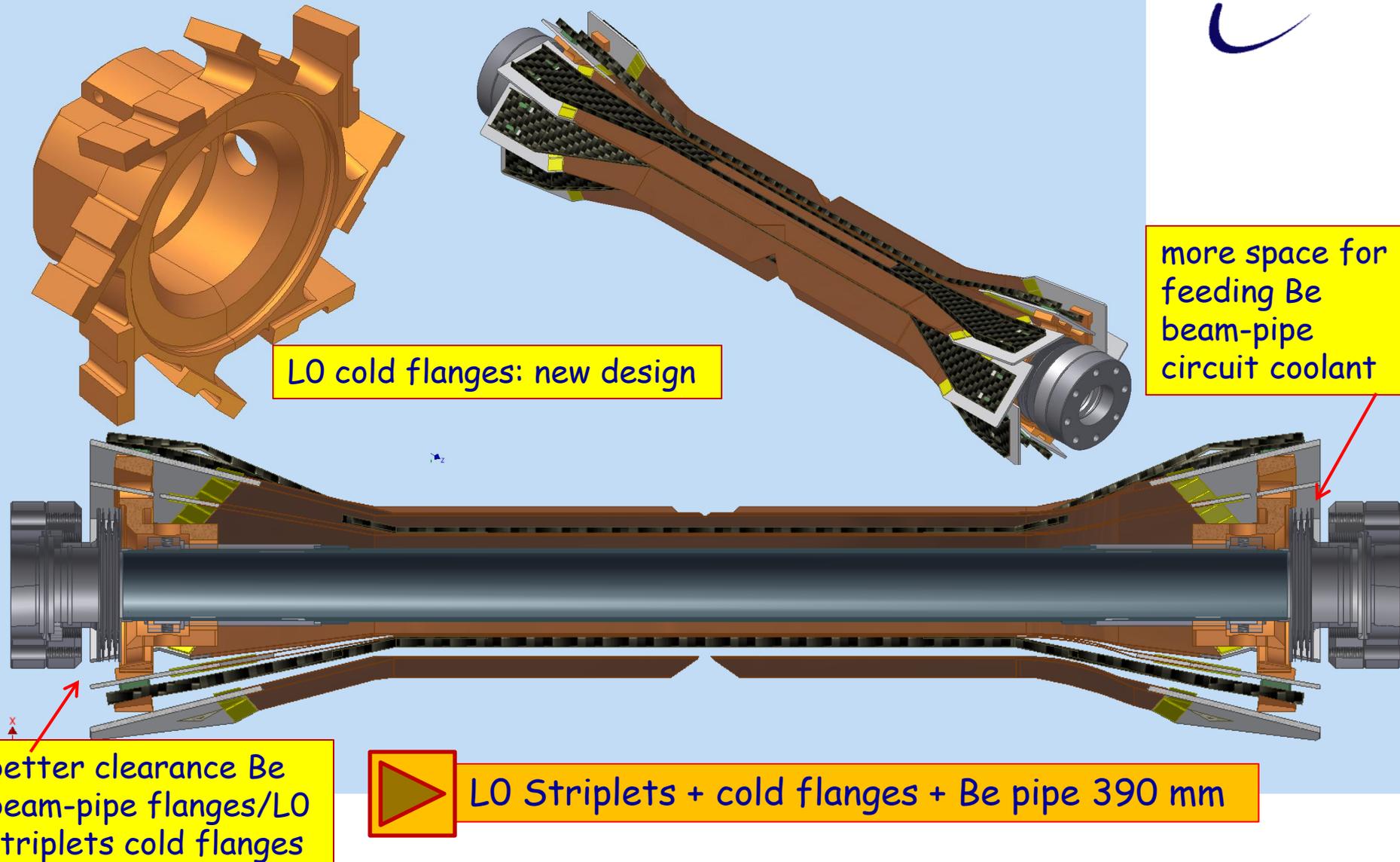
HDI/kapton inclined 10°



L0 C.F. supporting structure

Si sensor :
W=15.1 mm
L=104 mm
R=15.1 mm

Be beam-pipe



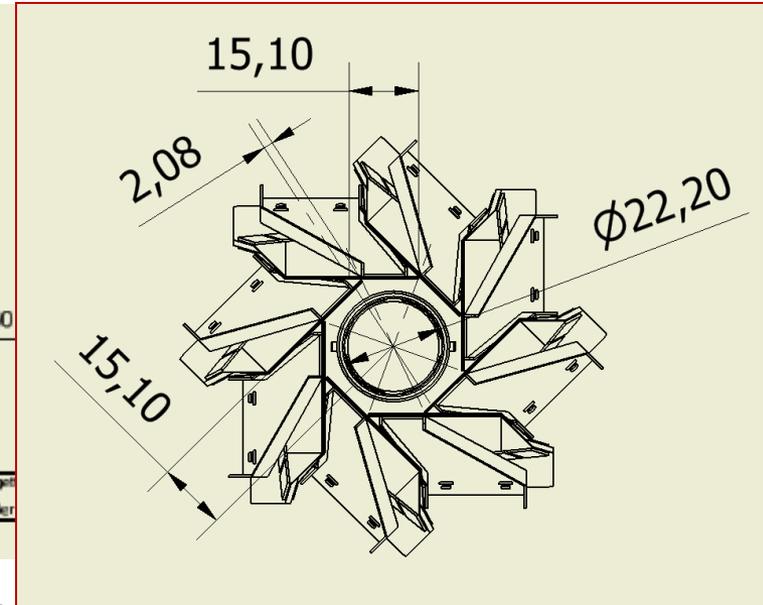
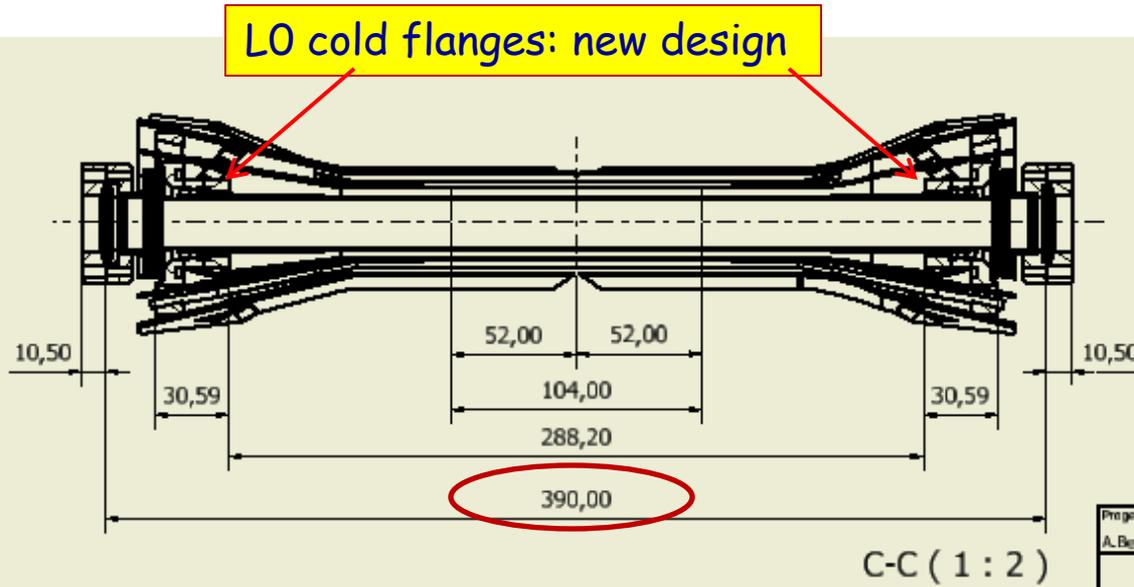
LO cold flanges: new design

more space for feeding Be beam-pipe circuit coolant

better clearance Be beam-pipe flanges/LO striplets cold flanges

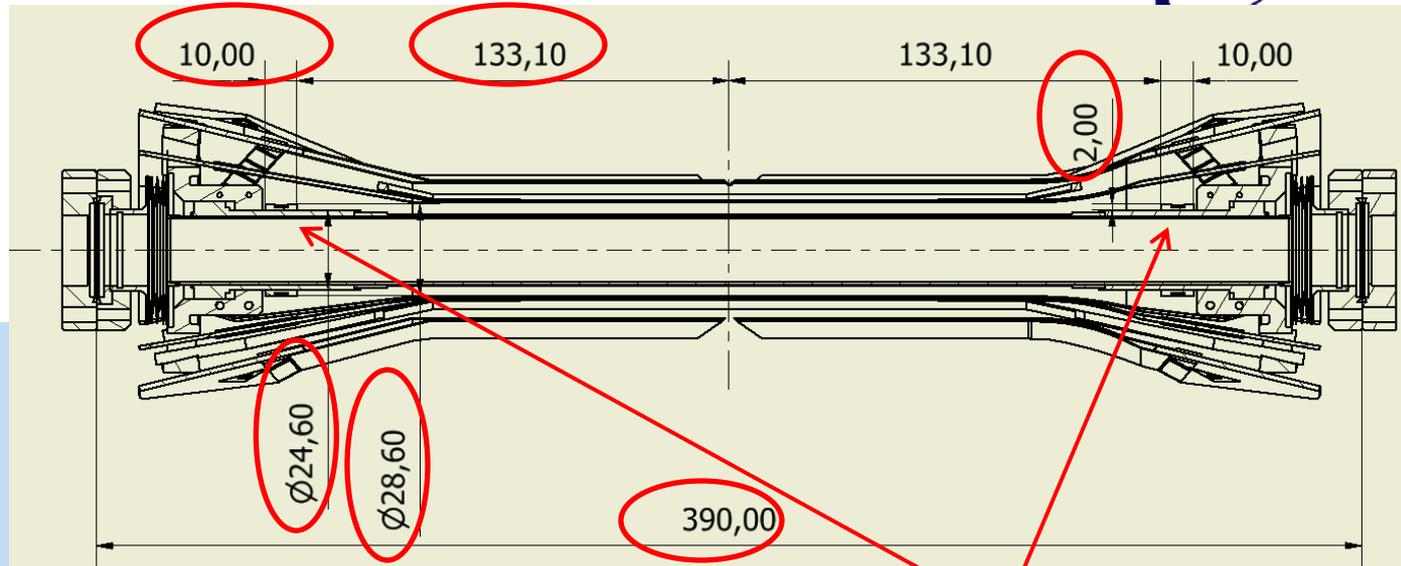
▶ LO Striplets + cold flanges + Be pipe 390 mm

L0 cold flanges: new design

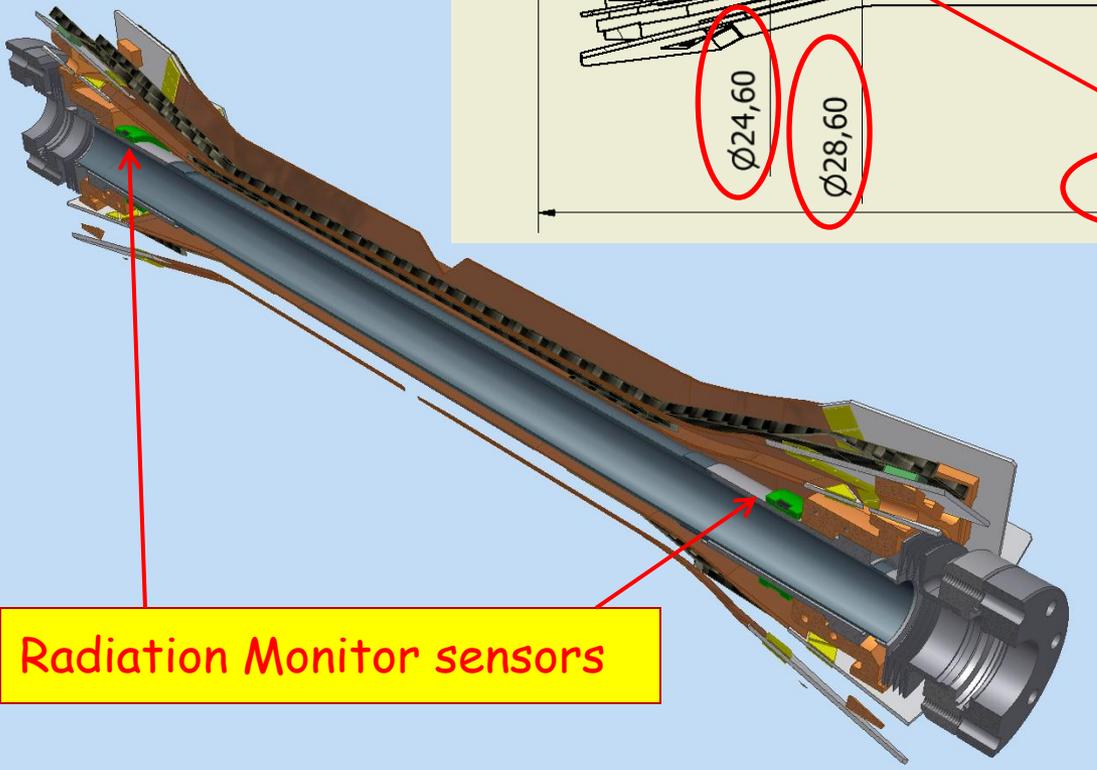


L0 Striplets + cold flanges + Be pipe 390 mm

Radiation Monitor

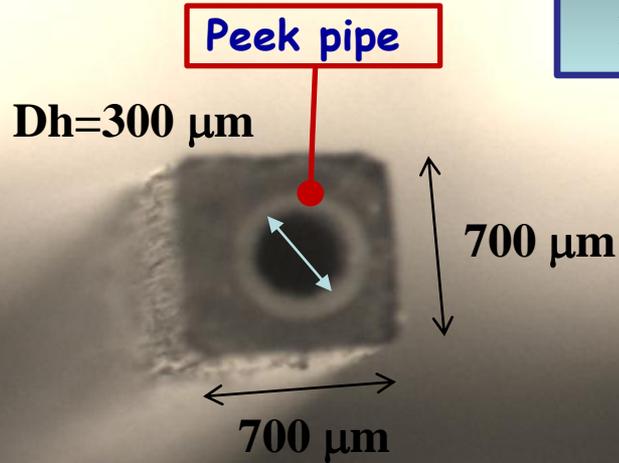


Radiation Monitor fixed on Be pipe L=390mm

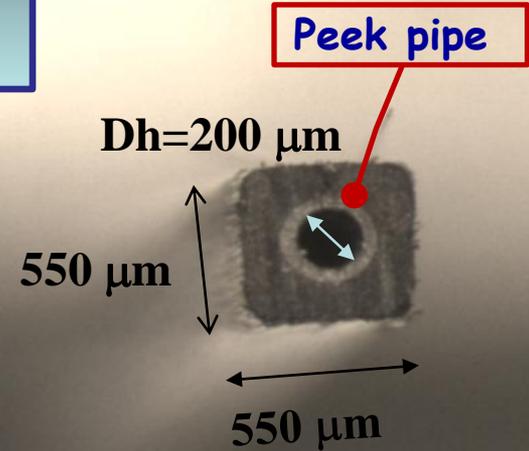


Radiation Monitor sensors

Further Miniaturization
microtube technology



Old Carbon Fiber Pultrusion



New Carbon Fiber Pultrusion

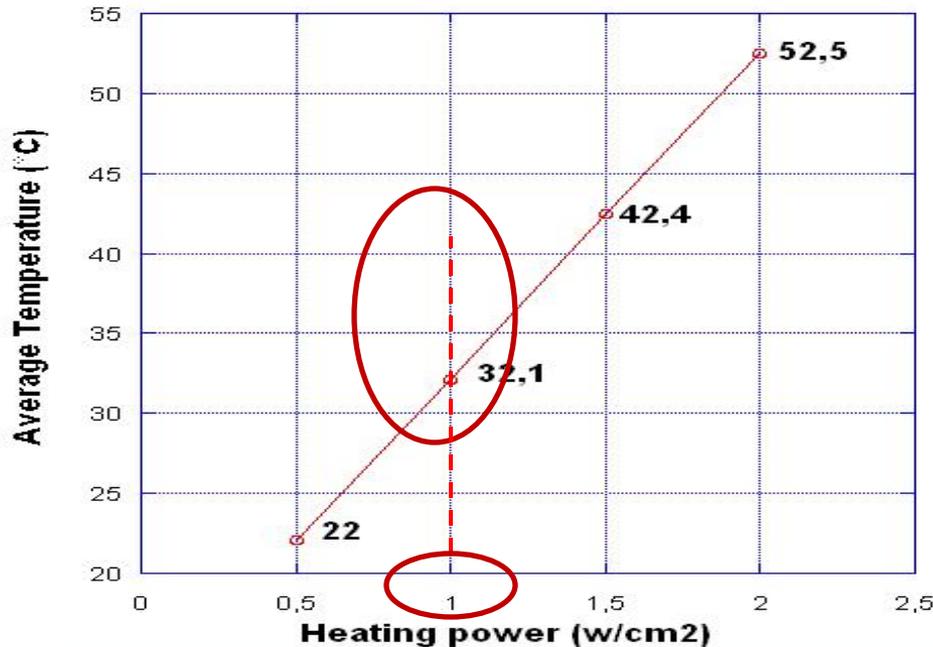
Full Module $X = 0,28 \% X_0$
Net Module $X = 0,15 \% X_0$



Full Module $X = 0,22 \% X_0$
Net Module $X = 0,11 \% X_0$

Microchannel support 550 μm th, Full and Net version tested at the TFD lab

MC-550-N-200#5

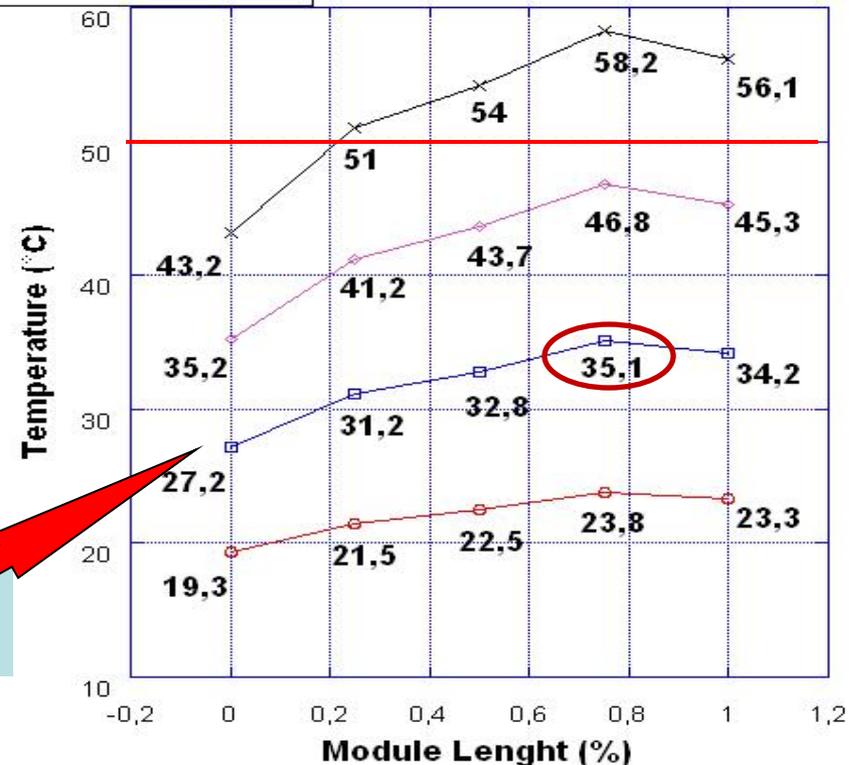


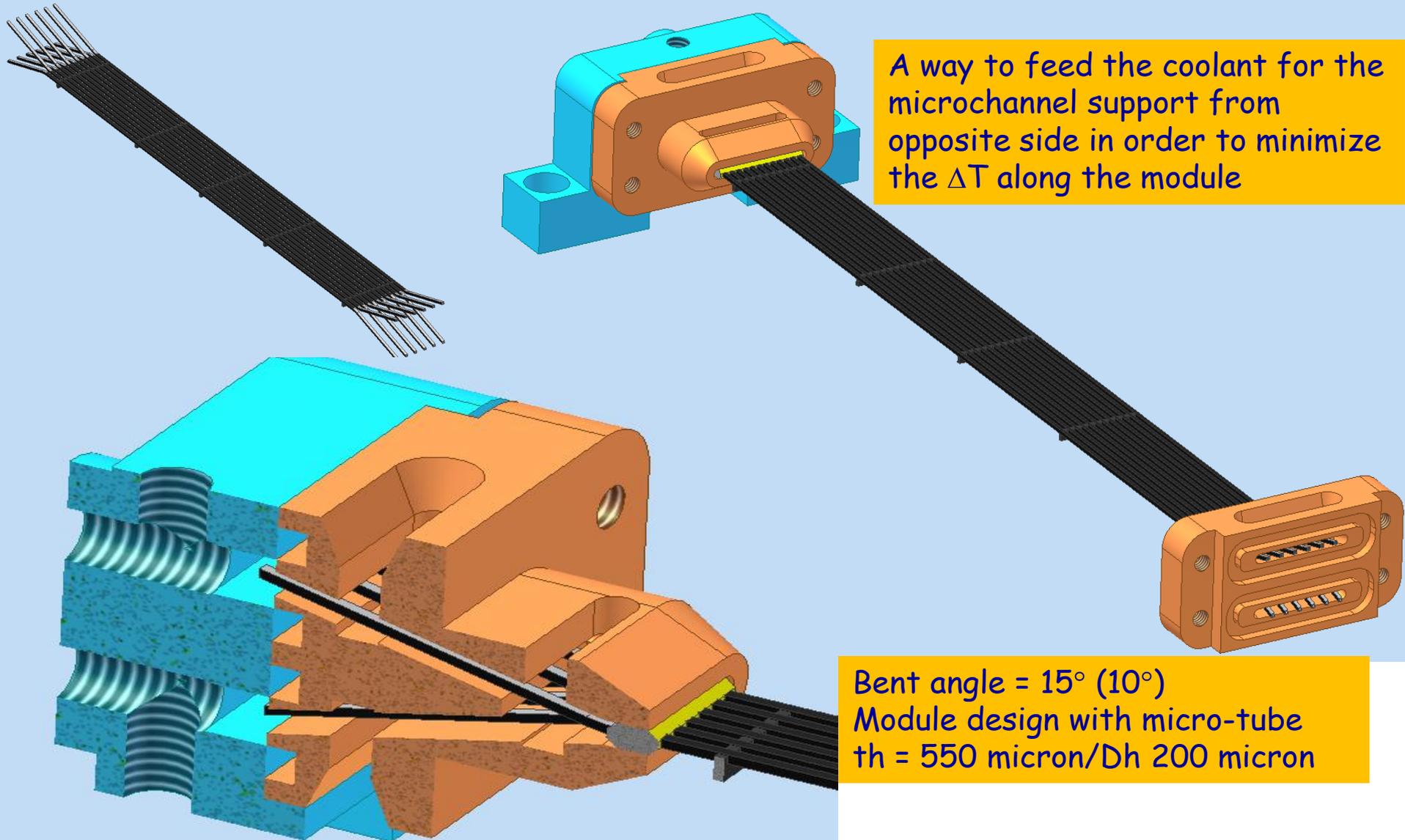
Average module Temperature vs Specific Power

Temperature along the module:
 $\Delta T = 7.9 \text{ }^\circ\text{C}$ at 1.0 W/cm^2 $\Delta p = 3.5 \text{ atm}$

Tests performed on net module sample (length = 120 mm) with water-glycol @ 10 °C as coolant ($\Delta p = 3,5 \text{ atm}$).

MC-550-N-200#5 ($\Delta P = 3,5 \text{ bar}$)

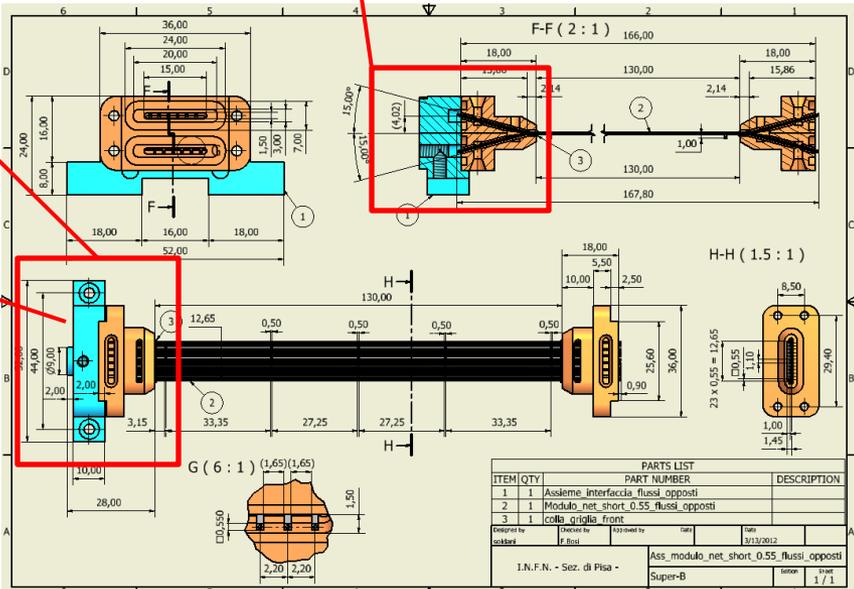
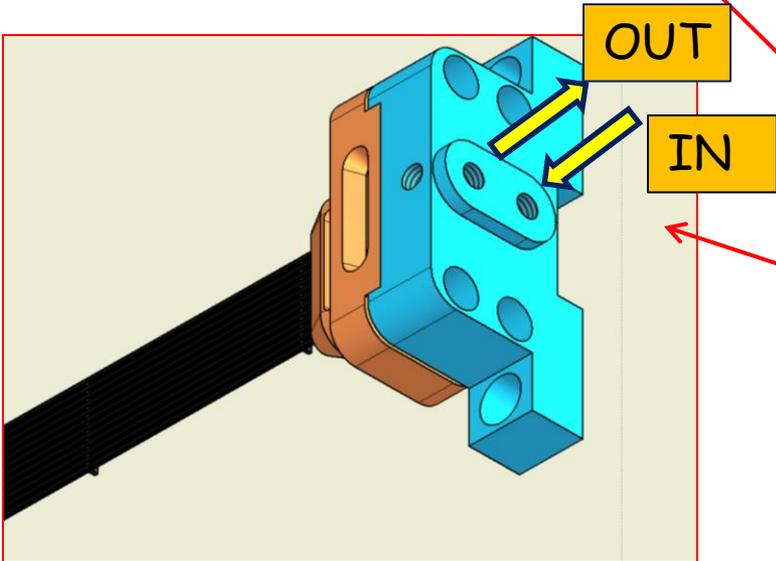
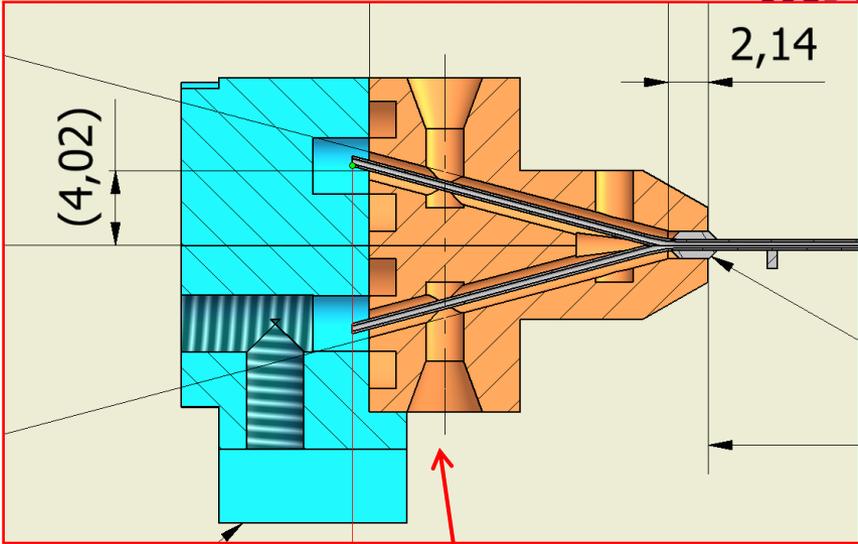
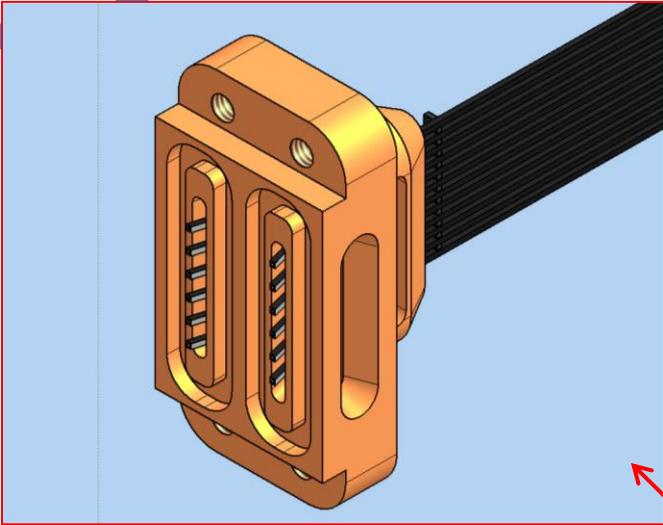




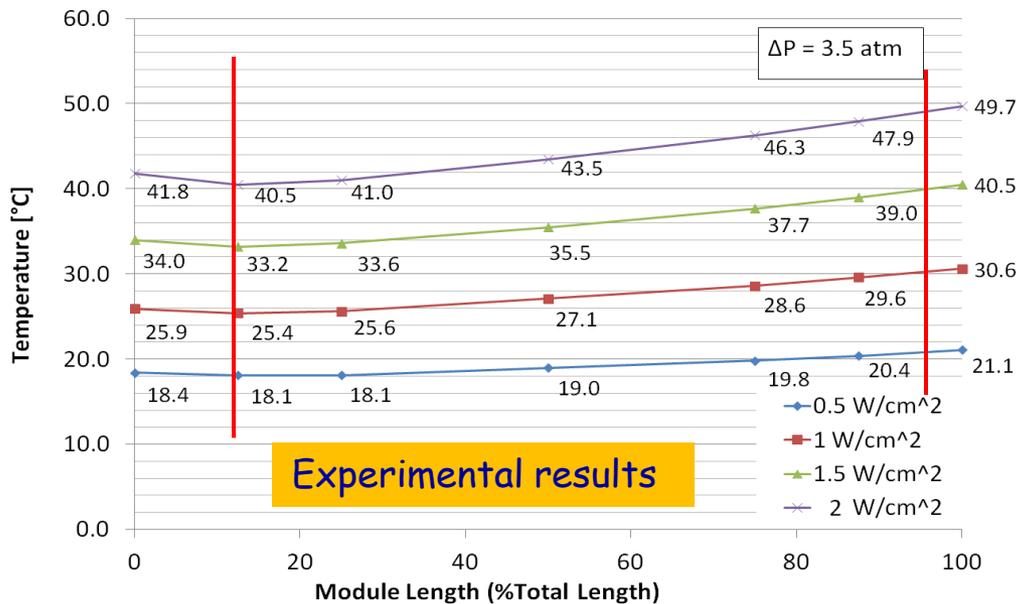
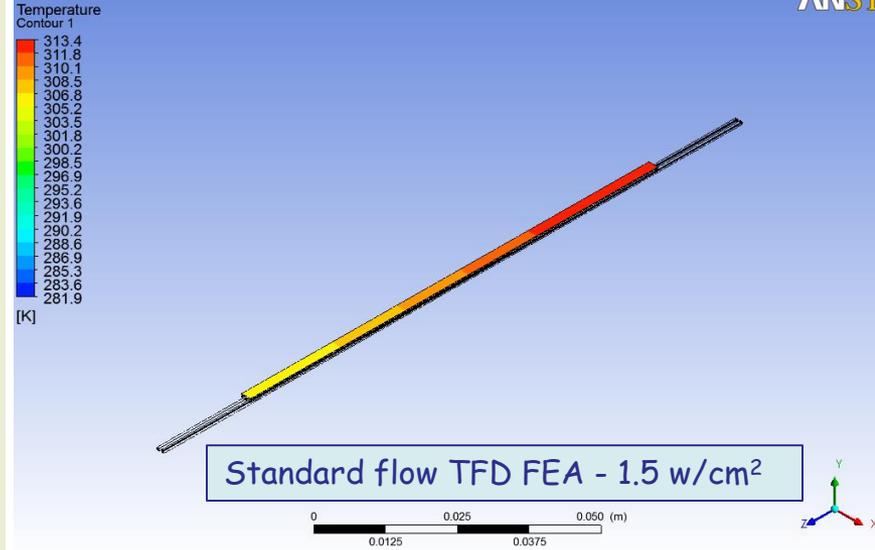
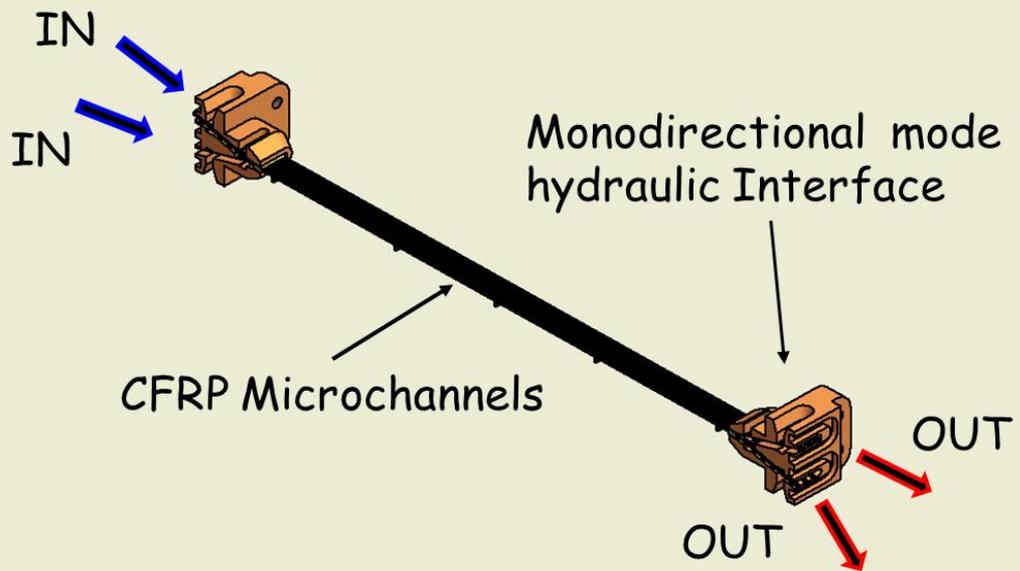
A way to feed the coolant for the microchannel support from opposite side in order to minimize the ΔT along the module

Bent angle = 15° (10°)
 Module design with micro-tube
 th = 550 micron/Dh 200 micron

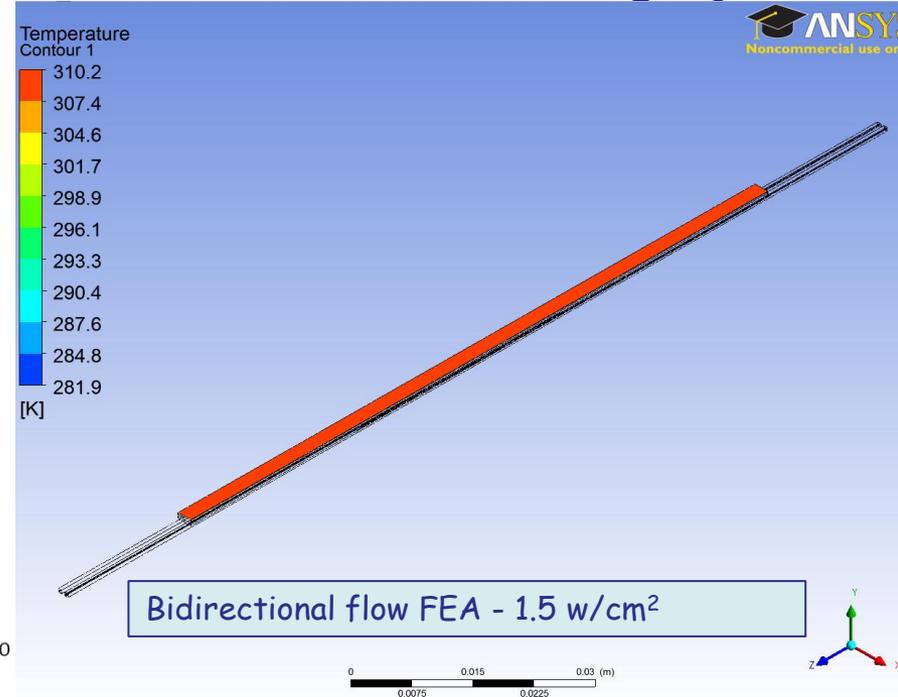
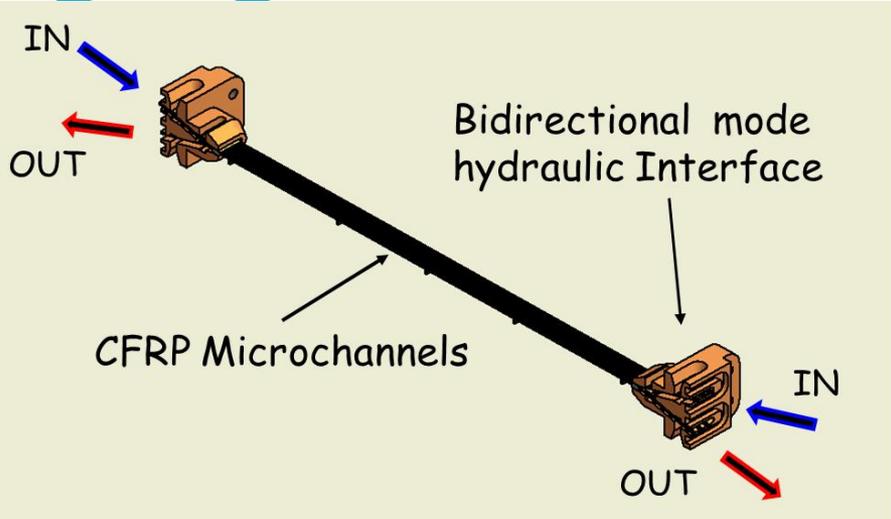
Opposite flow bent module



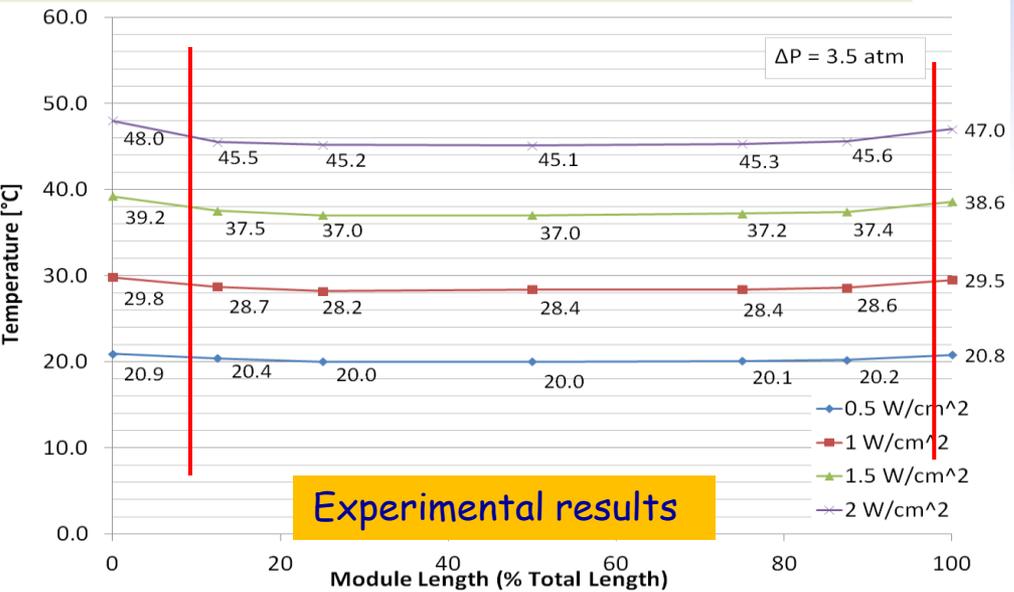
Monodirectional flow module

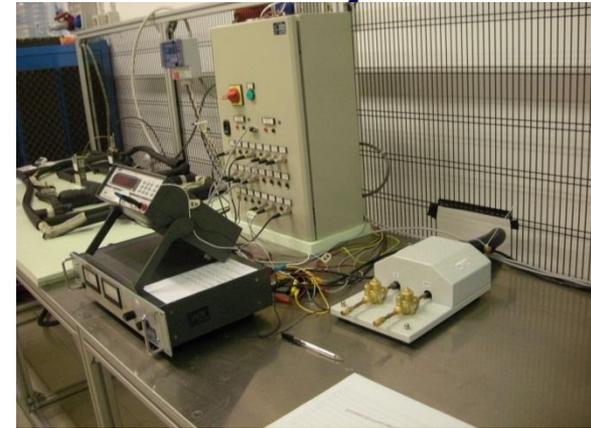
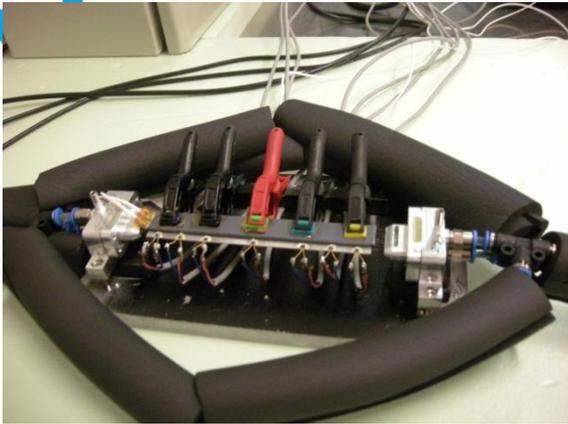


• Concordance results
FEA study/test at TFD lab



• Concordance results
FEA study/test at TFD lab





-Micro-channel support prototypes match the Layer 0 pixel detector requirements on material thickness (X_0). Efficient heat evacuation has been achieved by micro-channel technology through liquid forced convection.

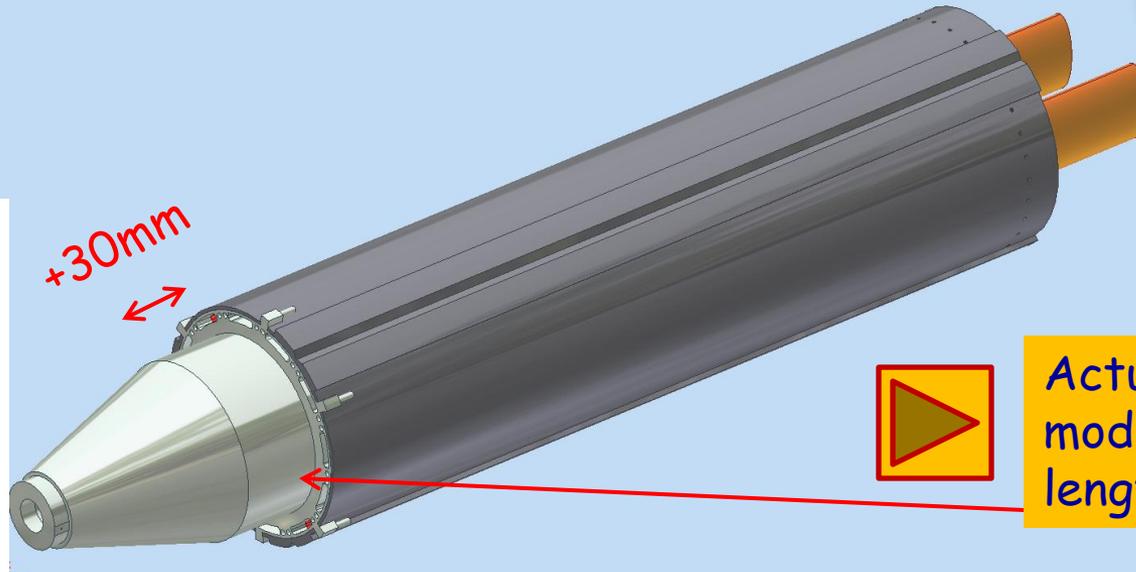
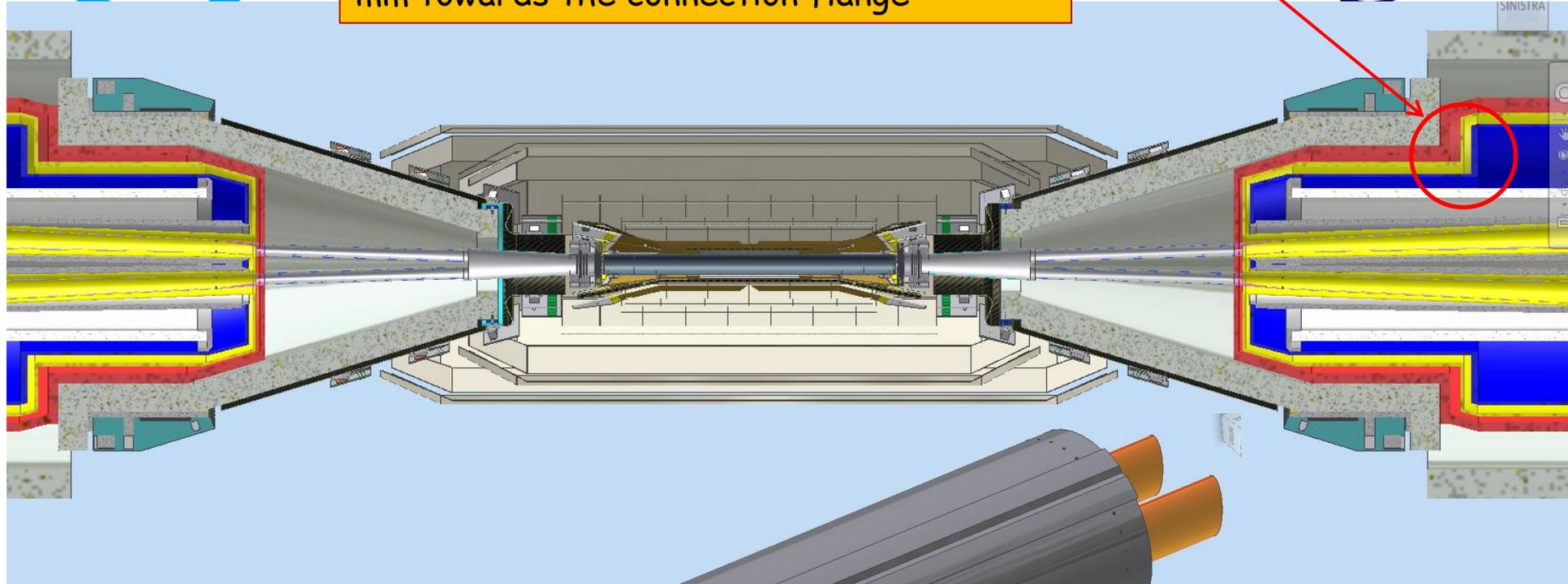
- The experimental results show that Net Module $th=550\mu m$ and $Dh=200\mu m$ is able to cool sensors with a power density up to $1.5 W/cm^2$ with a X_0 value of 0.11% (mechanical support/cooling budget) and keep the sensor below $50^\circ C$; with bidirectional coolant flow mode, it is possible to reduce the ΔT along the sensor below $2^\circ C$.

- Thermo-structural simulation and experimental test are still needed to check geometry sensor variations at the operational cooling conditions. Endurance cooling test are also in planning.

-Next step : proceed to the engineering of the hydraulic interface and the support flanges manifolds according to the experimental layout condition.

Transition Card

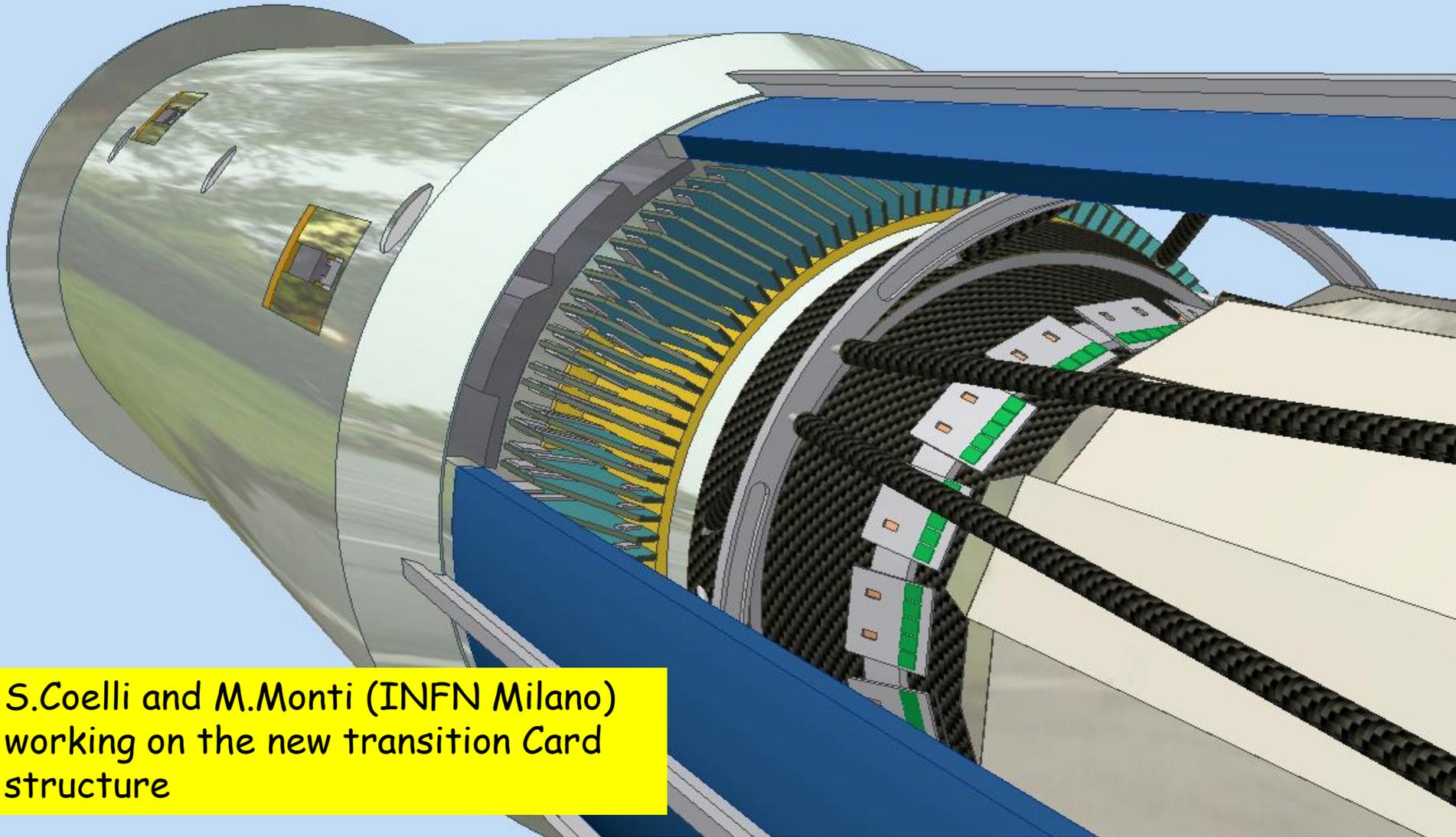
- Transition Card board 30 mm longer 30 mm towards the connection flange



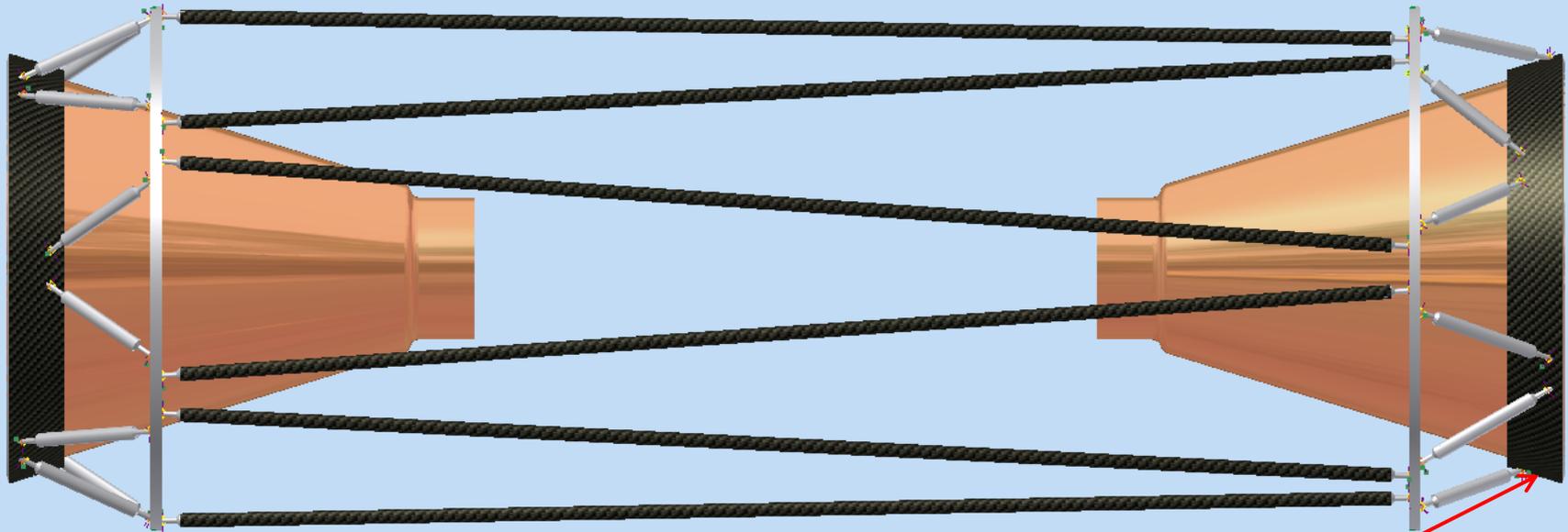
+30mm



Actual Criostat design modified with extra length!



S.Coelli and M.Monti (INFN Milano)
working on the new transition Card
structure



J. Morris and F. Gannaway are working on the technological aspect of C.F. Space-frame flanges :

- sandwich structure/full C.F. section
- choice of the best C.F tube
-

Space Frame, version 2



I.R. Architecture/quick demounting



- Present I.R. design has the goal to assume W conical shield independent from cylindrical shield to move less mass for quick demounting operation (all SVT components have minor diameter respect to W conical shield int.diam.) .

-In this configuration, criostat forw/back+SVT+LO+Be pipe+conical shield forw/back are one body (like in BaBar) but, in SuperB, to gain in X_0 , is not present the C.F. BaBar supporting tube and the Be pipe and SVT are the weak part of the mechanical chain .

-Quick demounting plans to insert-remove a temporary cage to make rigid SVT /Be pipe during sliding operation to replace LO in short time.

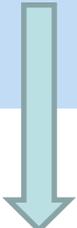
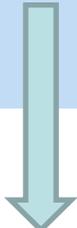
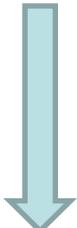
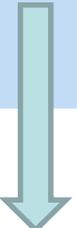
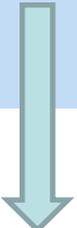
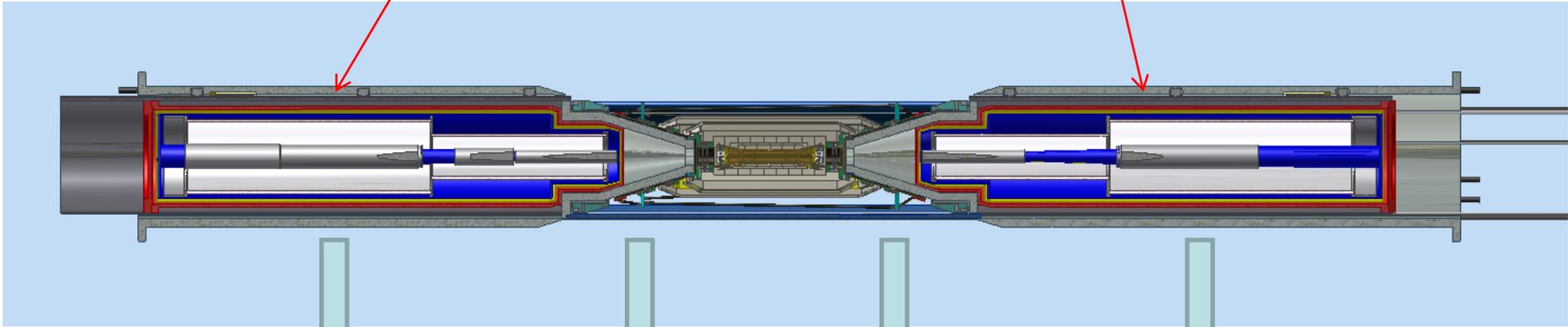
-Has been asked to assume $R=255$ (+10 mm respect now) as ext interface of D.C. in order to have minimum radial space to design the mechanics of operation.

-The temporary cage should put together the two opposite W conical shield from a remote region (FCAL) previous blocking the external tube forw/back to the internal part of cylindrical W shield.

I.R. Assembly

Cylindrical
Shielding 1300 Kg

Cylindrical
Shielding 1300 Kg



$350\text{Kg} + 150\text{Kg} + 50\text{Kg} = 550\text{Kg}$
Cryostate+External
tube+Cables

200 Kg
Conical
Shielding

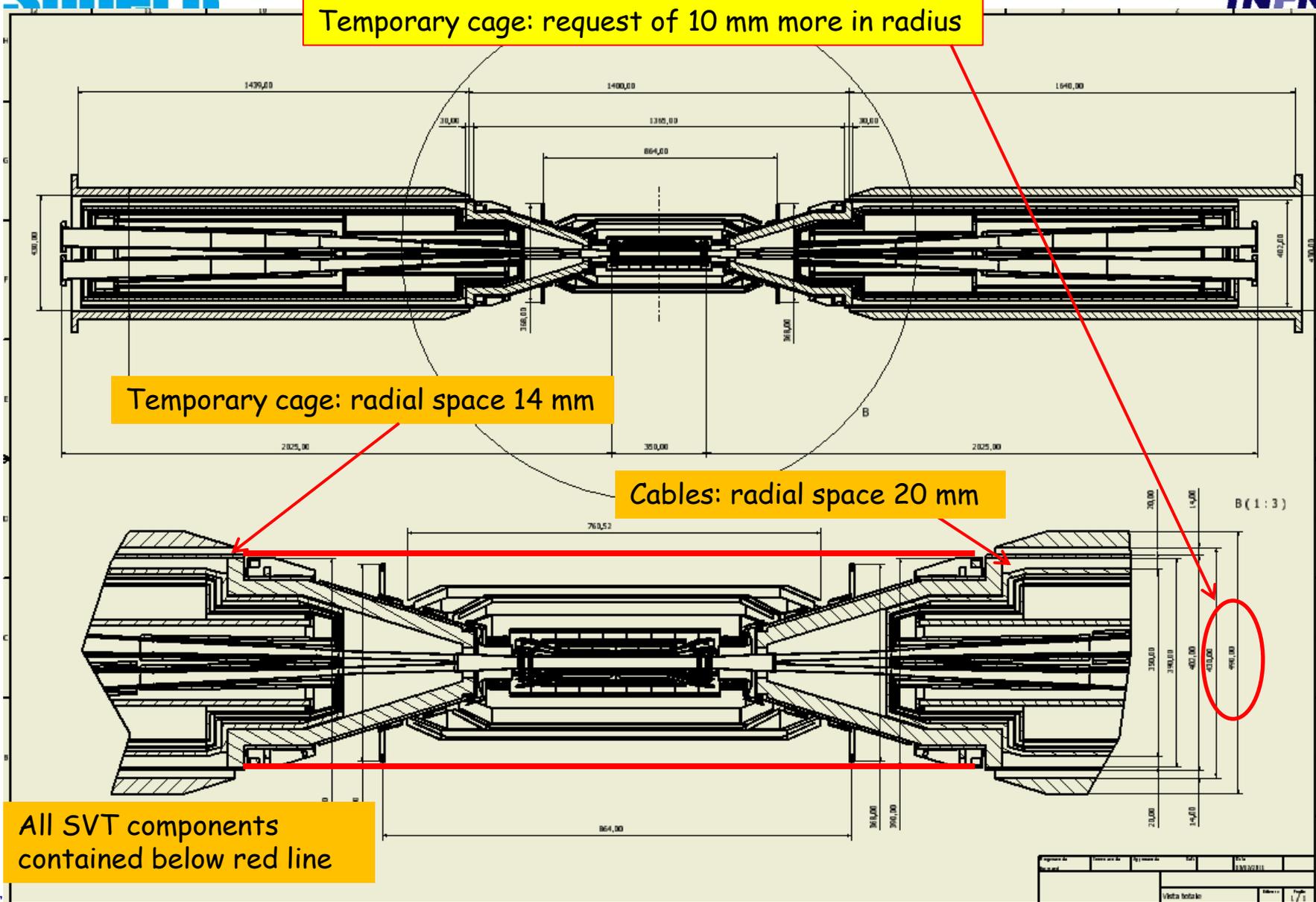
200 Kg
Conical
Shielding

$350\text{ Kg} + 150\text{ Kg} + 50\text{Kg} = 550\text{Kg}$
Cryostate+External tube+Cables

$50\text{ Kg} + 80\text{ Kg} = 100\text{Kg}$
SVT + Temporary Cage

Total weight to move for quick demounting $\approx 1630\text{ Kg}$

Quick demounting





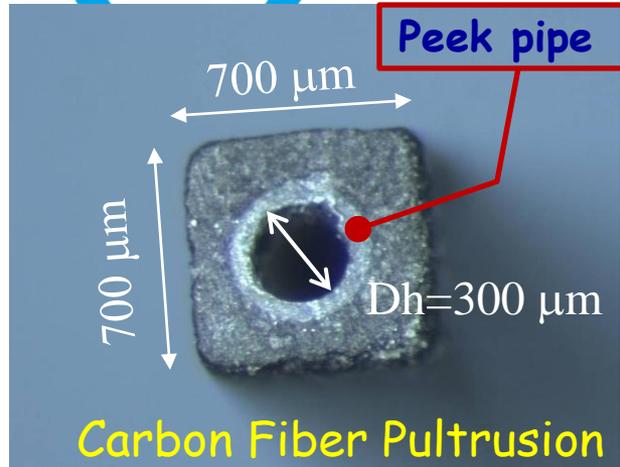
Conclusion



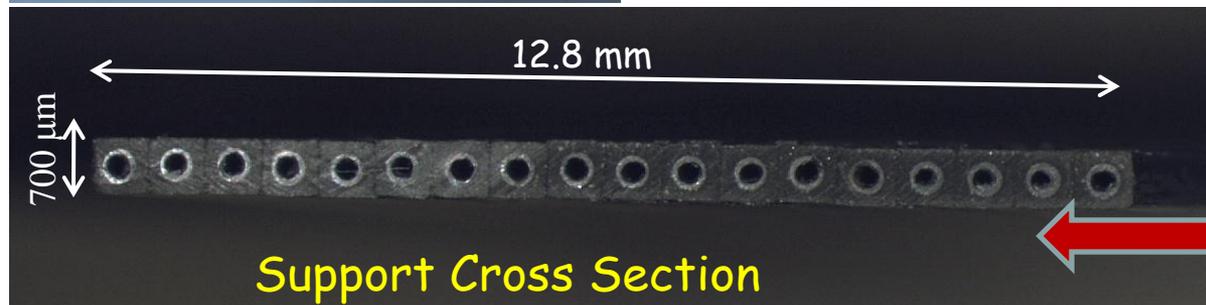
- 1) Some update realized on Ribs/fanouts SVT models.
- 2) L0 striplets: cold flanges and Be pipe 390 new design
- 3) Some change in the SVT layout design : cryostat flange position-layer2/Layer3 module shift- L1/2 cooling ring/HDI - Conical Shield (below gimbal ring) for L0 cables transit...
- 4) L0 pixel: micro-channel bidirectional flow module support match the X_0 and operational temperature requested
- 5) Quick Demounting: solution proposed to be confirmed from Integration/Tech. Board (request more space in radius for temporary cage: 10 mm)
- 6) Start work on writing TDR, expected time to complete mechanics issue : end of June
- 7) Need to start to design jig and features to make mechanical prototype layer 5B



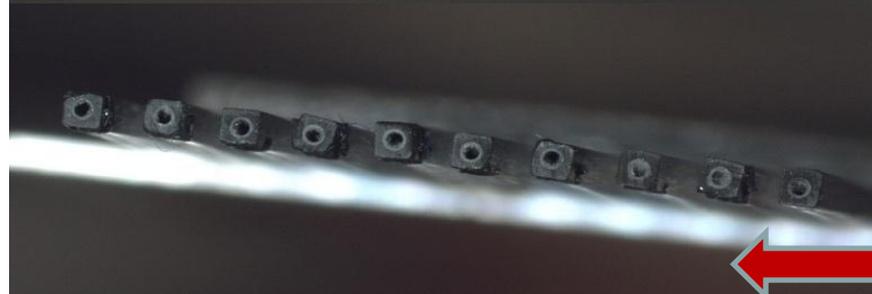
BACKUP



The single base microchannel unit
 A square CF micro-tube with an internal peek tube
 50 μm thick used to avoid moisture on carbon fiber



Full micro-channel module
 The total radiation length (*)
 of this support is **0.28 % X_0**



Net micro-channel module
 Same dimensions of full micro-channel but
 vacancies of tubes in the structure.
 The total radiation length (*) is **0.15 % X_0**

(*): Material of the support structure: (All C.F. material + peek tube + Water)



Microchannel Module Comparison Data

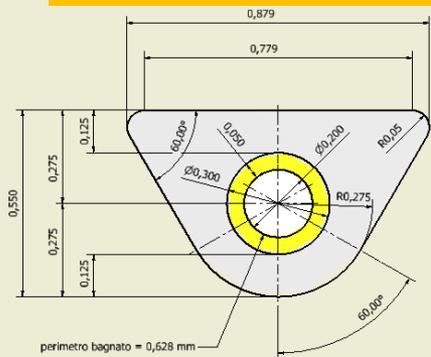


Comparison List

		% X0	T max °C	Specific Power W/cm ²	Hydraulic. Diam. μm	Flow rate g/min	Δ Temperature °C
1	Microchannel FULL Module support H=700 μm		35.8	2.0	300	244	5.3
2	Microchannel NET Module support H=700 μm	0.15	38.3	1.5	300	128	6,7
3	Microchannel Full Module support H=550 μm	0.22	34.3	1.5	200	33*	7.1*
4	Microchannel NET Module support H=550 μm	0.11	34.2	1.0	200	24	7.0

Tests performed on net module sample (length = 120 mm) with water-glycol @ 10 °C as coolant at $\Delta p = 3,5$ atm (Δp not valid for value *).

Trapezoidal 0.55



SUPERFICI:

S1 Carbon Fiber = 0,2671 mm²
 S2 Peek = 0,0393 mm²
 S3 H2O = 0,0314 mm²

RAPPORTO SUPERFICI:

$$(S1+S2)/S3 = (0,2671+0,0393)/0,0314 = 9,758$$

PERCENTUALE DI X0:

X0 Carbon Fiber = 28 cm
 X0 Peek = 25 cm
 X0 H2O = 36,08 cm

CALCOLO SU 1,40:

Carbon Fiber = $0,2671/1,40 = 0,1908$
 Carbon Fiber = $(0,1908/280) \times 100 = 0,0681$ % di X0
 Peek = $0,0393/1,40 = 0,0281$
 Peek = $(0,0281/250) \times 100 = 0,0112$ % di X0
 H2O = $0,0314/1,40 = 0,0224$
 H2O = $(0,0224/360,8) \times 100 = 0,0062$ % di X0

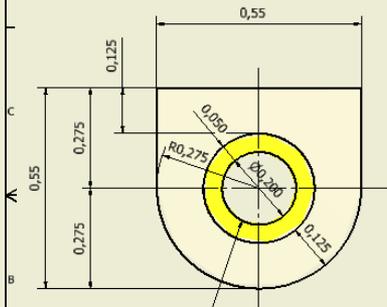
PERCENTUALE TOTALE X0 (1,40) Net: 0,0855 % di X0

CALCOLO SU 0,879:

Carbon Fiber = $0,2671/0,879 = 0,3039$
 Carbon Fiber = $(0,3039/280) \times 100 = 0,1085$ % di X0
 Peek = $0,0393/0,879 = 0,0447$
 Peek = $(0,0447/250) \times 100 = 0,0179$ % di X0
 H2O = $0,0314/0,879 = 0,3572$
 H2O = $(0,3572/360,8) \times 100 = 0,0987$ % di X0

perimetro bagnato = 0,628 mm

Square-Round 0.55



SUPERFICI:

S1 Carbon Fiber = 0,1994 mm²
 S2 Peek = 0,0393 mm²
 S3 H2O = 0,0314 mm²

RAPPORTO SUPERFICI:

$$(S1+S2)/S3 = (0,1994+0,0393)/0,0314 = 7,6019$$

PERCENTUALE DI X0:

X0 Carbon Fiber = 28 cm
 X0 Peek = 25 cm
 X0 H2O = 36,08 cm

CALCOLO SU 1,10:

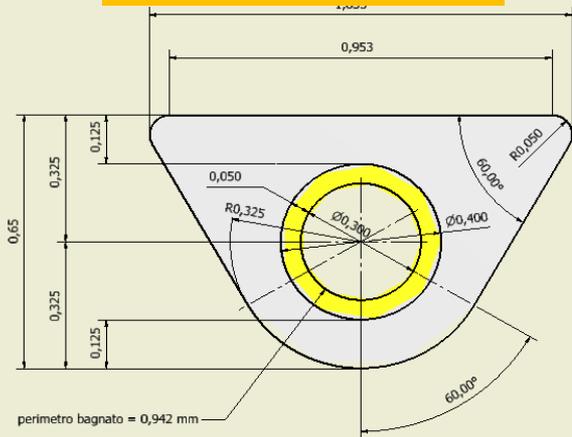
Carbon Fiber = $0,1994/1,10 = 0,1813$
 Carbon Fiber = $(0,1813/280) \times 100 = 0,0647$ % di X0
 Peek = $0,0393/1,10 = 0,0357$
 Peek = $(0,0357/250) \times 100 = 0,0156$ % di X0
 H2O = $0,0314/1,10 = 0,0285$
 H2O = $(0,0285/360,8) \times 100 = 0,0078$ % di X0

PERCENTUALE TOTALE X0 (1,10) Net: 0,089 di X0

CALCOLO SU 0,55:

Carbon Fiber = $0,1994/0,55 = 0,3625$
 Carbon Fiber = $(0,3625/280) \times 100 = 0,1295$ % di X0
 Peek = $0,0393/0,55 = 0,0715$
 Peek = $(0,0715/250) \times 100 = 0,0286$ % di X0
 H2O = $0,0314/0,55 = 0,5709$
 H2O = $(0,5709/360,8) \times 100 = 1,5823$ % di X0

Trapezoidal 0.65



SUPERFICI:

S1 Carbon Fiber = 0,3474 mm²
 S2 Peek = 0,055 mm²
 S3 H2O = 0,0707 mm²

RAPPORTO SUPERFICI:

$$(S1+S2)/S3 = (0,3474+0,055)/0,0707 = 5,6917$$

PERCENTUALE DI X0:

X0 Carbon Fiber = 28 cm
 X0 Peek = 25 cm
 X0 H2O = 36,08 cm

CALCOLO SU 1,40:

Carbon Fiber = $0,3474/1,40 = 0,2481$
 Carbon Fiber = $(0,2481/280) \times 100 = 0,0886$ % di X0
 Peek = $0,055/1,40 = 0,0393$
 Peek = $(0,0393/250) \times 100 = 0,0157$ % di X0
 H2O = $0,0707/1,40 = 0,0505$
 H2O = $(0,0505/360,8) \times 100 = 0,0140$ % di X0

PERCENTUALE TOTALE X0 (1,40) Net: 0,1183 % di X0

CALCOLO SU 1,053:

Carbon Fiber = $0,3474/1,053 = 0,3299$
 Carbon Fiber = $(0,3299/280) \times 100 = 0,1178$ % di X0
 Peek = $0,055/1,053 = 0,0522$
 Peek = $(0,0522/250) \times 100 = 0,0209$ % di X0
 H2O = $0,0707/1,053 = 0,0671$
 H2O = $(0,0671/360,8) \times 100 = 0,0186$ % di X0

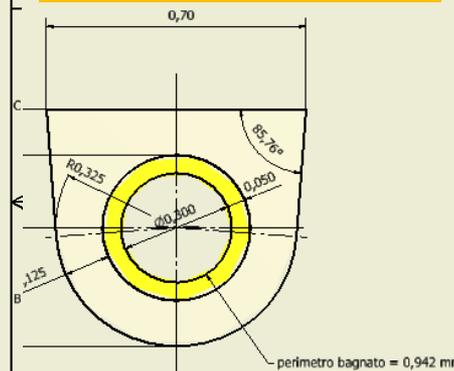
PERCENTUALE TOTALE X0 (1,053) Full: 0,1573 % di X0

DIAMETRO IDRAULICO:

Dh = 0,300 mm

perimetro bagnato = 0,942 mm

Square-round 0.65



SUPERFICI:

S1 Carbon Fiber = 0,2599 mm²
 S2 Peek = 0,055 mm²
 S3 H2O = 0,0707 mm²

RAPPORTO SUPERFICI:

$$(S1+S2)/S3 = (0,2599+0,055)/0,0707 = 4,4540$$

PERCENTUALE DI X0:

X0 Carbon Fiber = 28 cm
 X0 Peek = 25 cm
 X0 H2O = 36,08 cm

CALCOLO SU 1,40:

Carbon Fiber = $0,2599/1,40 = 0,1856$
 Carbon Fiber = $(0,1856/280) \times 100 = 0,0663$ % di X0
 Peek = $0,055/1,40 = 0,0393$
 Peek = $(0,0393/250) \times 100 = 0,0157$ % di X0
 H2O = $0,0707/1,40 = 0,0505$
 H2O = $(0,0505/360,8) \times 100 = 0,0140$ % di X0

PERCENTUALE TOTALE X0 (1,40) Net: 0,0874 di X0

CALCOLO SU 0,70:

Carbon Fiber = $0,2599/0,70 = 0,3713$
 Carbon Fiber = $(0,3713/280) \times 100 = 0,1326$ % di X0
 Peek = $0,055/0,70 = 0,0786$
 Peek = $(0,0786/250) \times 100 = 0,0314$ % di X0
 H2O = $0,0707/0,70 = 0,101$
 H2O = $(0,101/360,8) \times 100 = 0,0280$ % di X0

PERCENTUALE TOTALE X0 (0,70) Full: 0,192 % di X0

DIAMETRO IDRAULICO:

Dh = 0,300 mm

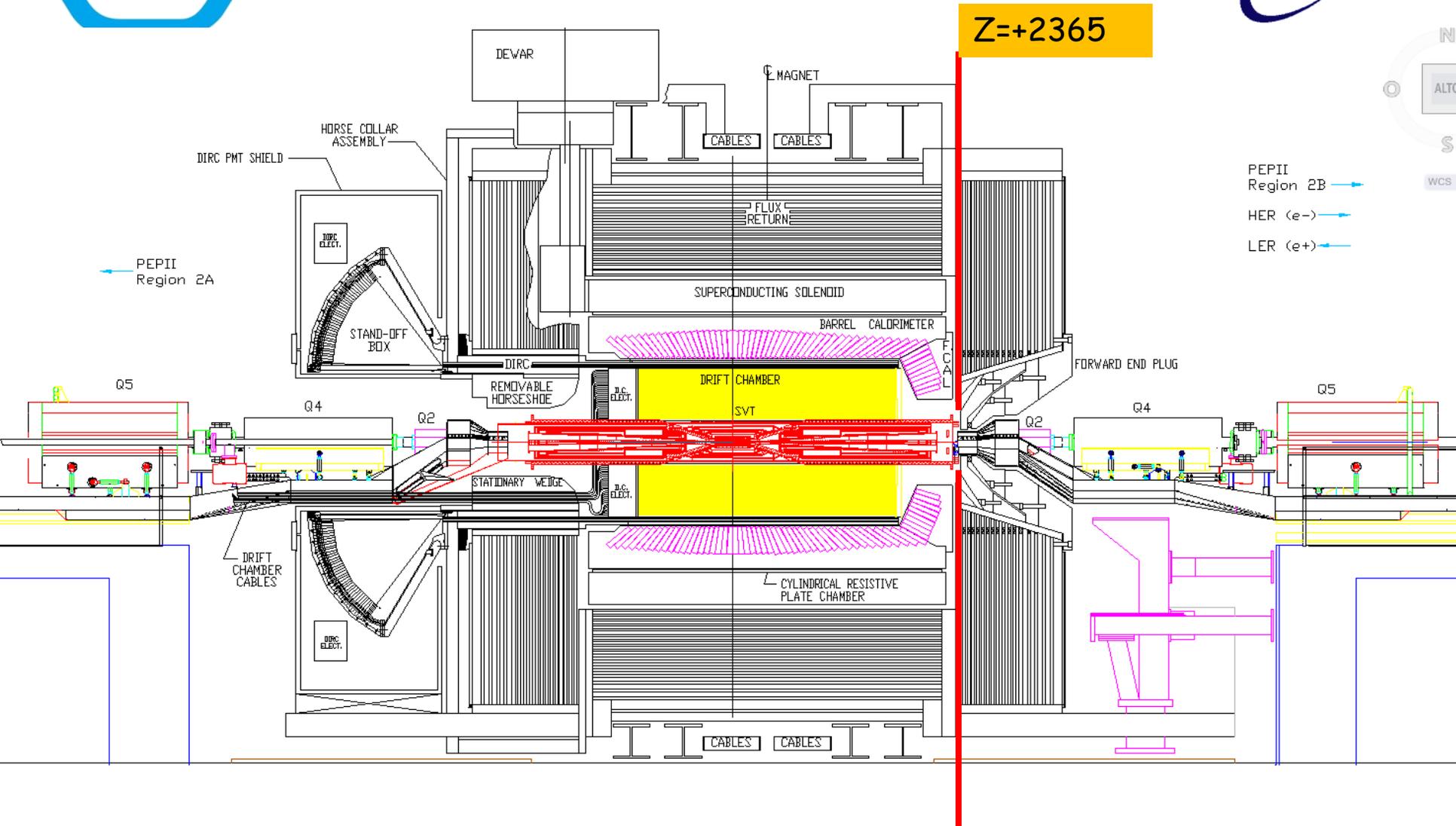
Progettato da F. Bosi	Controllato da	Approvato da	Data	Data
			11/02/2012	
Istituto Nazionale di Fisica Nucleare-Sezione di Pisa		Assieme_tubo_trapezio_065_040		
SuperB		Revisione	1/1	

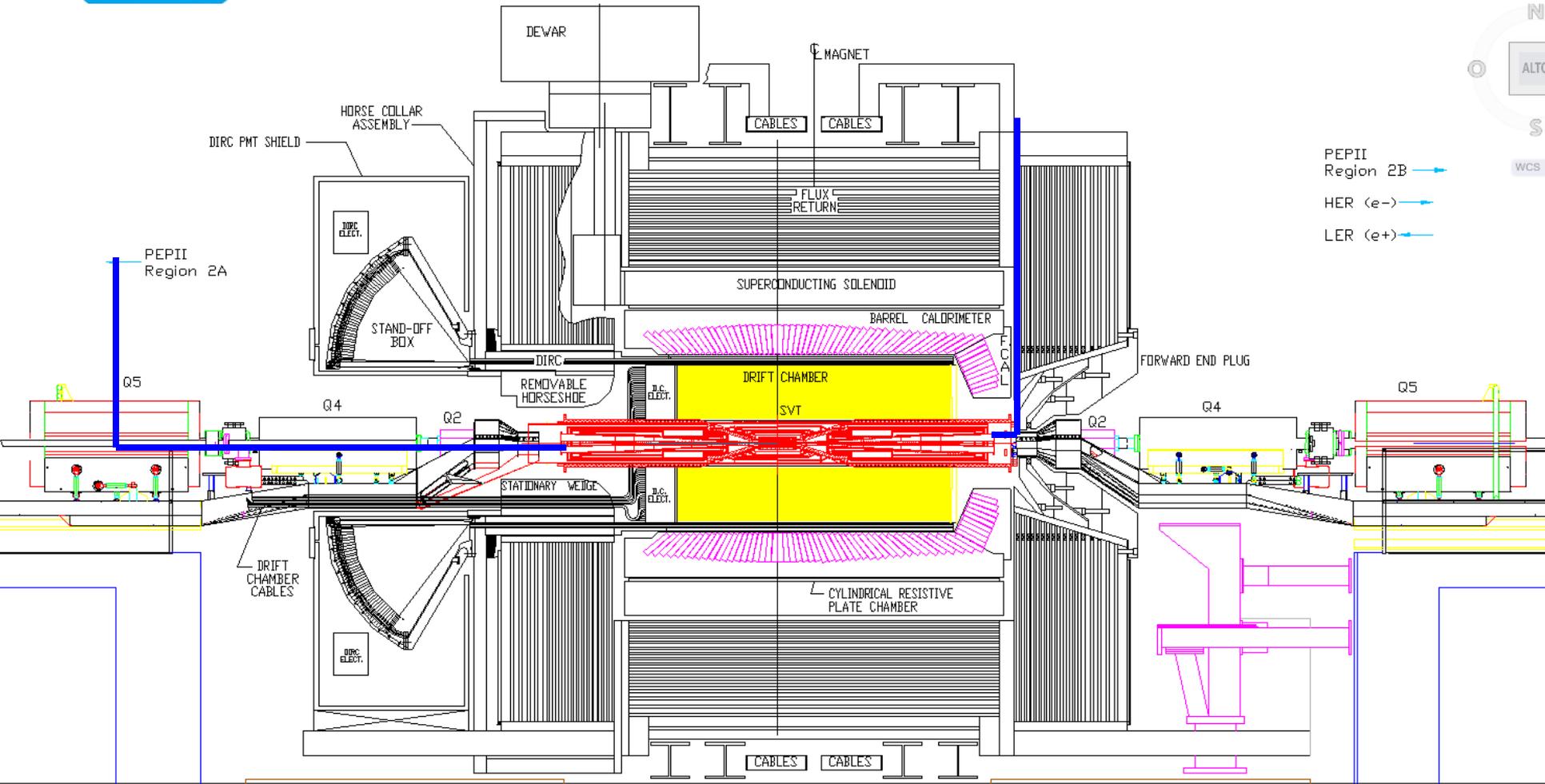


Thermal simulation / X_0



			% X0 tot		W/cm ²	0,5	1	1,5	2
		N° MC							
Quadrato 0,7 mm (full)		18	0,241300431		T [°C]		22,03	28,08	34,12
Quadrato 0,7 mm (net)		10	0,134055795		T [°C]	20,26	30,38	40,51	50,63
Quadrato 0,55 mm (full)		23	0,192561102		T [°C]	16,1	22,18	28,29	34,41
Quadrato 0,55 mm (net)		12	0,100466662		T [°C]	20,9	31,64	42,37	53,11
Trapezio 065_40 (net)		10	0,129312491		T [°C]	18,38	26,69	35,01	43,33
Trapezio 065_40 (full)		12	0,15517499		T [°C]	17,19	24,3	31,43	38,55
Trapezio 0,55 mm (full)		14	0,130882981		T [°C]	17,76	25,42	33,09	40,76
Trapezio 0,55 mm (net)		12	0,112185412		T [°C]	18,87	27,63	36,41	45,18
Quadrato tondo 0,7 mm (full)		18	0,189068288		T [°C]	16,08	22,13	28,2	34,28
Quadrato tondo 0,55 mm (full)		23	0,171383647		T [°C]	16,14	22,26	28,41	34,56
Quadrato tondo 0,55 mm (net)		12	0,089417555		T [°C]	20,91	31,67	42,42	53,18





PEP-II Region 2B →
 HER (e-) →
 LER (e+) →

